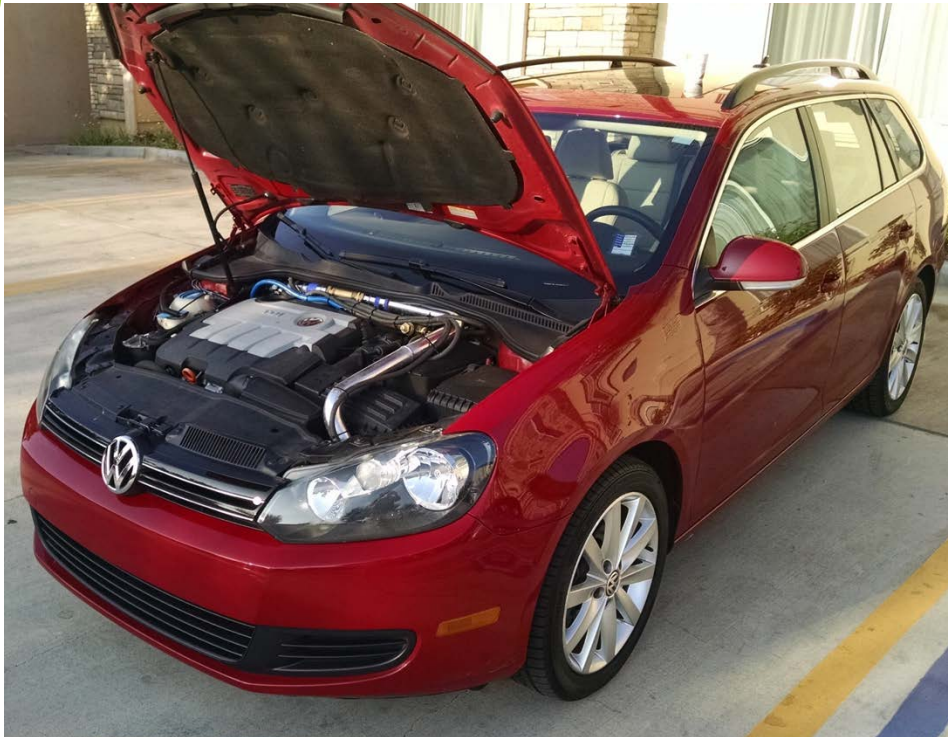


Cool Particulate Regeneration

Global Clean Diesel

Dr. Seshu Srinivasan, Matthew Lawyer, Mason Carroll,
Florida Polytechnic University

Brett Bailey, Global Clean Diesel



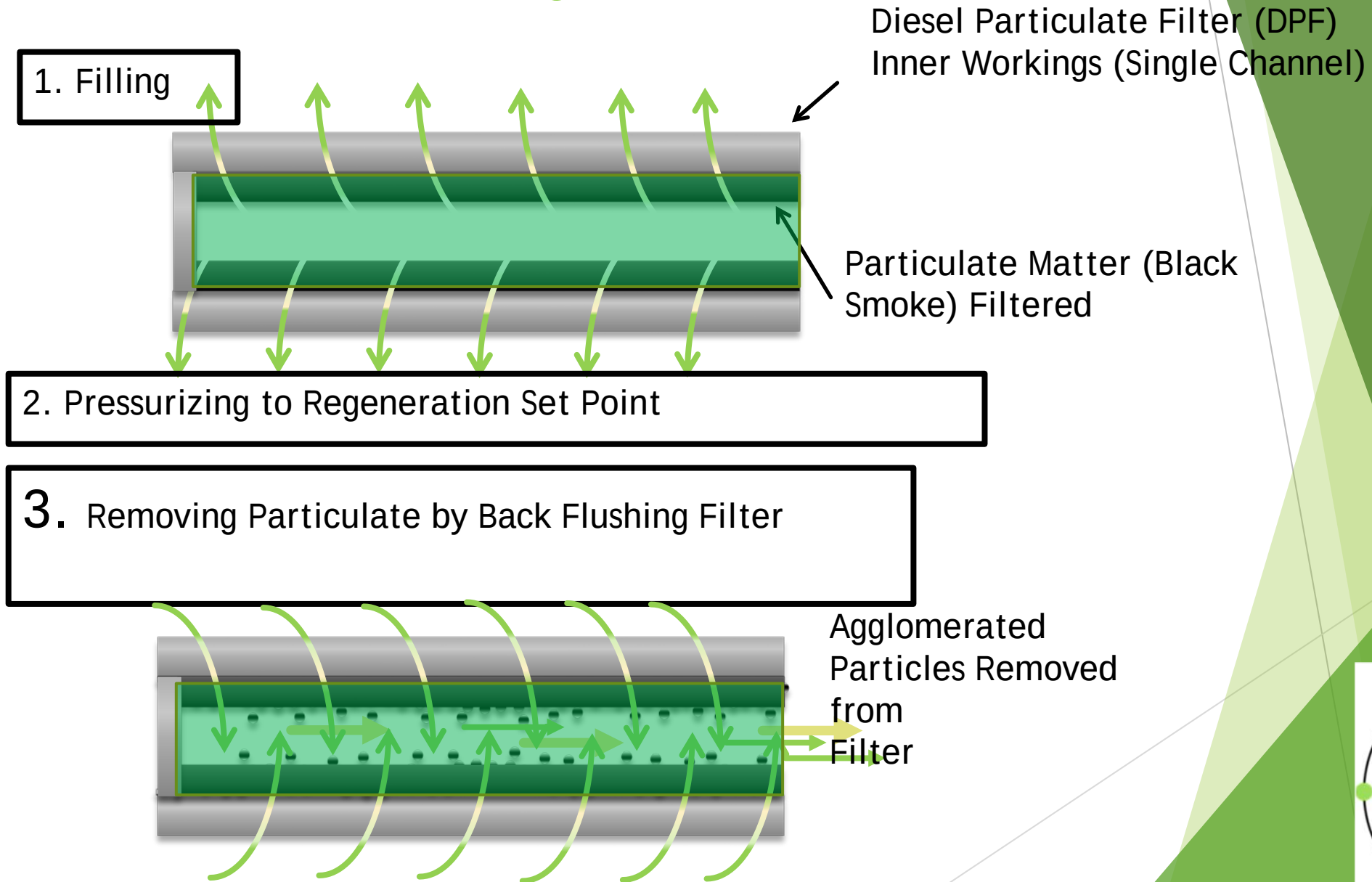
Motivation

Global Real World Emissions Reduction

Cool Particulate Regeneration (CPR)

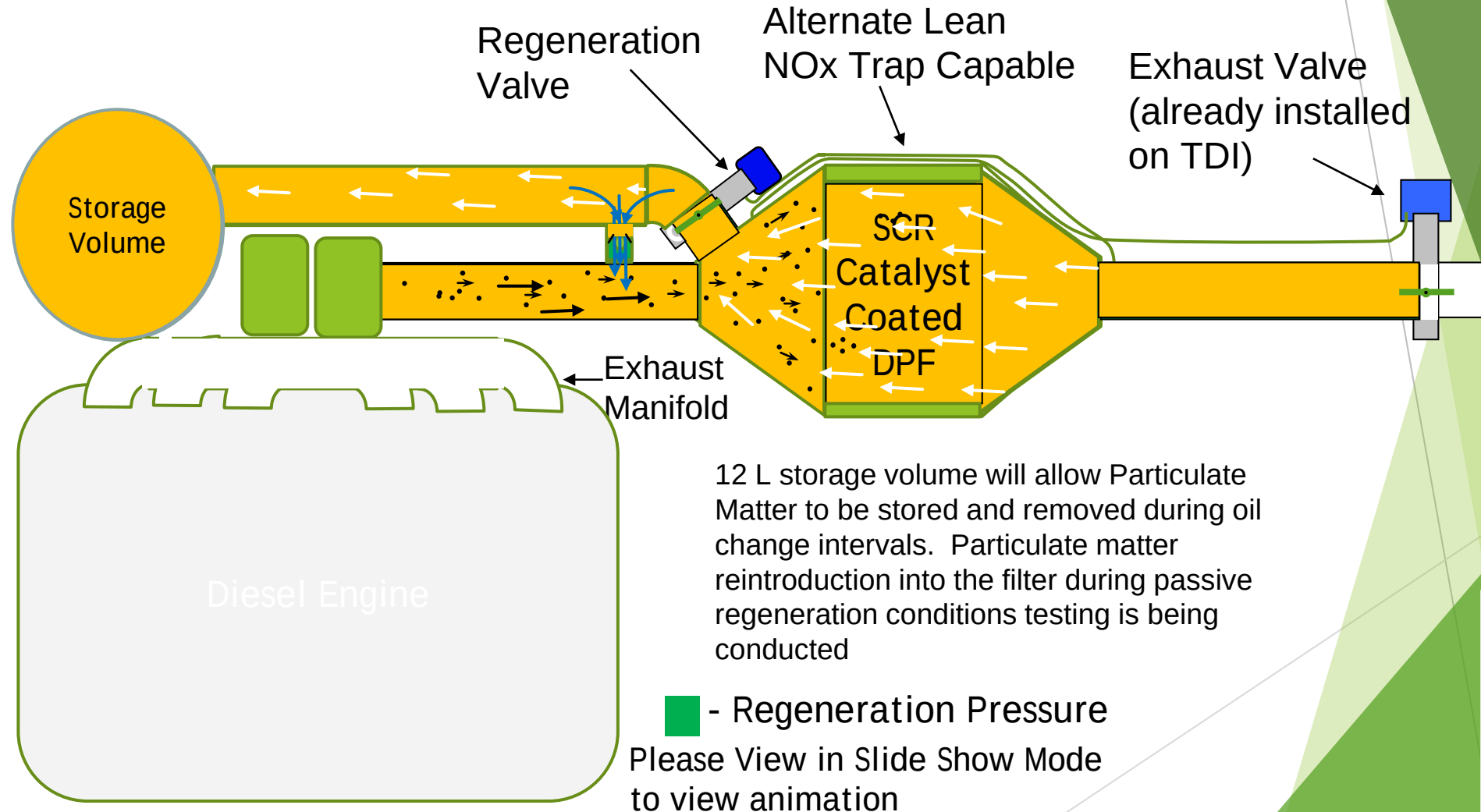
- ▶ **Cost Saving** by Reduction in Engine and Aftertreatment Complexity, Size, and Weight.
- ▶ **Fuel Economy Improvement**- Regeneration Utilizes Unused Vehicle Kinetic Energy during Braking Conditions.
- ▶ **Real World Emissions Enhanced** – Quicker Light off of SCR Catalyst and Reduction of Particulate Number (PN#) emissions
- ▶ **Performance and Drivability** – Richer Air/Fuel Ratio Acceleration
- ▶ **Emerging Markets** - Ultra Low Cost Mechanical Engine (High Particulate Matter (PM)) and Fuel Tolerant Technology (Sulfur Tolerant) Solution
- ▶ **Durability/Reliability** - Reduction in catalyst thermal degradation and filter thermal stresses. Ongoing Ash removal. Aftertreatment Life Corresponding with Engine Life Expected.
- ▶ **Future Powertrain** - Hybrid, Exhaust Energy Recovery (EER) and Stop/Start enabler. Regeneration within Engine Operation

Cool Particulate Regeneration Fundamentals



Cool Particulate Regeneration™, CPR™

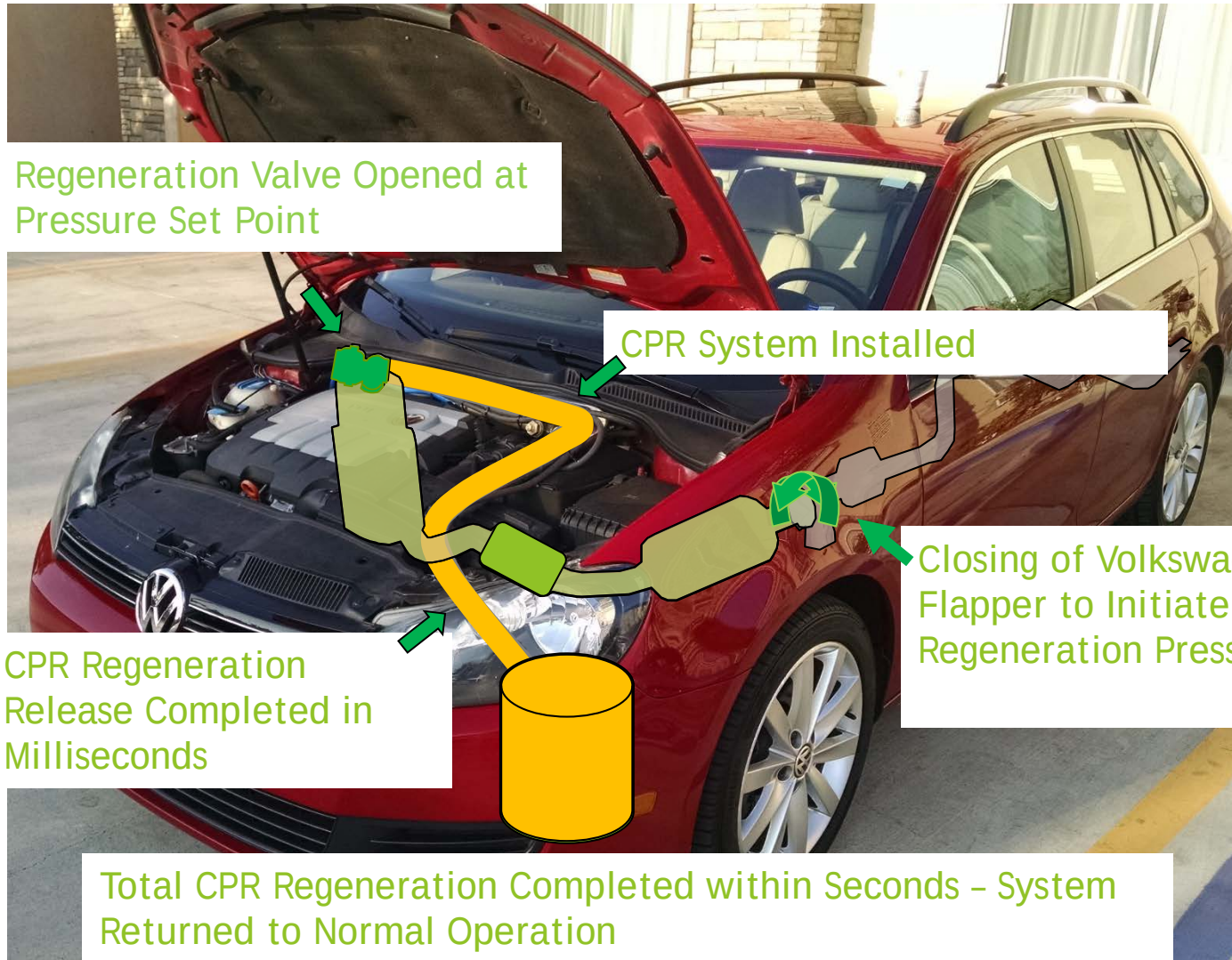
How it works



DOC – Diesel Oxidation Catalyst DPF – Diesel Particulate Filter LNT – Lean NOx Trap



CPR Demonstration Vehicle



Modified Jetta Hardware



Cost Reduction Enables Next Generation



Emerging
Market Diesel
Engine



Hybridization of
the Diesel in
Developed
Markets for CO₂



CPR



Too Costly
Poor Real World
Emissions



EU LDVs Diesel Emissions Cost Ever Increasing

- ▶ Diesel Engine Costs Allowing Gasoline and Electric Vehicle traction in the market place
- ▶ CPR DPF \$337
- ▶ DPF/SCR Combined ~\$306
- ▶ Mechanical/Low Press Common Rail ~\$337
- ▶ EGR Removal

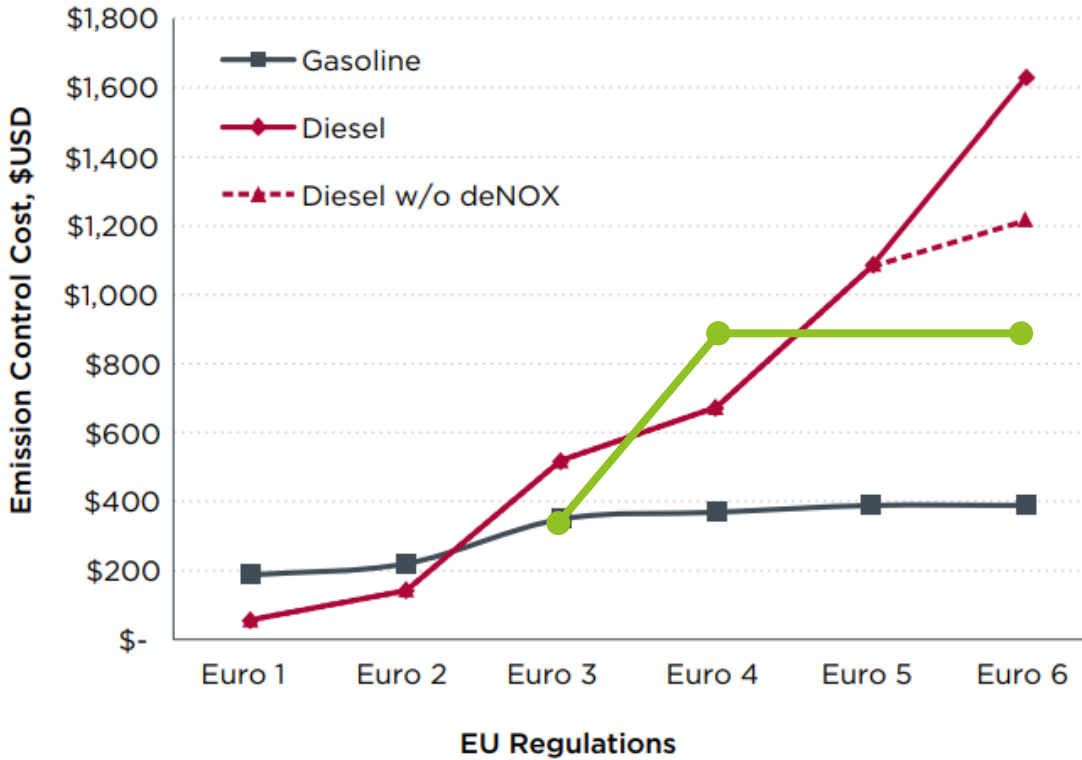


Figure ES-2 Estimated cumulative emission control technology cost for vehicles assuming a 2.0 L engine

Table ES-1 Incremental costs for LDVs meeting European standards (2010 dollars)

ENGINE TYPE	VEHICLE CLASS	EURO 1 (BASELINE)	EURO 1 TO EURO 2	EURO 2 TO EURO 3	EURO 3 TO EURO 4	EURO 4 TO EURO 5	EURO 5 TO EURO 6	NO CONTROL TO EURO 6
Gasoline	4 cylinders Vd= 1.5 L	\$142	\$63	\$122	\$25	\$10	--	\$362
Gasoline	4 cylinders Vd = 2.5 L	\$232	\$3	\$137	\$15	\$30	--	\$417
Diesel	4 cylinders Vd = 1.5 L	\$56	\$84	\$337	\$145	\$306	\$471	\$1,399
Diesel	4 cylinders Vd = 2.5 L	\$56	\$89	\$419	\$164	\$508	\$626	\$1,862



Emerging Market Main Engine

Developed Country Hybrid Diesel

Table 5. In-cylinder control and OBD costs.

Technology	Cost
Fuel system, common rail or extra with respect to unit injector systems	\$750 + 10% for each successive standard
VGT (extra cost with respect to turbocharger)	\$370
EGR system	\$439
EGR intercooler	\$108 (\$85-130)
Full OBD	\$425 (\$350-\$500)





Computational Fluid Dynamics (CFD) Analysis of CPR on the Volkswagen Jetta TDI

What is Florida Polytechnic University?

- Established in 2014 and regionally accredited in 2017, Florida Polytechnic University is a science, technology, engineering, and mathematics (STEM) focused university.
- Florida Polytechnic University has a IBM Supercomputer which was utilized in unison with Converge Science CFD to complete this project.



FLORIDA POLYTECHNIC
UNIVERSITY

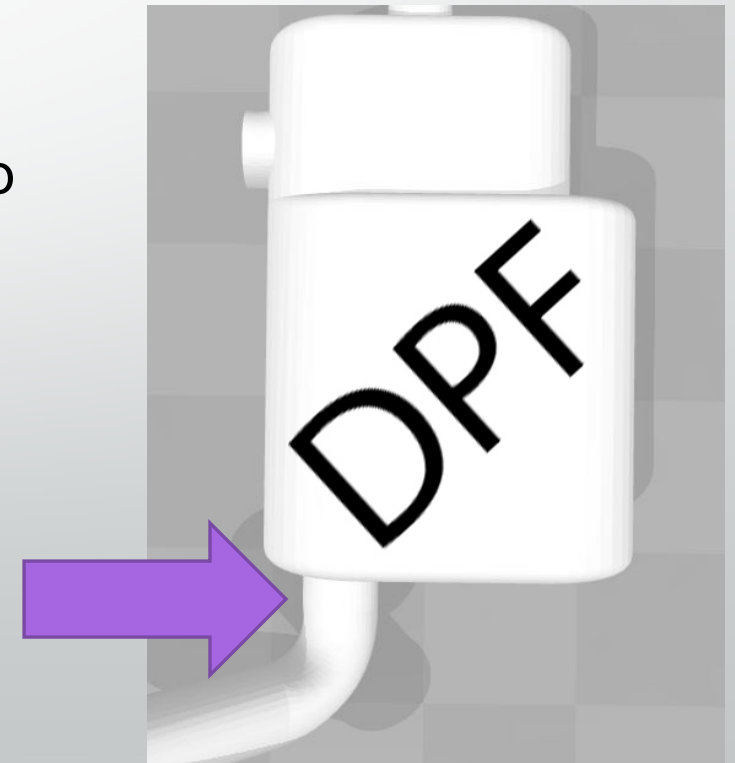


Background

- The success of GCD's marketing demonstrations put the project on hold until May of 2018. GCD's success, in the Indian market, shifted the focus to low speed operation
- Limited testing, under Indian low load, demonstrated a build up of backpressure.
- CFD analysis of the Jetta's CPR began in the summer of 2018 to gain a better understanding of CPR regeneration.
- The CFD analysis revealed that the regenerations were suffering from several flow issues.

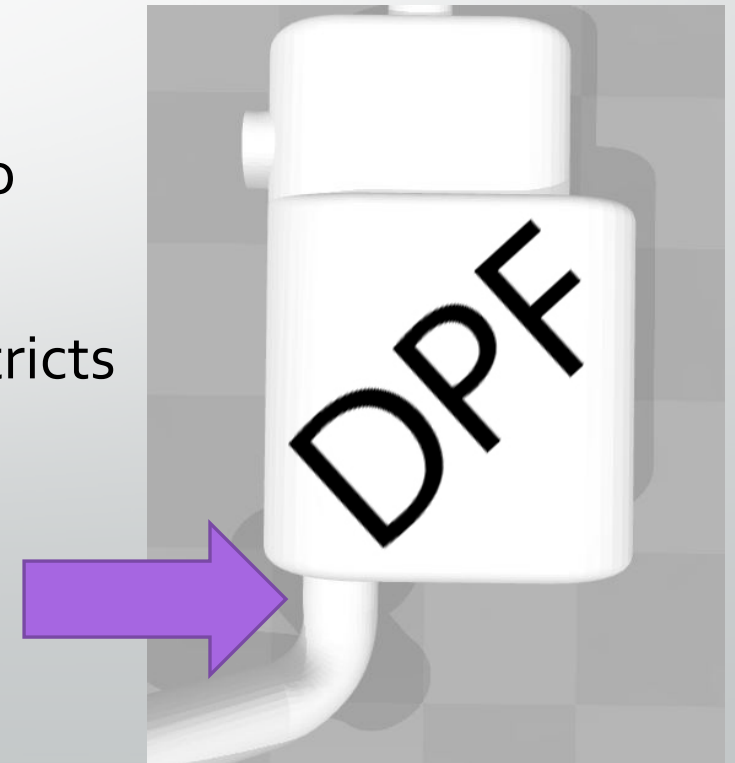
Flow Challenges

- Design modifications were limited by packaging constraints
- The off center engine exhaust outlet creates a bias to one side of the filter during reverse flow.



Flow Challenges

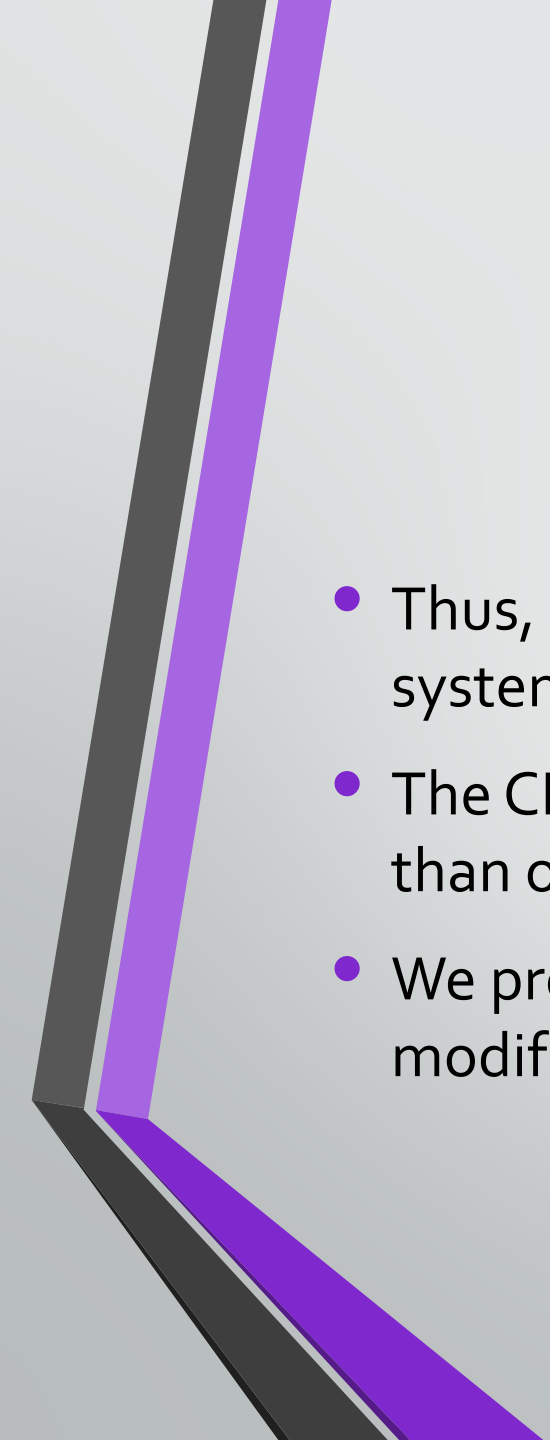
- Design modifications were limited by packaging constraints
- The off center engine exhaust outlet creates a bias to one side of the filter during reverse flow.
- The bottom of DPF is close to the exhaust which restricts the flow to the right creating a left flow bias.



Flow Challenges

- Design modifications were limited by packaging constraints
- The off center engine exhaust outlet creates a bias to one side of the filter during reverse flow.
- The bottom of DPF is close to the exhaust which restricts the flow to the right creating a left flow bias.
- Additionally his lip further restricts flow on the right side.





Our Solutions

- Thus, given the previously stated design challenges, simply attaching a CPR system to the Jetta does not achieve our development goal.
- The CFD simulations suggest that certain parts of the filter receive less flow than others, reducing the effectiveness of regeneration.
- We propose a 20% reduction of filter volume and a slight design modification to allow for more uniform flow.

Our Solutions (cont.)

Before



After



Filter length
reduced by 46
MM



Lip length
reduced by 13
MM



Our Solutions (cont.)

- With this new filter design modeled and simulated, we've addressed two solutions that represent two opposites sides of a spectrum. These two being with no vacuum and with a vacuum. However many possible solutions exist within these two bounds.
- Solution 1: Simply increase the regeneration pressure.
- Solution 2: Pull a vacuum, in the storage tank, to keep regeneration pressures low.

Solution 1

(no vacuum)

- A 17 liter storage tank and a 66 mm Poppet valve were mounted to the top of the oxidation catalyst.

17 liter tank at
ambient pressure.

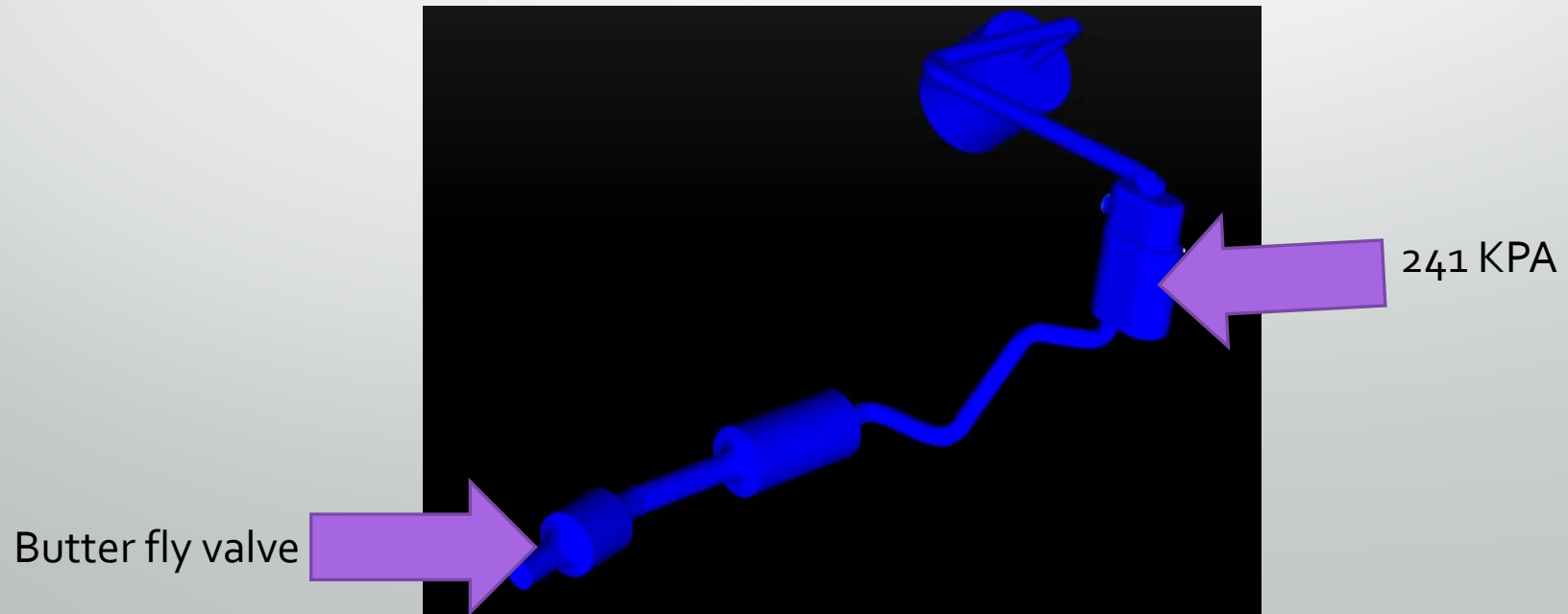


66 mm Poppet
valve

Solution 1

(no vacuum)

- Then, the butterfly valve, already installed for EGR, was closed to pressurize the system (not including the 17 liter tank) to 240 kPa gauge.



No Vacuum Results



No Vacuum Results

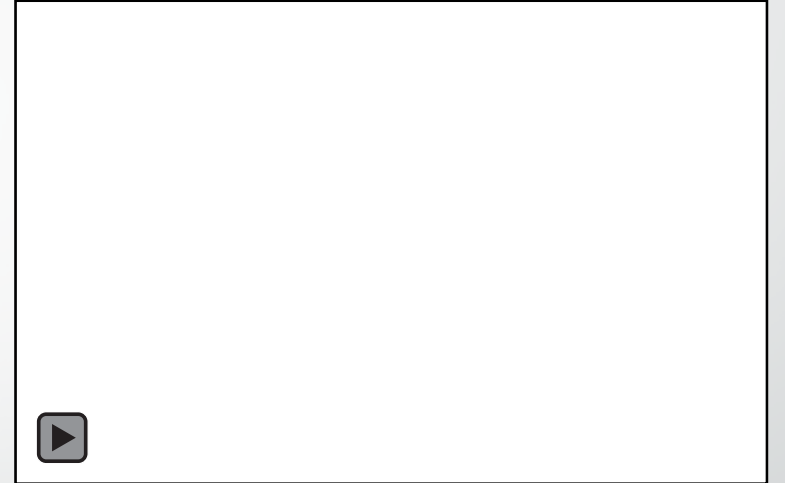


No Vacuum Results



Solution 1 (no vacuum) Conclusion

- The particle traces demonstrate a successful regeneration in 1.26 s.
- The system takes exactly 1.16 s to build up pressure.
- The system takes 0.1 s to complete the regeneration.



Solution 1 (no vacuum) Conclusion

Advantages:

- Very simple
- Low cost
- Effective

Disadvantages:

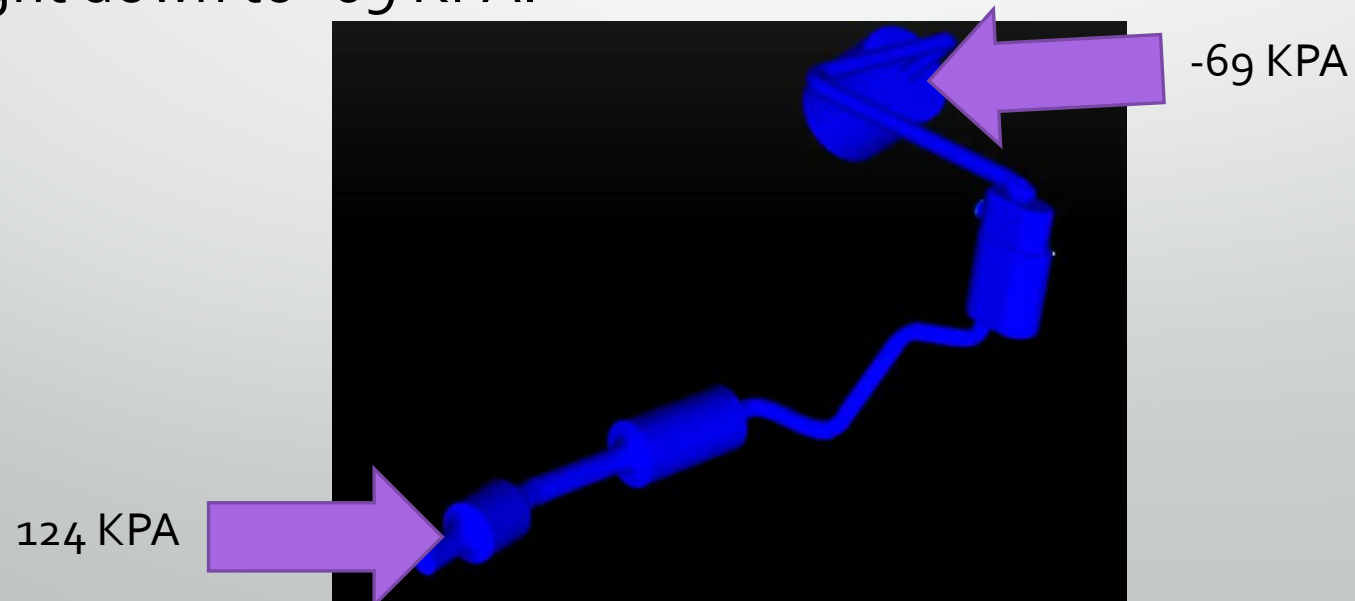
- Requires higher regeneration pressure

We envision this solution being used on everyday cars and large vehicles

Solution 2

(vacuum)

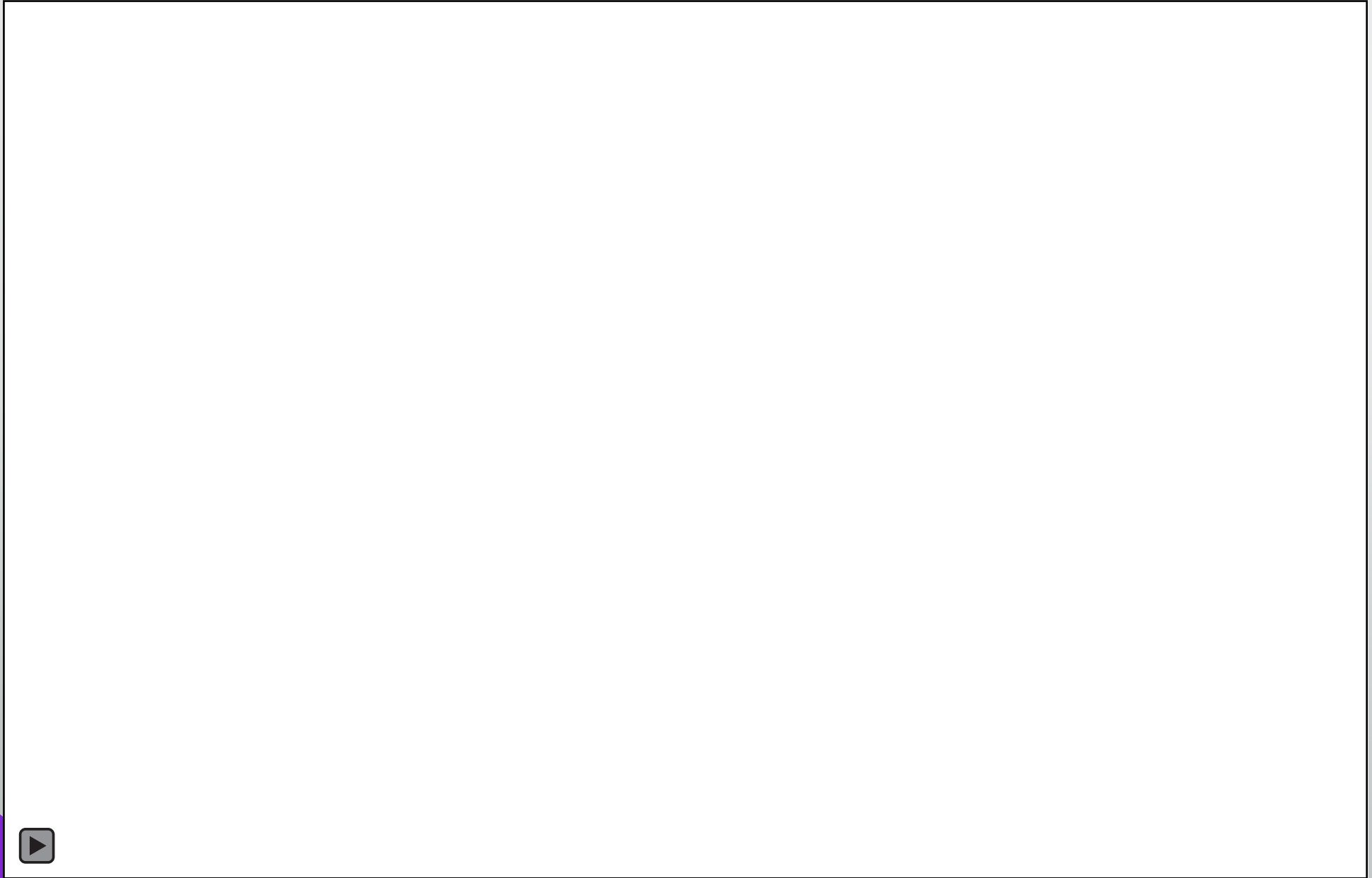
- The design is identical, the system was pressurized (not including the 17 litre tank) to 124 KPA. Then, the pressure in the 17 litre tank and the Poppet value was brought down to -69 KPA.



Vacuum Results



Vacuum Results

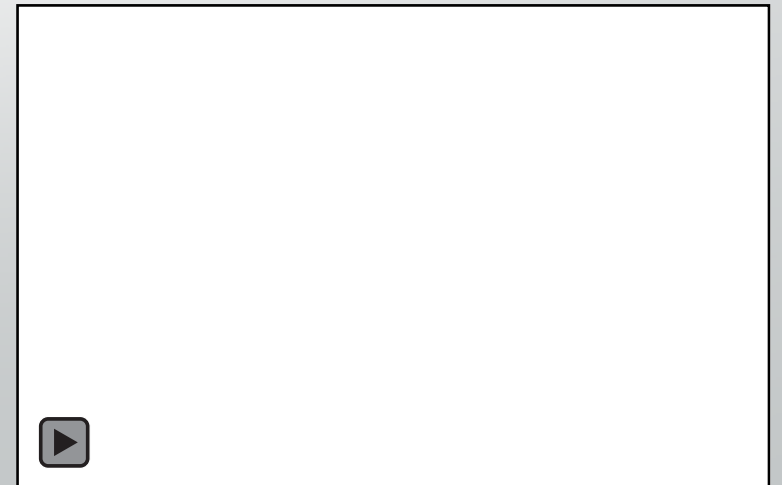
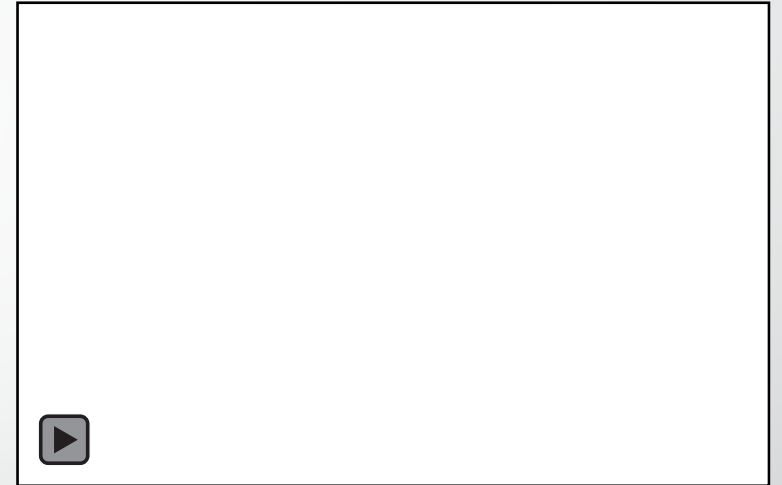


Vacuum Results



Solution 2 (vacuum) Conclusion

- The flow is mostly uniform and the particle traces demonstrate a successful regeneration in 0.5 s.
- The system takes 0.4 s to build up pressure.
- The system takes 0.1 s to complete the regeneration.



Solution 2 (vacuum) Conclusion

Advantages:

- Better distribution of flow
- Faster regeneration

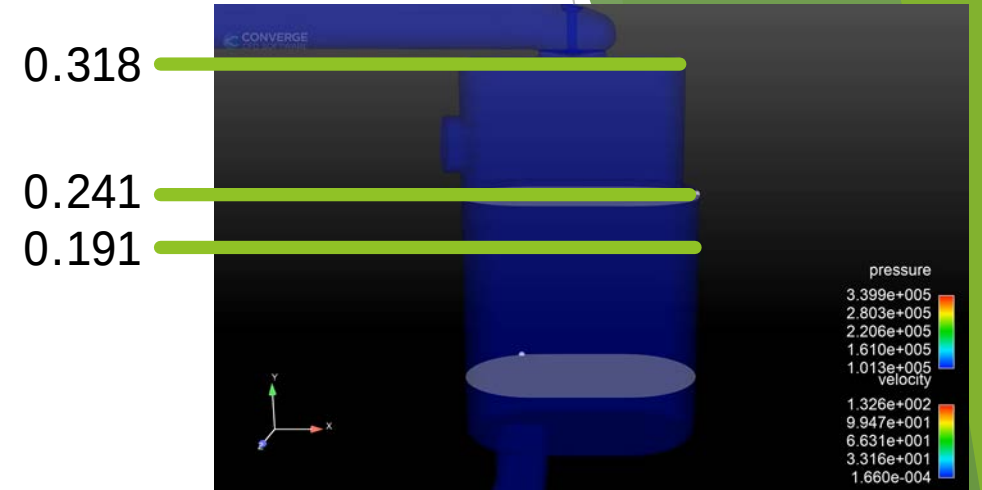
Disadvantages:

- More complex
- More expensive

We envision this solution being used on luxury cars where efficiency is important and cost is not as much of an issue.

Final Thoughts On CFD

- We overcame the challenges and with CFD we were able to demonstrate it is possible to implement CPR in the Volkswagen Jetta TDI.
- These solutions were also capable of no passive regeneration conditions with a minor design change of the filter.
- The CPR regeneration is quick and efficient requiring no thermal active regeneration.

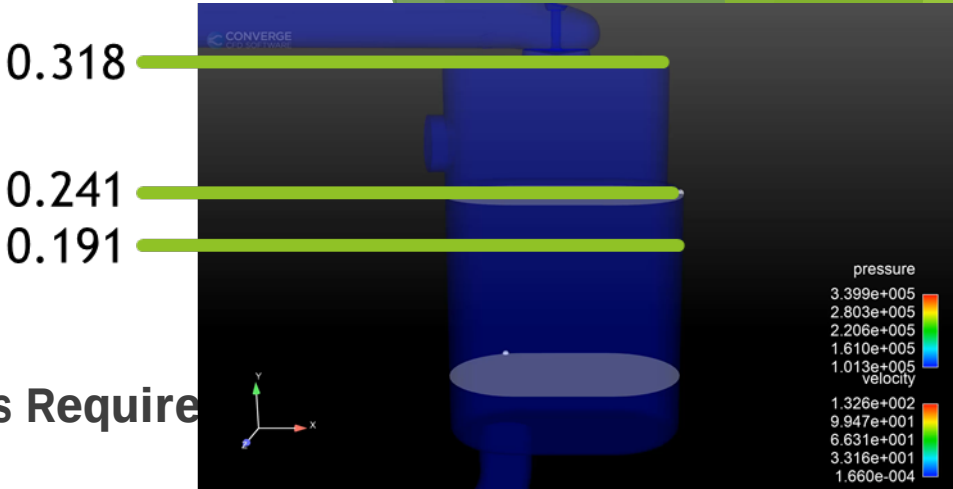


Cool Particulate Regeneration Unsteady 1D Code

Quickly Research and Develop new CPR system demonstrations

CPR 1D Modeling Tests

Test #	RegenP	Valve	Tank	RevDist	Delta P	Annulus Required
	kPa	mm	I	kPa	I	
▶ Test 1	240	46	10.2	0.164	17.15	Y
▶ Test 2	340	46	10.2	0.22	33.4	Y
▶ Test 3	340	46	15.4	0.269	58	Y
▶ Test 4	240	66	15.4	0.1843	24	Y
▶ Test 5	340	66	15.4	0.269	58	?
▶ Test 6	340	66				



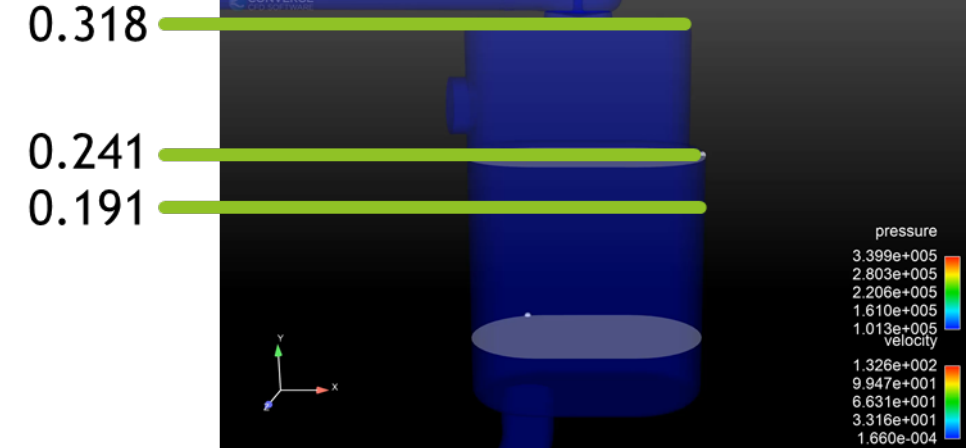
Test 1 Baseline Arrangement

240kPa (20psi) 10.2l (2.75gal) Holding

46mm Regen Valve

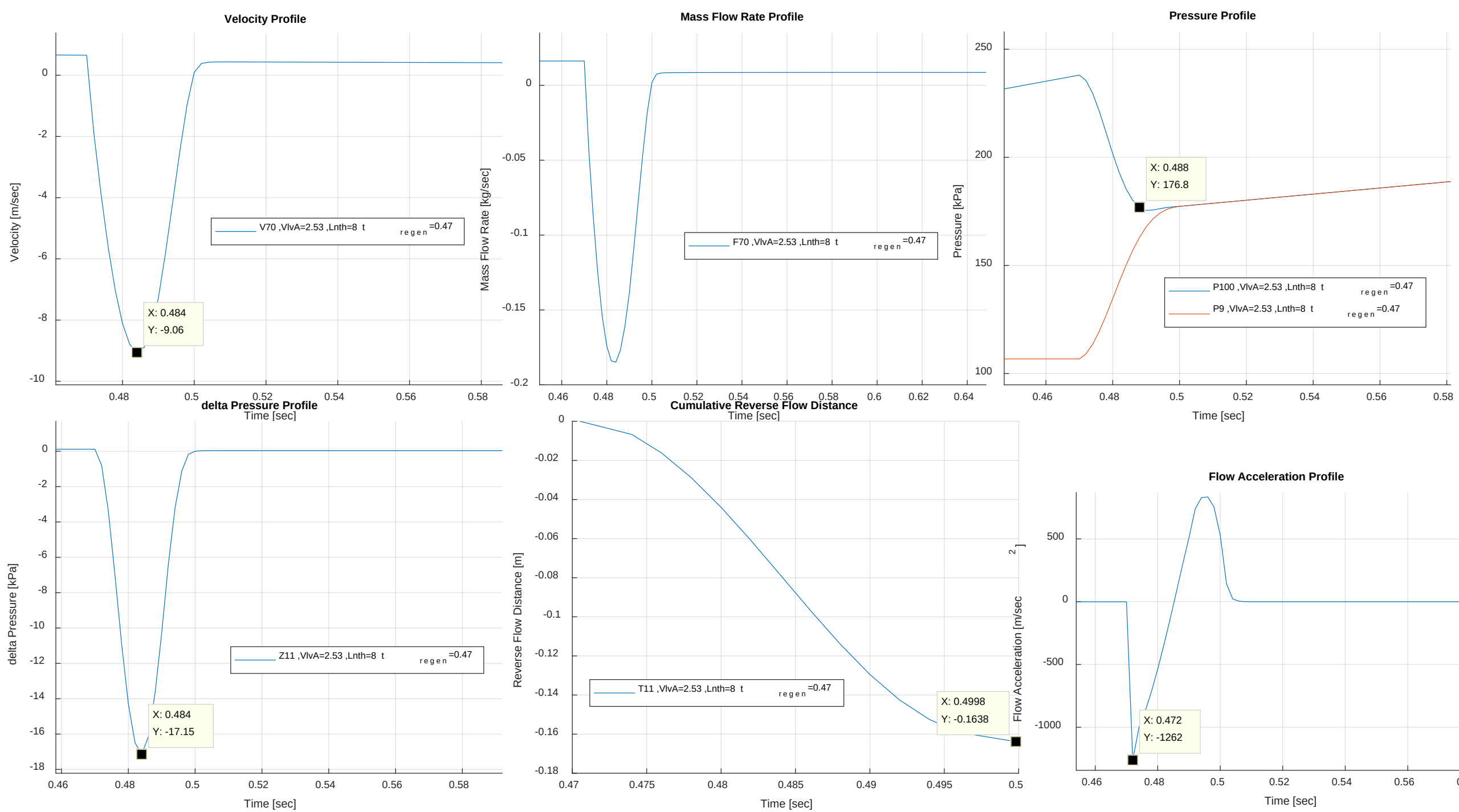
Base Holding Tank Size (10L)

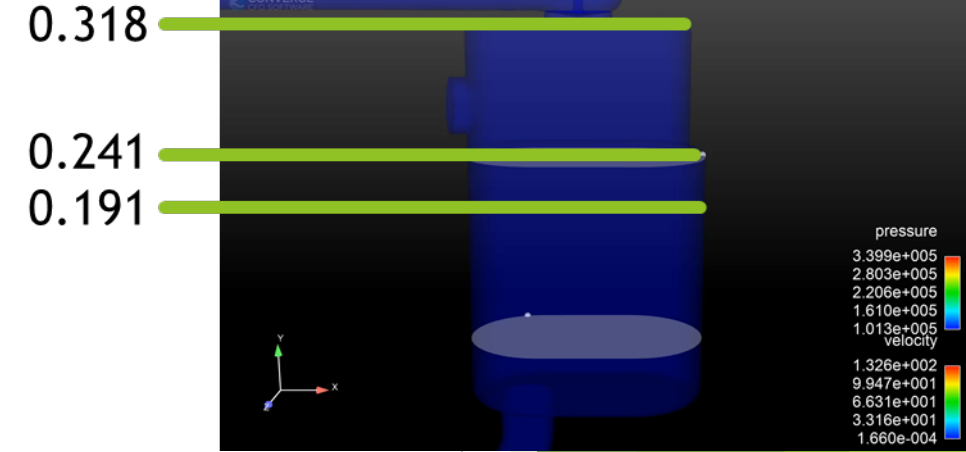
Regeneration through an Annulus Design



Annulus Design or Outlet between DOC and DPF required for low Loads

- ▶ 0.164 m reverse flow distance
- ▶ 17.2kPa Delta P pressure
- ▶ 9 m/s reverse flow velocity
- ▶ 86% reversed flowed out of the filter



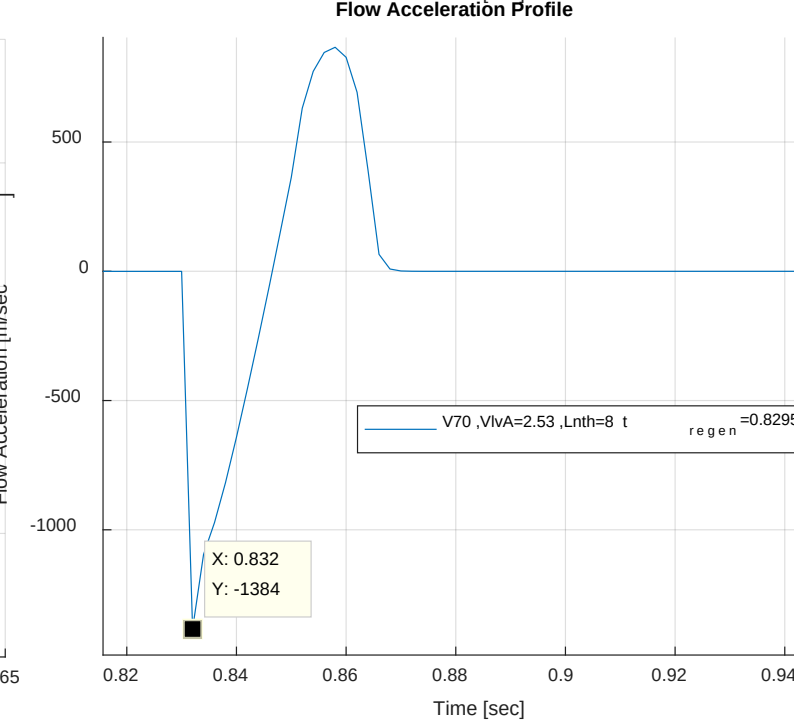
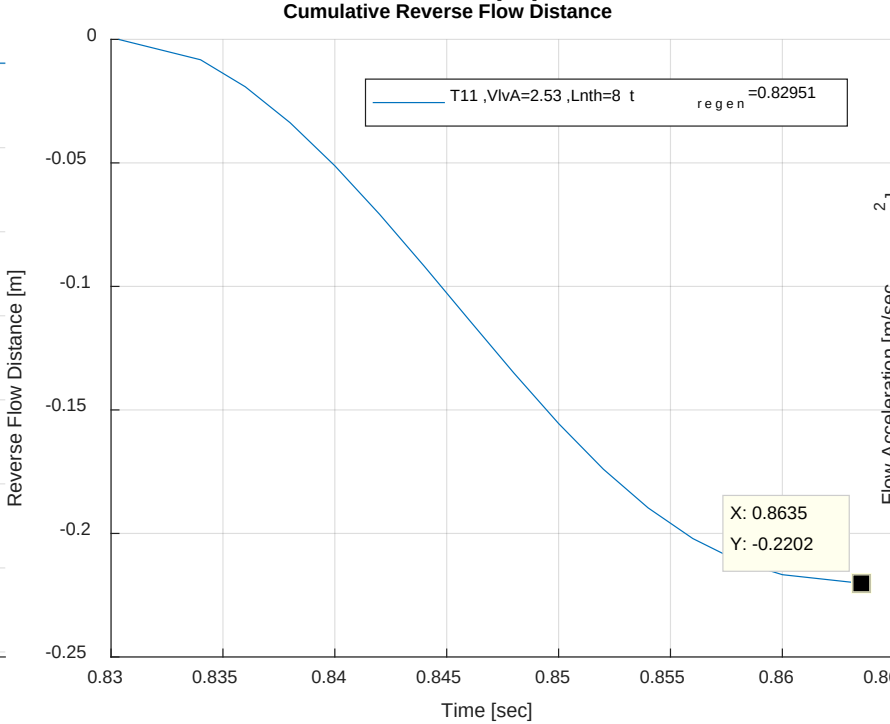
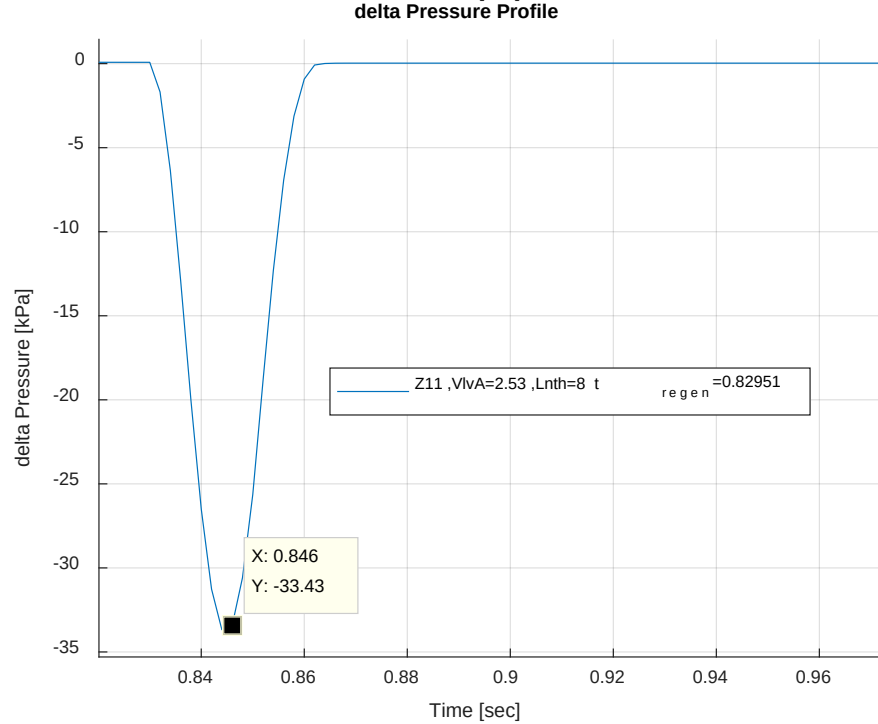
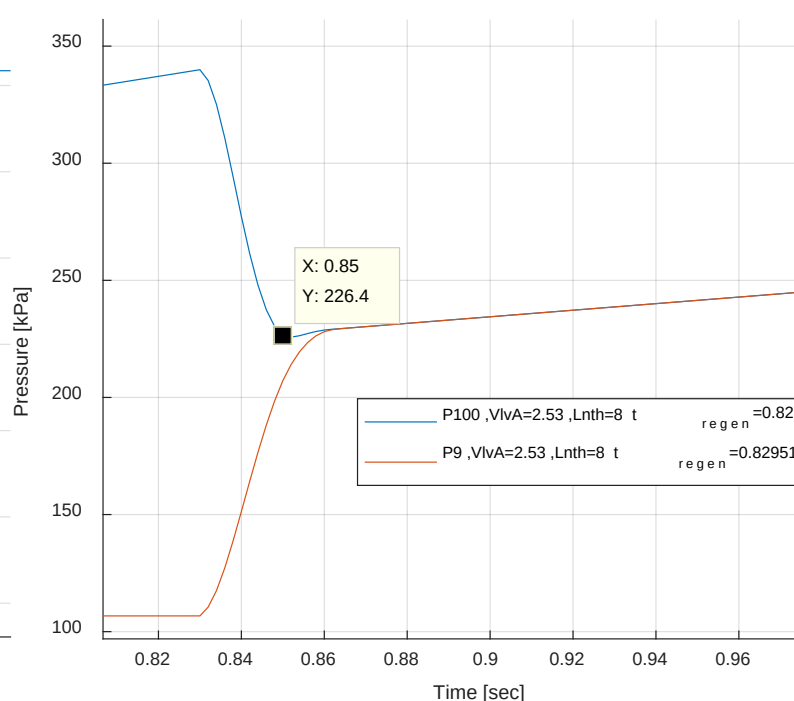
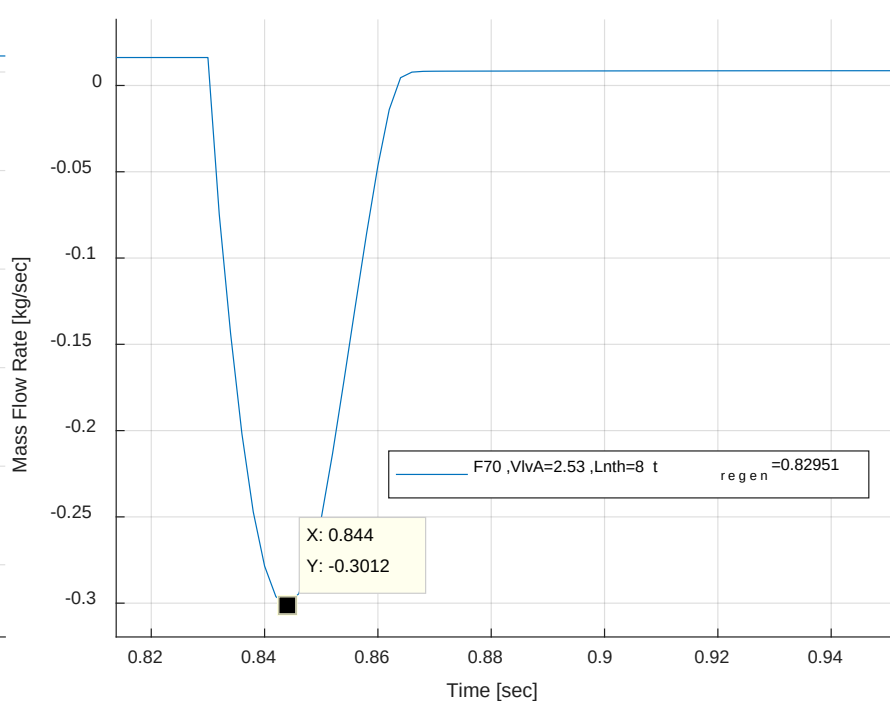
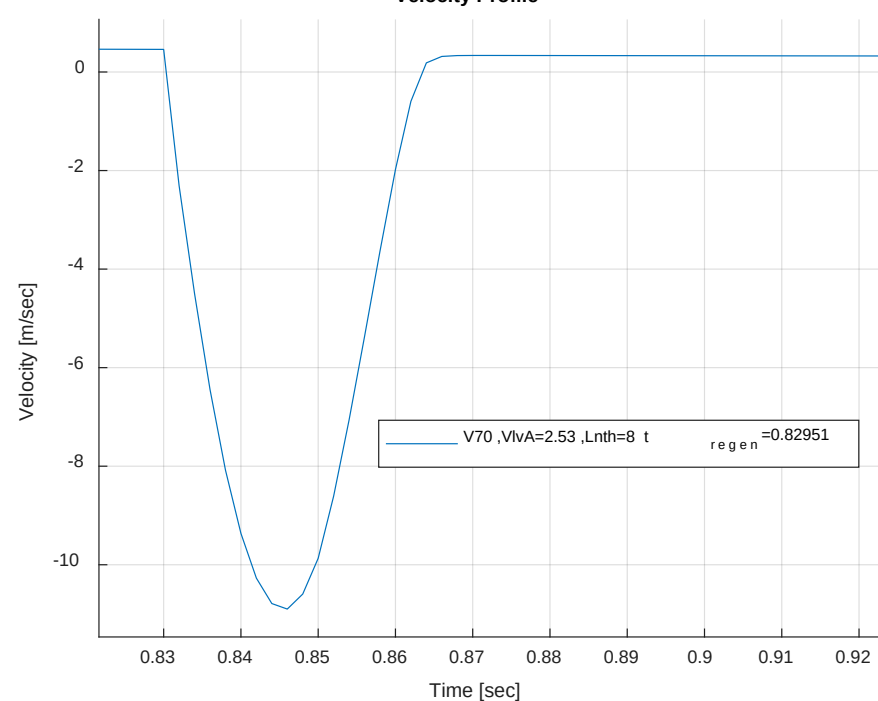


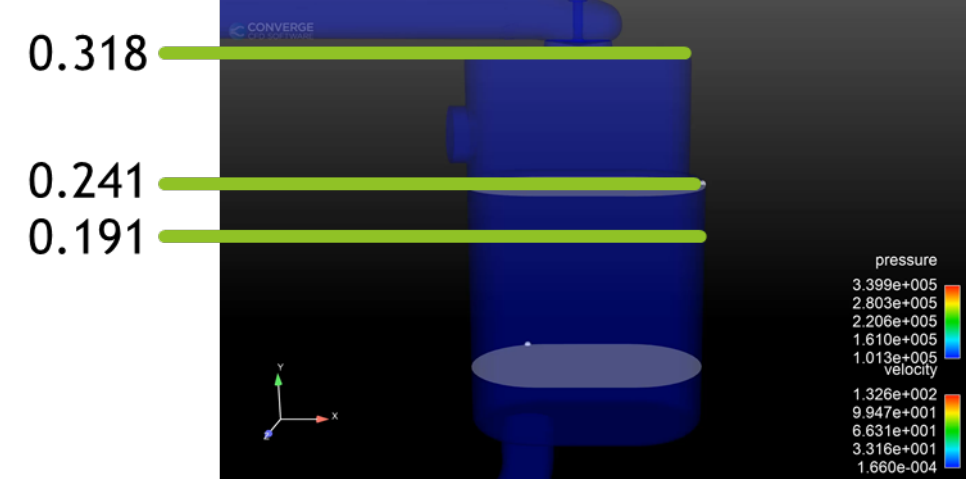
Test 2 Baseline Arrangement with Increased Pressure

340kPa (35psi) 10.2l (2.75gal) Holding

Base Holding Tank Size (10L)

Regeneration through an Annulus Design

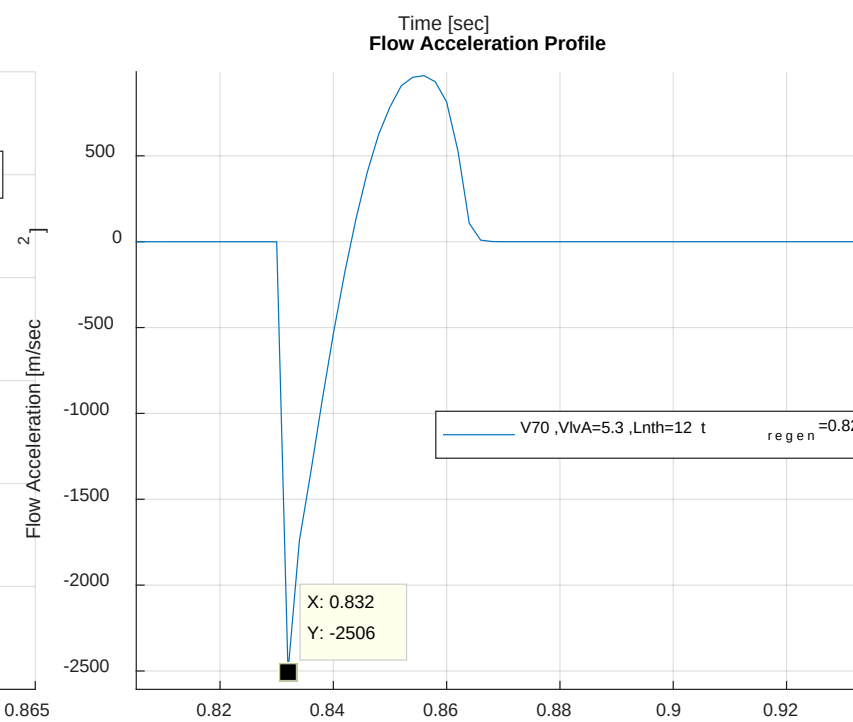
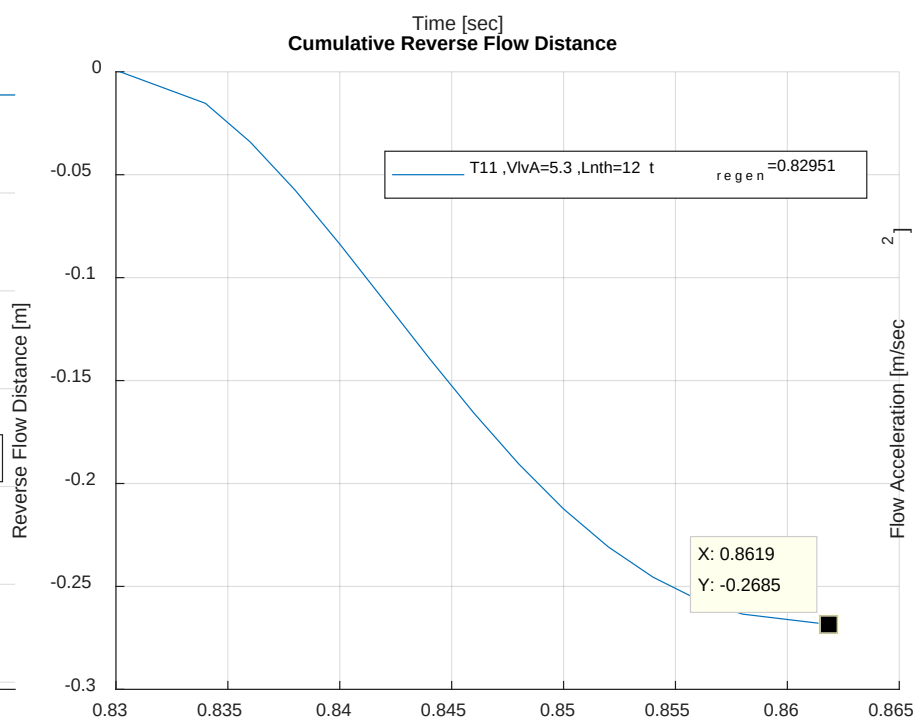
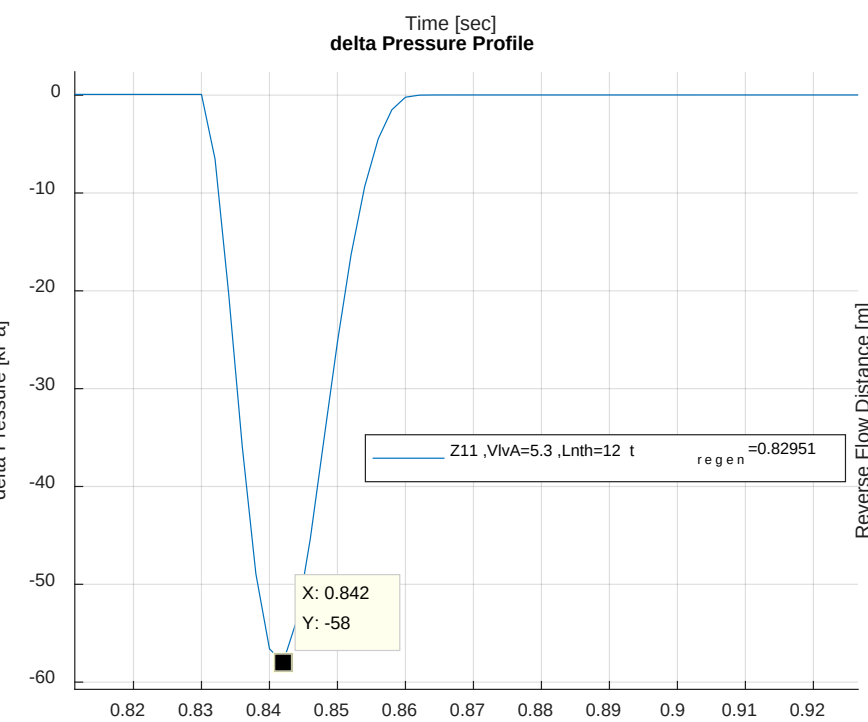
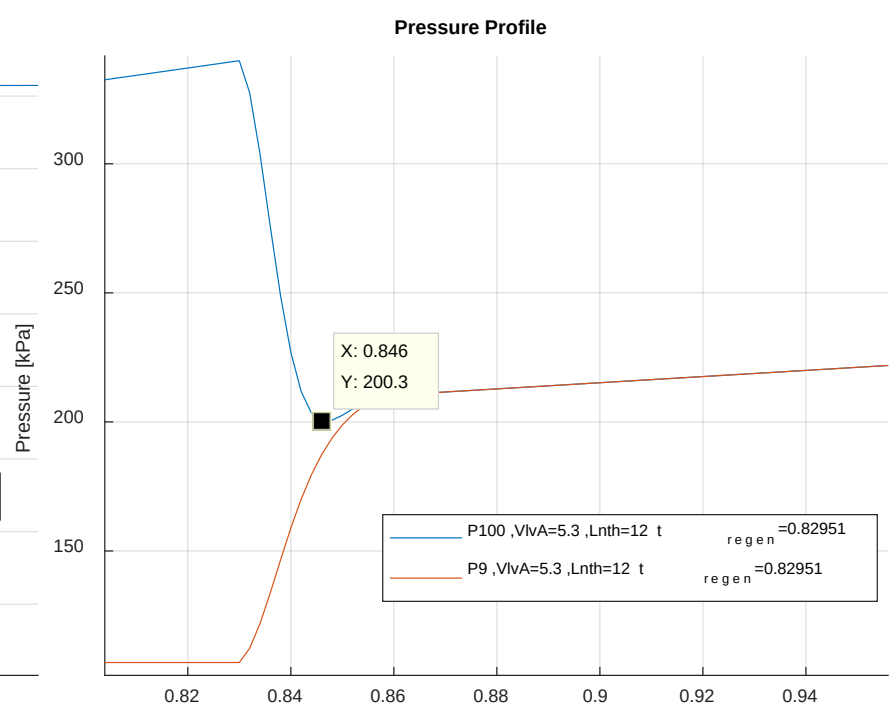
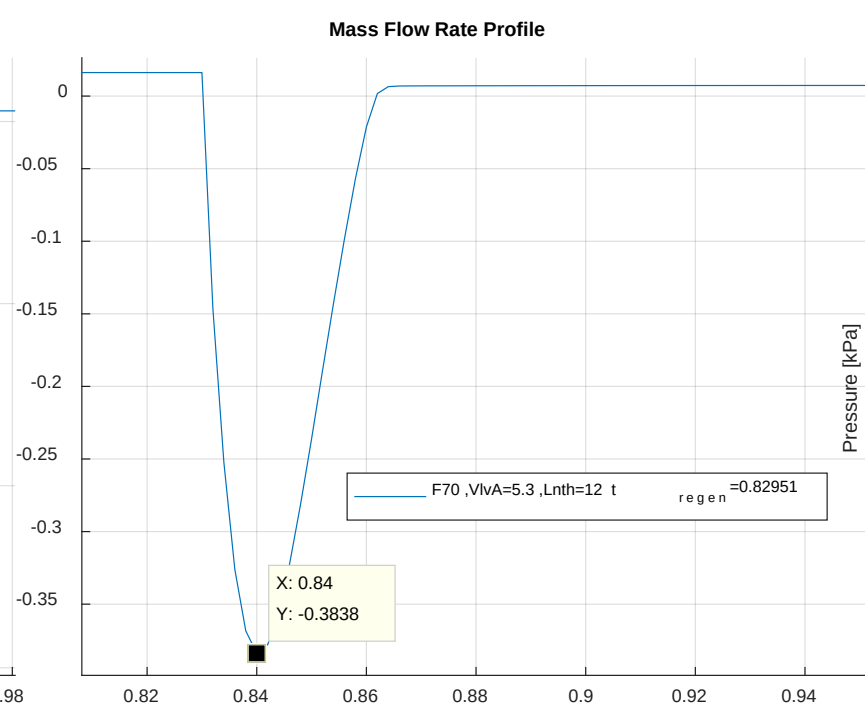
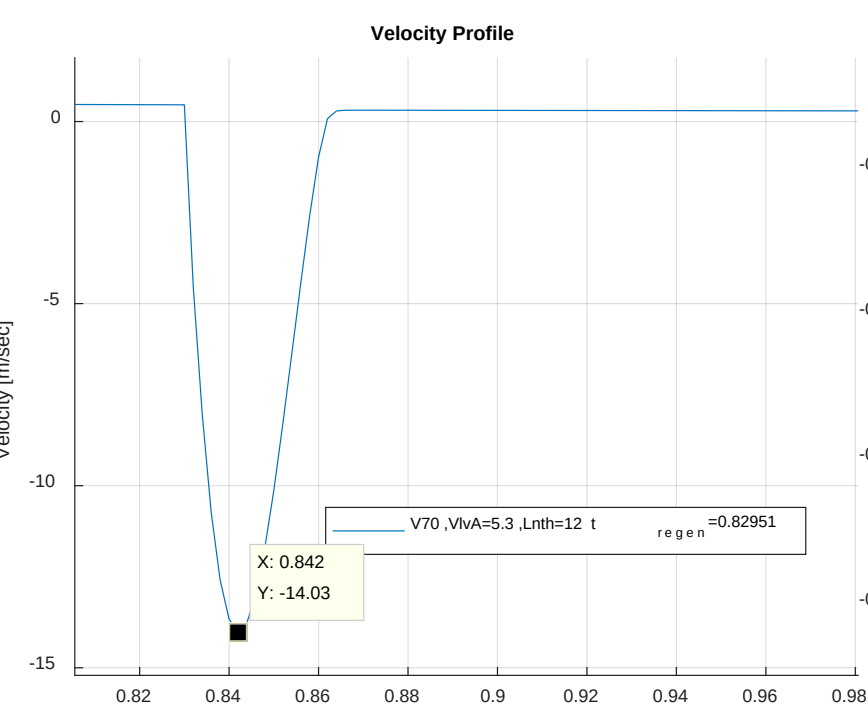


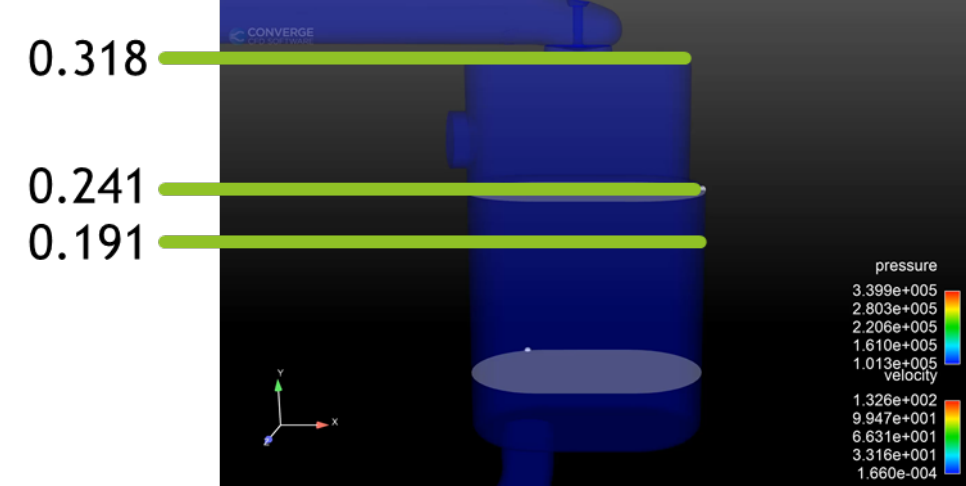


Test 3 340kPa (35psi) 15.4l (4gal) Holding

Base Holding Tank Size (10L)

Regeneration through an Annulus Design



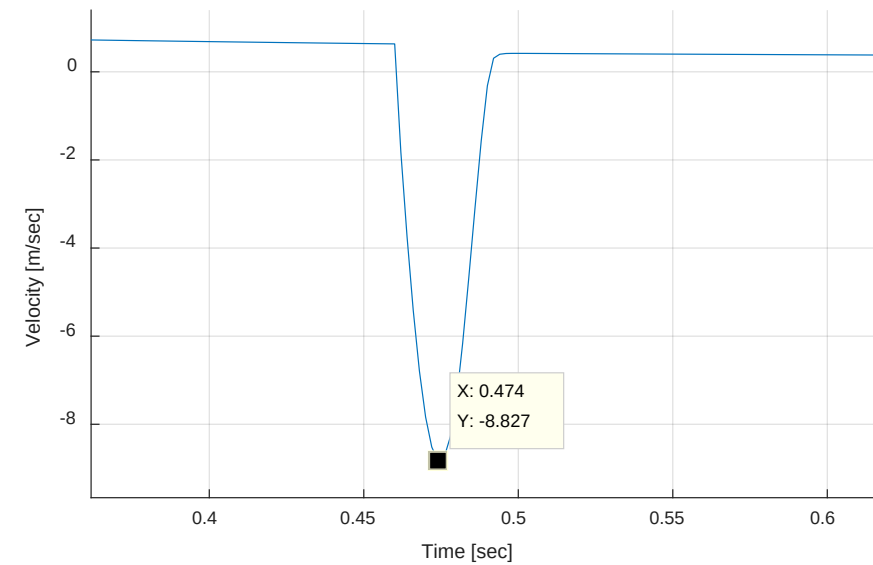


Test 4 240kPa (20psi) 10l Holding

Base Holding Tank Size (10L)

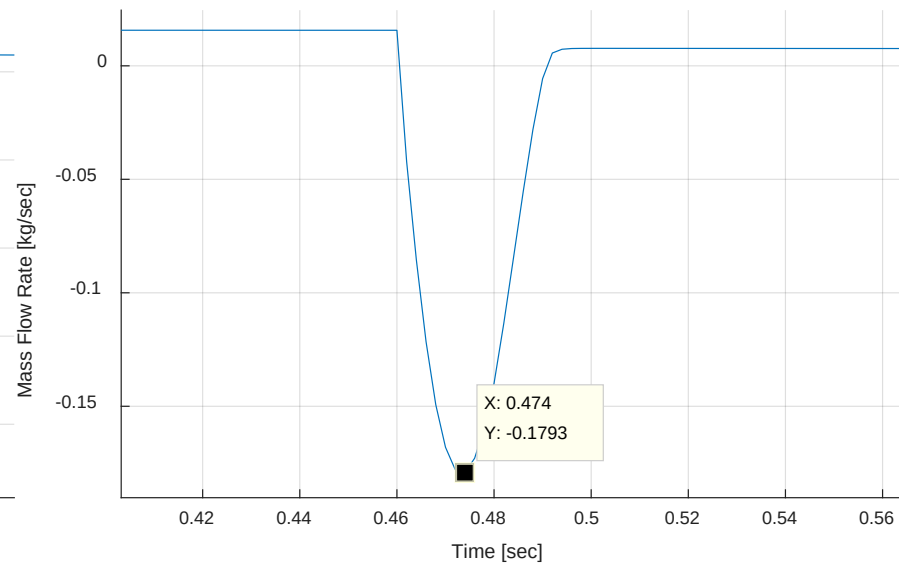
Regeneration through an Annulus Design

Velocity Profile



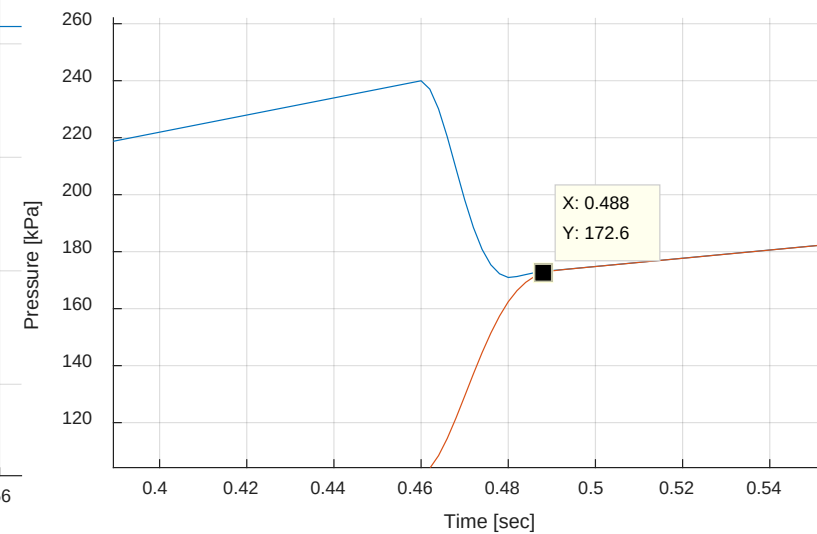
V70 ,VlvA=2.58 ,Lnth=8 t_{regen}=0.4591

Mass Flow Rate Profile



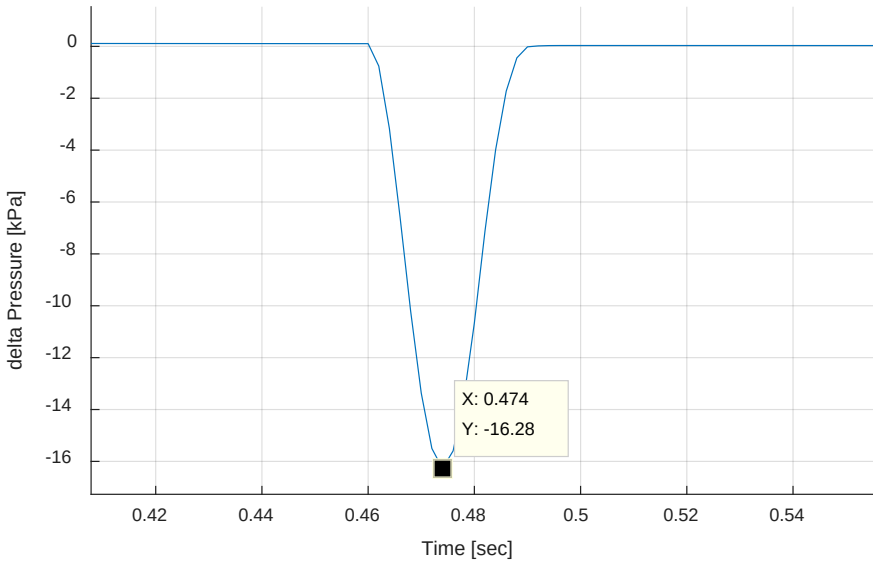
F70 ,VlvA=2.58 ,Lnth=8 t_{regen}=0.4591

Pressure Profile



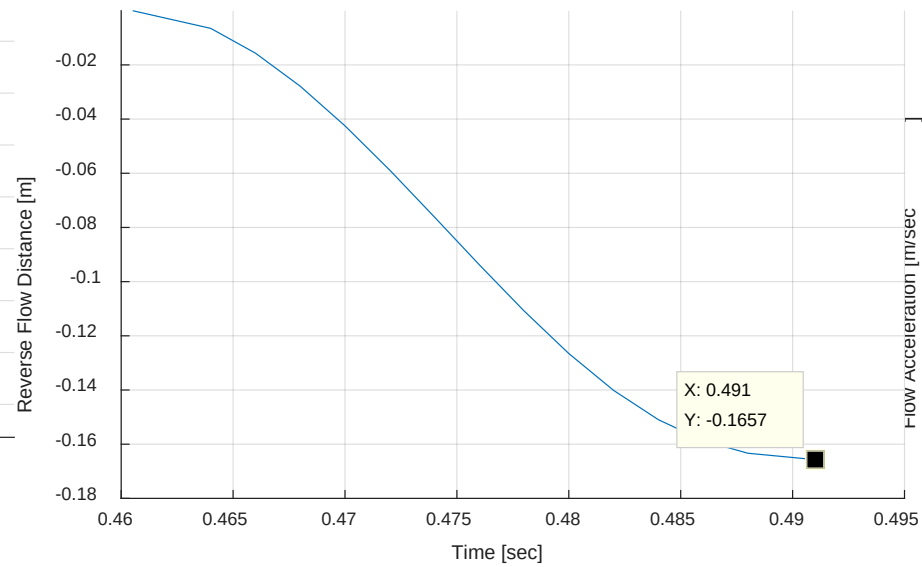
P3 ,VlvA=2.58 ,Lnth=8 t_{regen}=0.4591
P9 ,VlvA=2.58 ,Lnth=8 t_{regen}=0.4591

delta Pressure Profile



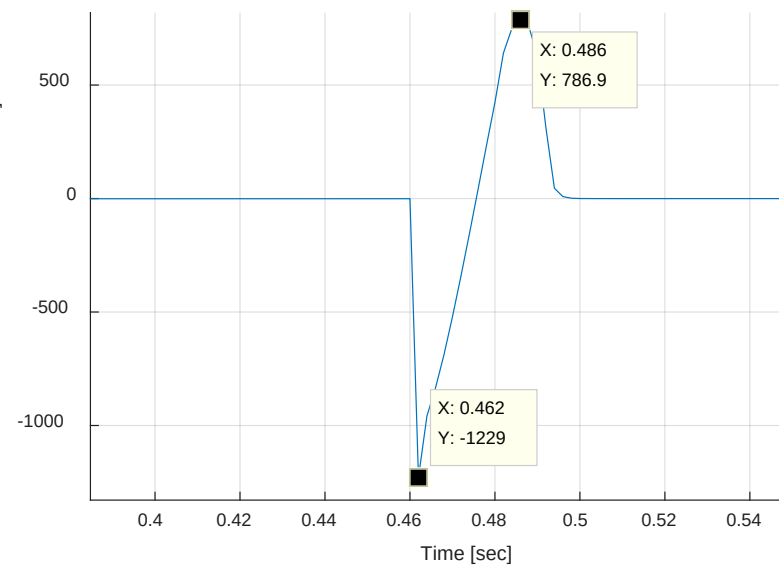
Z11 ,VlvA=2.58 ,Lnth=8 t_{regen}=0.4591

Cumulative Reverse Flow Distance

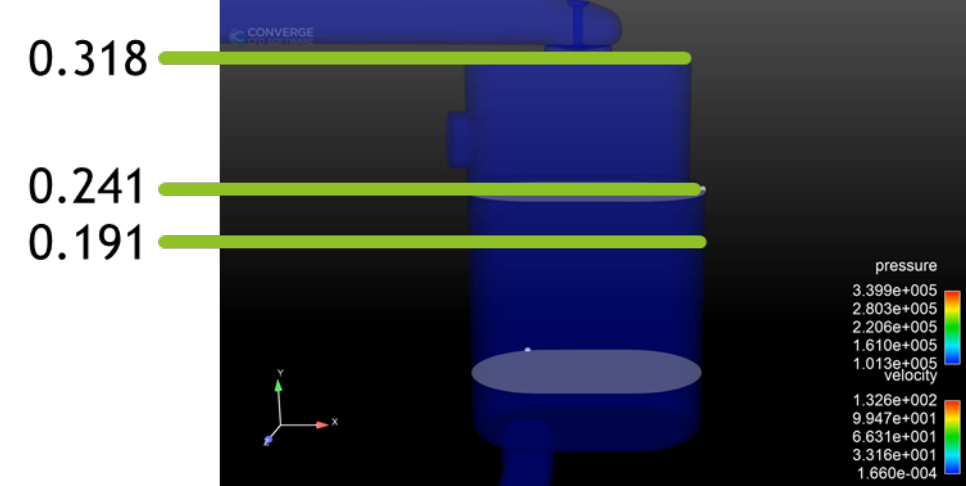


T11 ,VlvA=2.58 ,Lnth=8 t_{regen}=0.4591

Flow Acceleration Profile

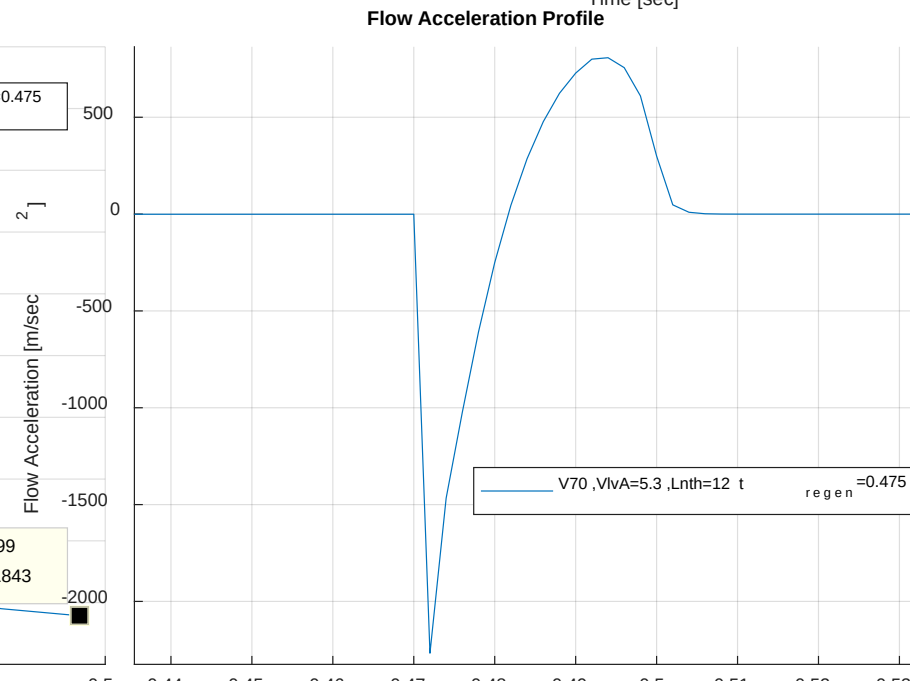
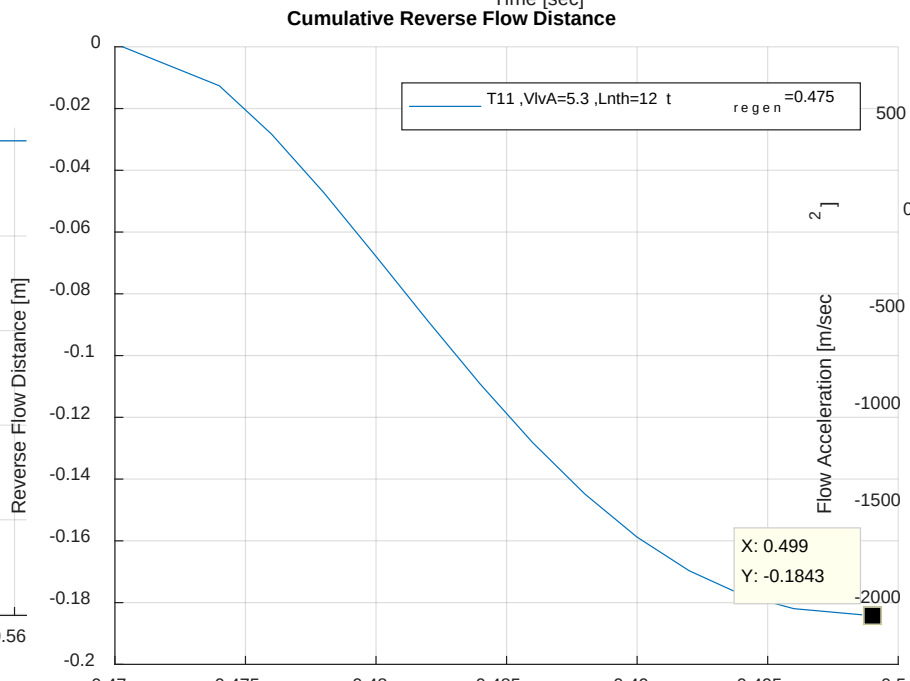
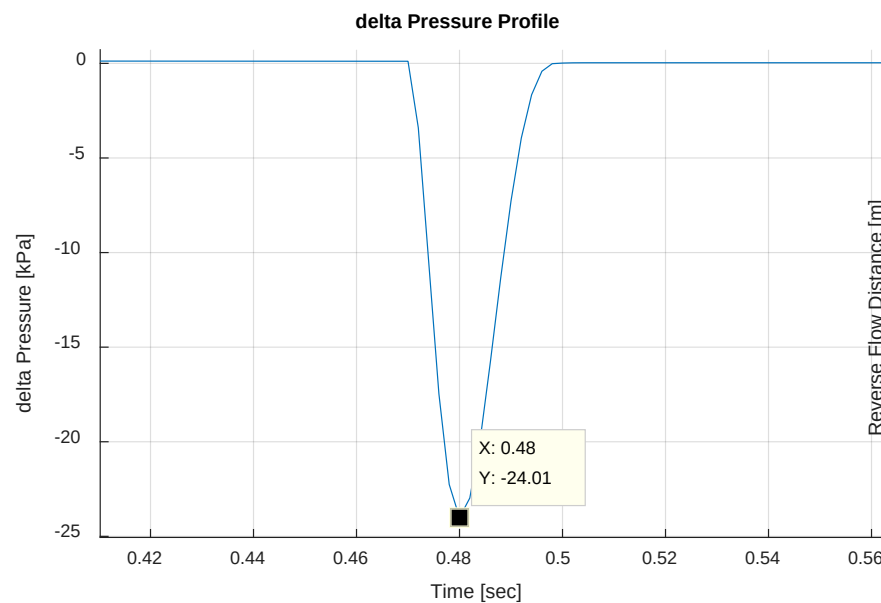
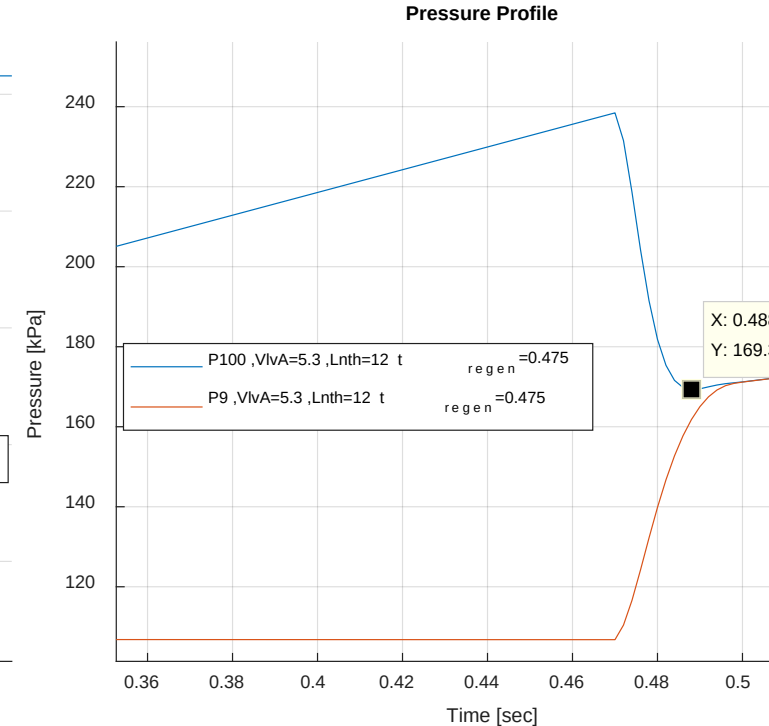
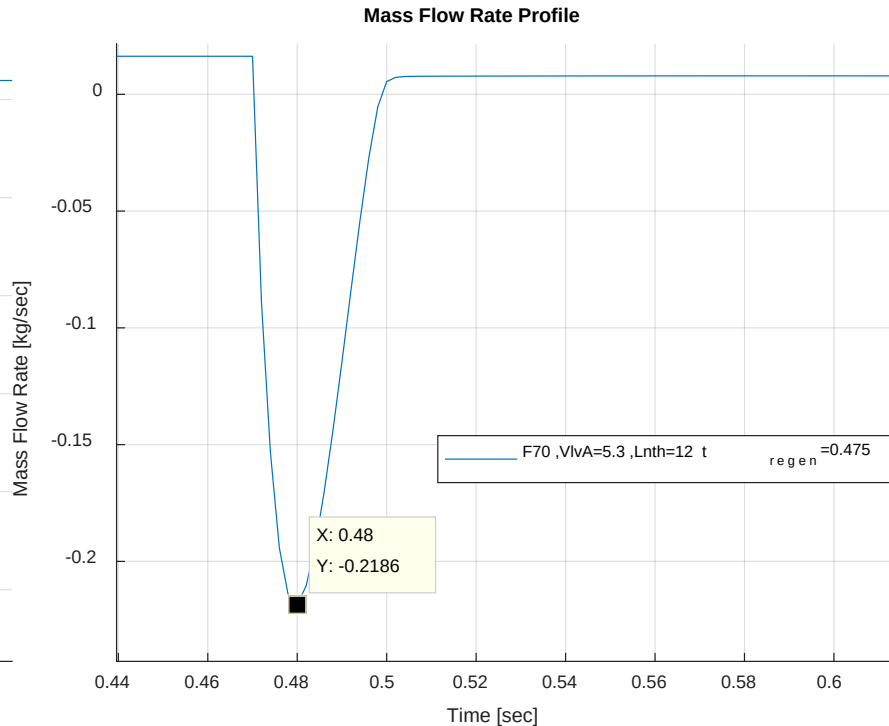
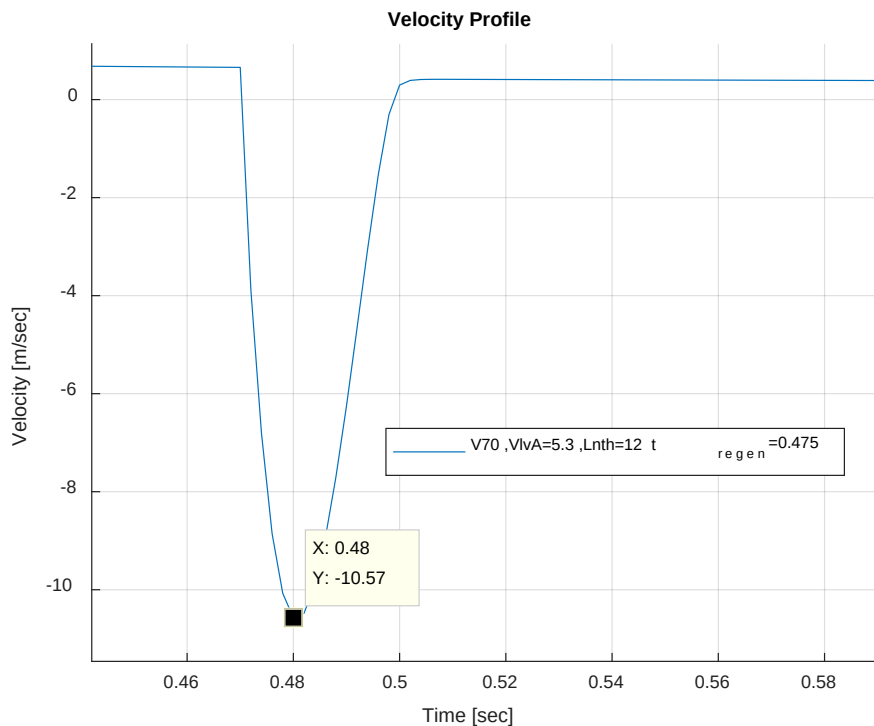


V70 ,VlvA=2.58 ,Lnth=8 t_{regen}=0.4591



Test 5 240kPa (20psi) 15.4l Holding

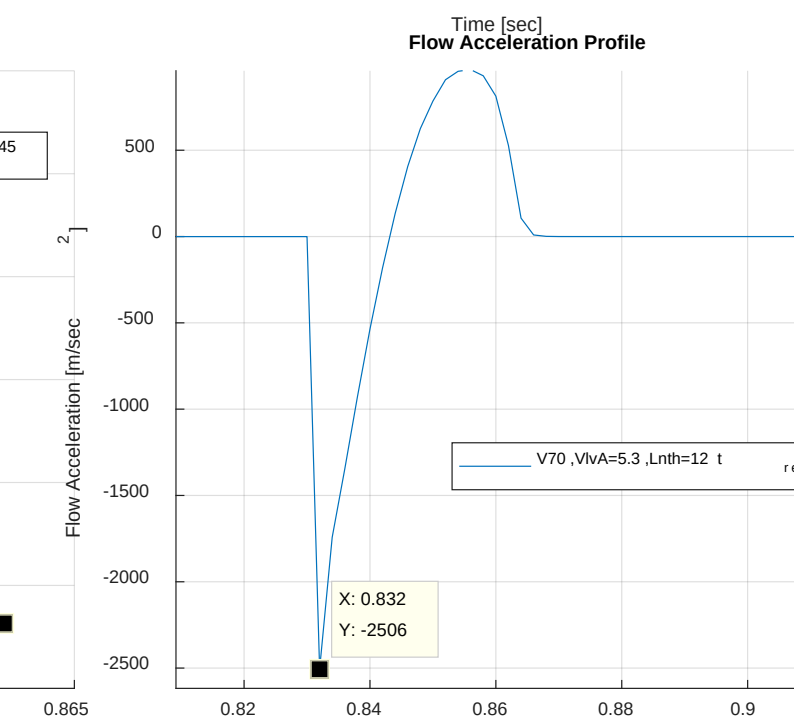
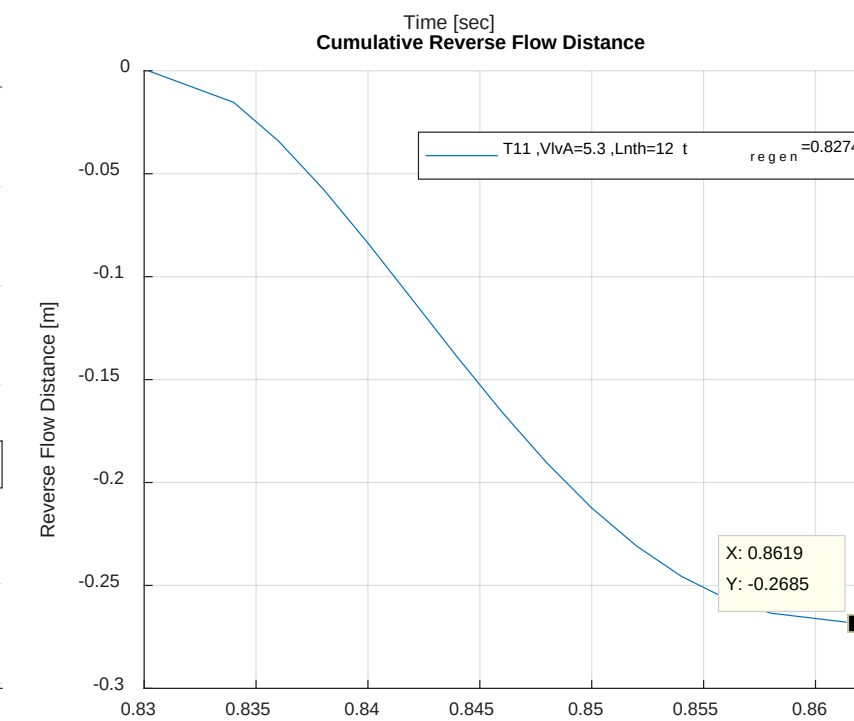
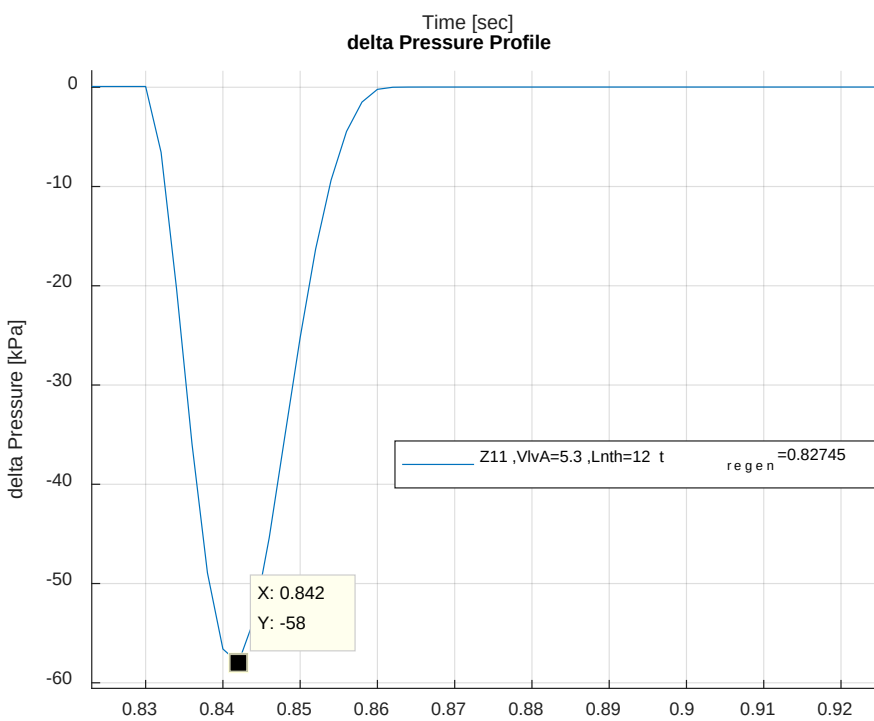
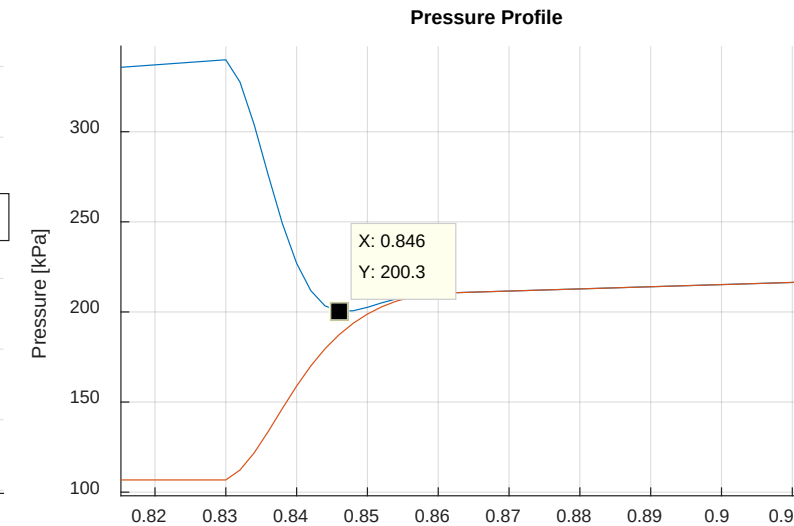
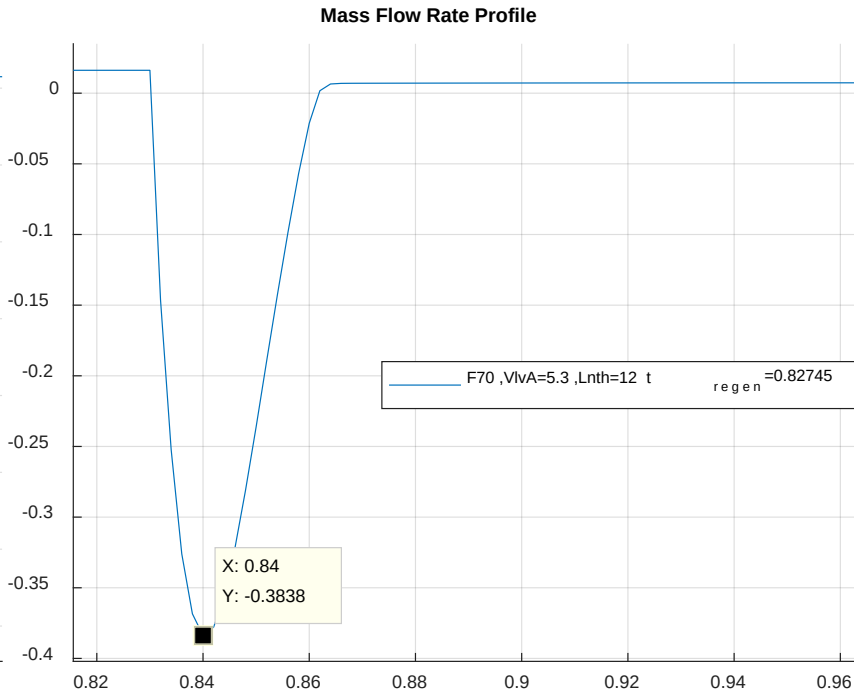
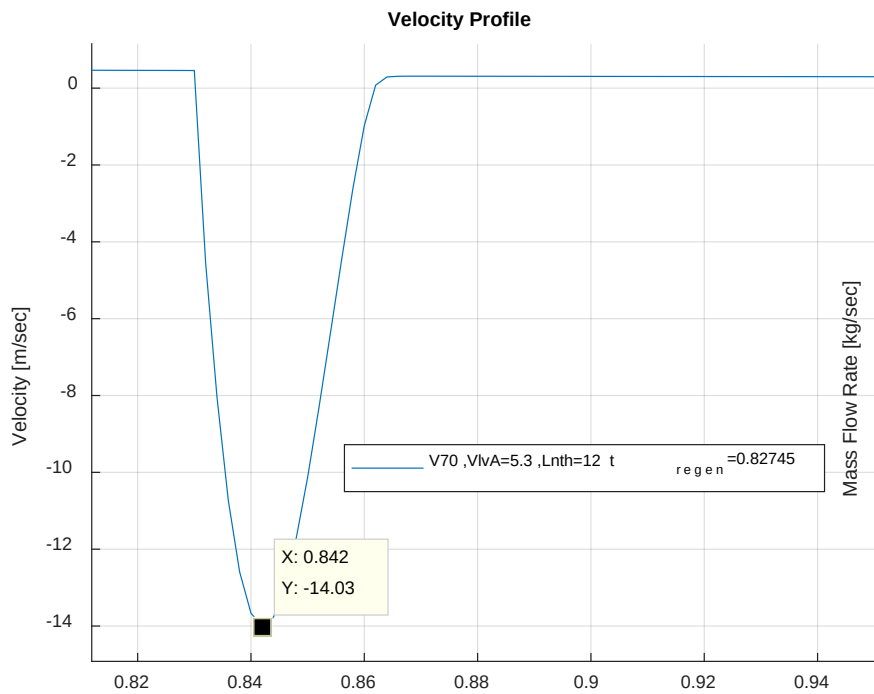
Base Holding Tank Size 15.4L (4 gal)



Baseline Test 6 340kPa (35psi) Regen Pres

66mm Regeneration Valve Diameter

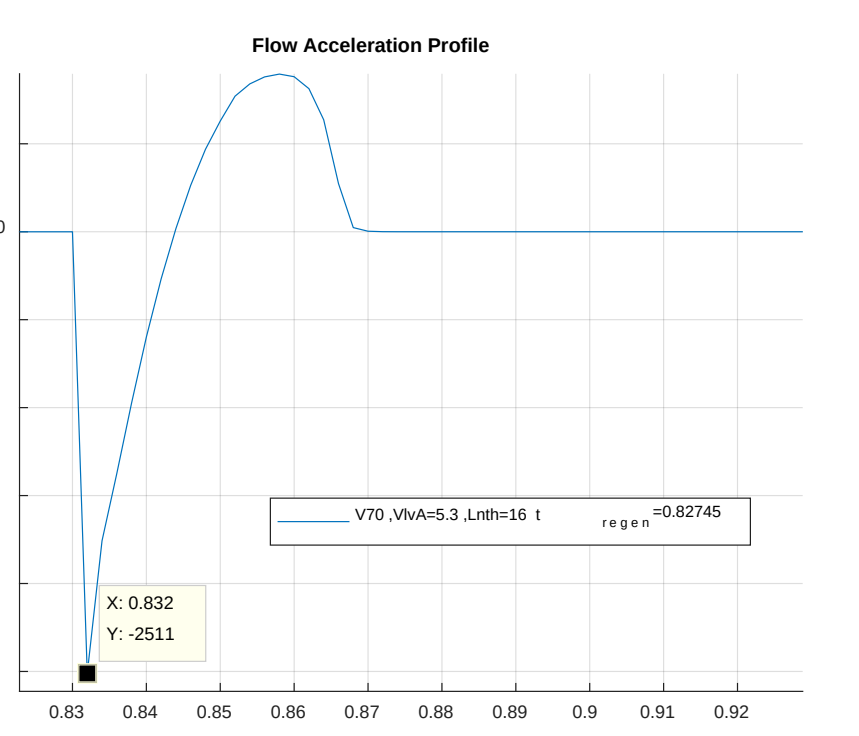
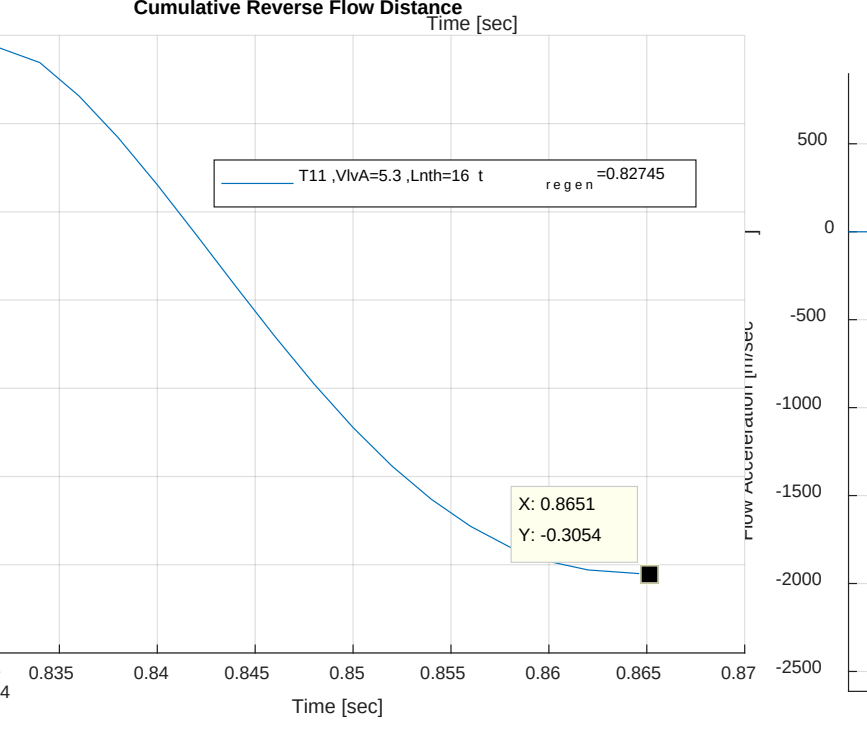
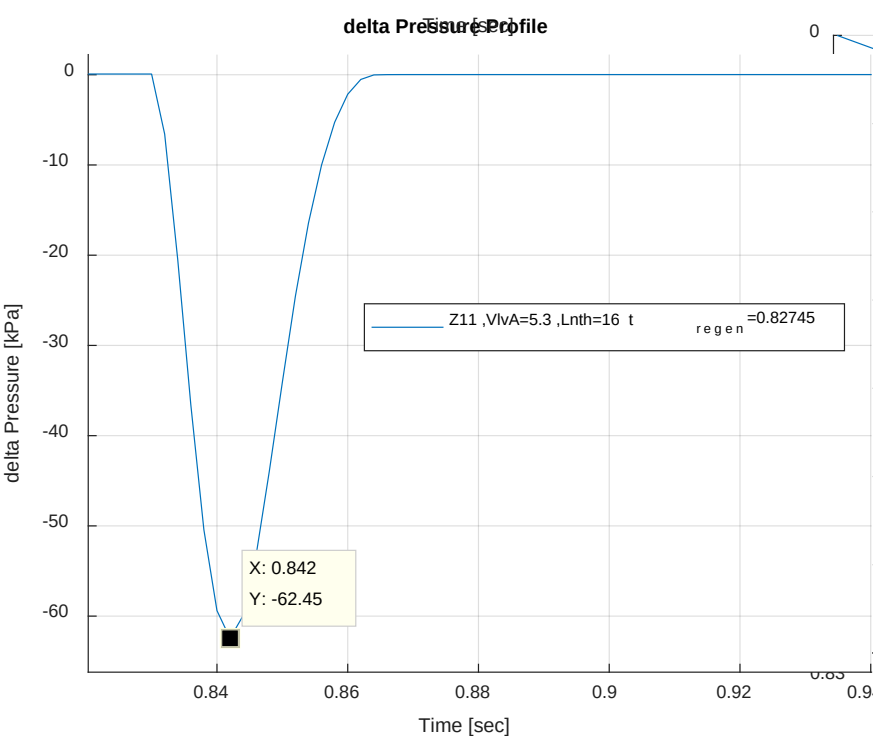
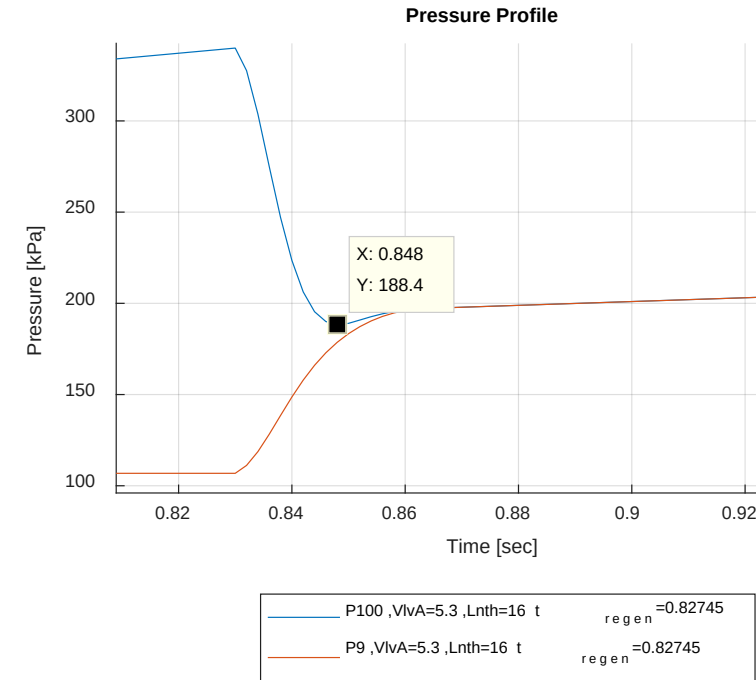
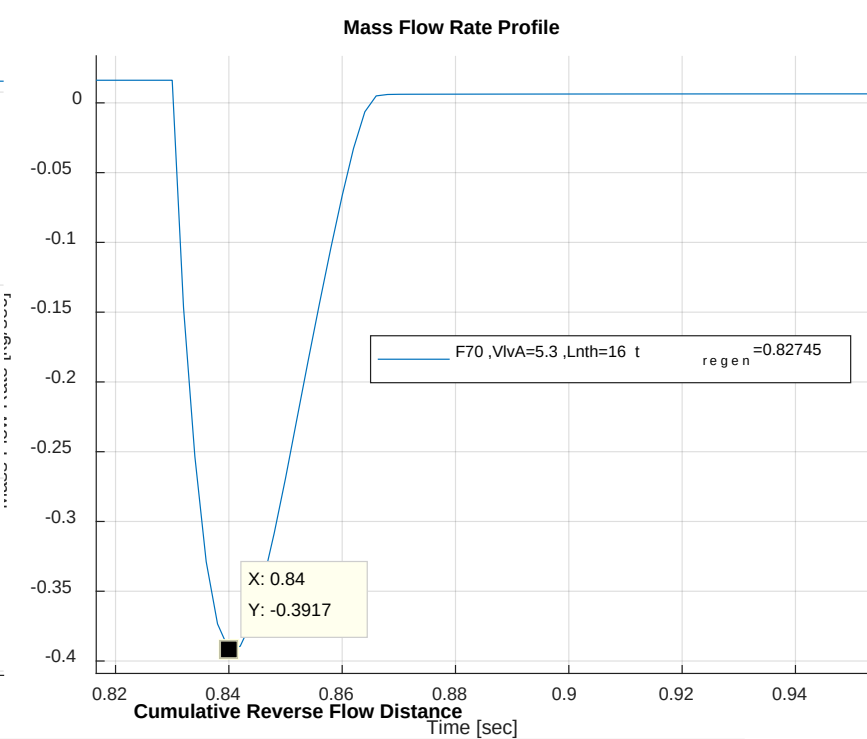
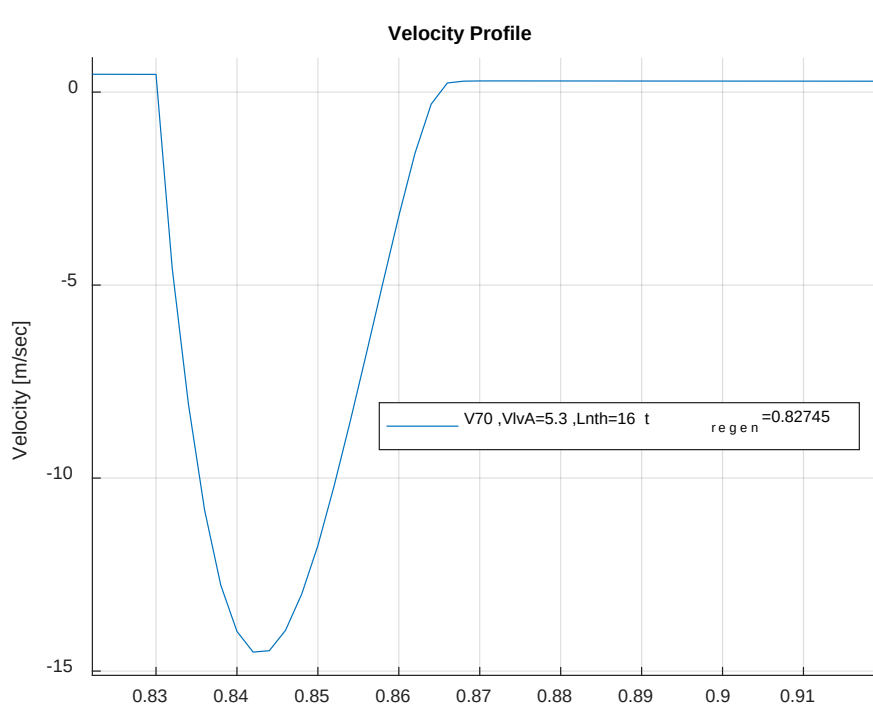
15.4 (4 gal) Holding Tank



Baseline 340kPa (35psi) Regen Pres

66mm Regeneration Valve Diameter

20.6 (5.5gal) Holding Tank

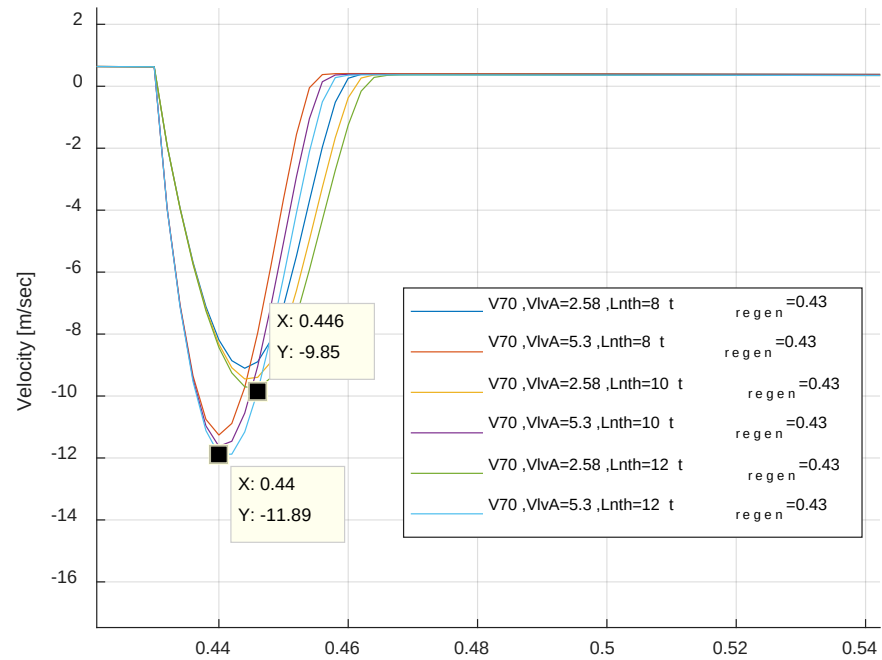


Baseline 140kPa (20psi) Regen Pres

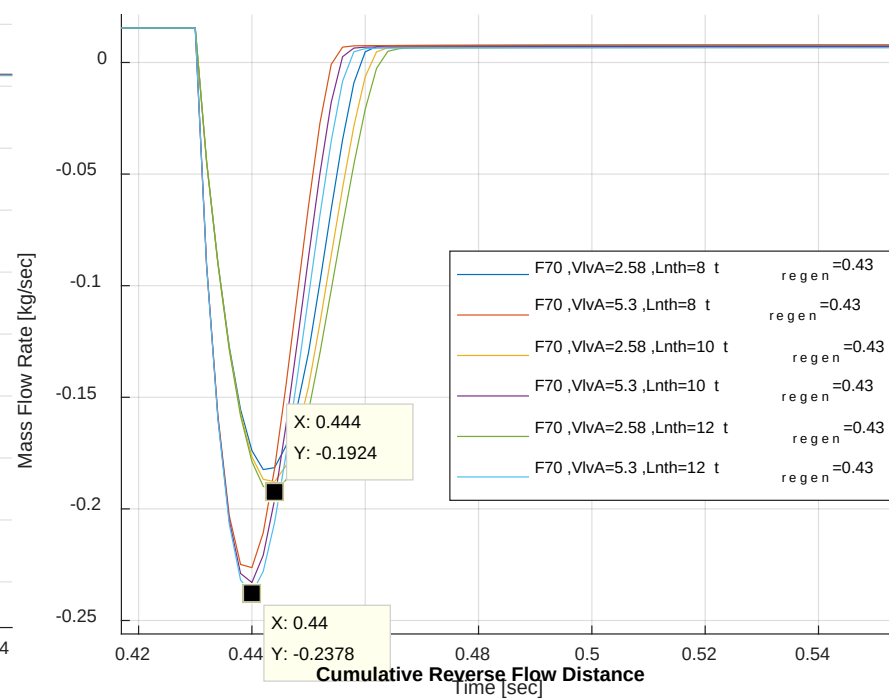
46 & 66mm Regeneration Valve Diameters

10.3L, 12.8, 15.4 (2.75, 3.4, & 4 gal) Holding Tank

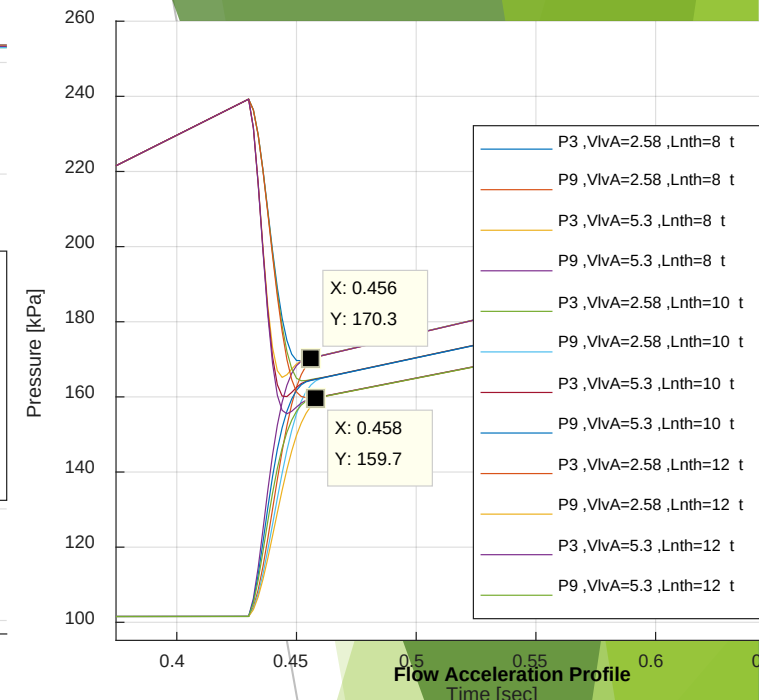
Velocity Profile



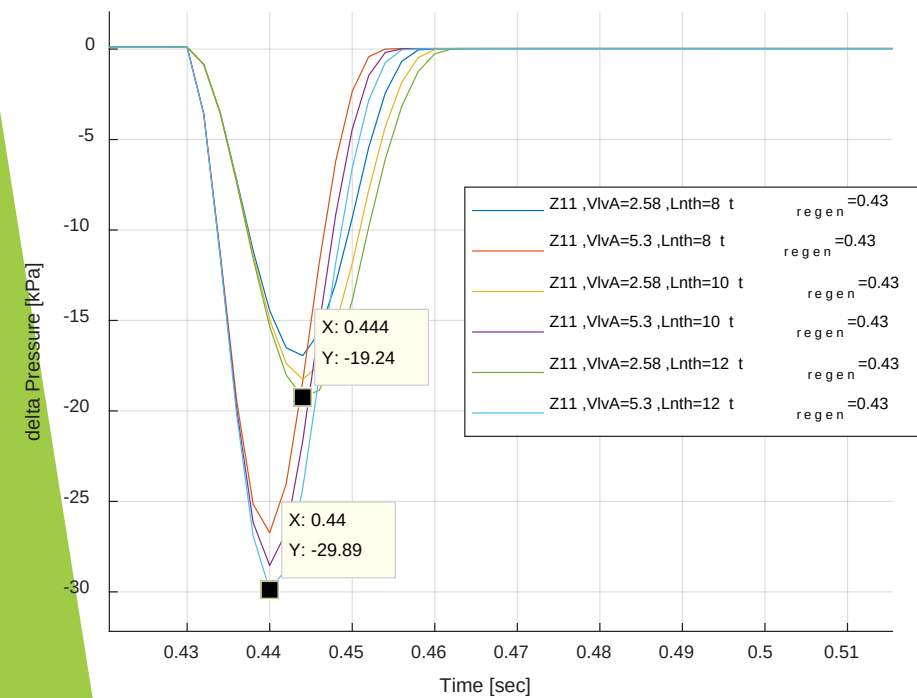
Mass Flow Rate Profile



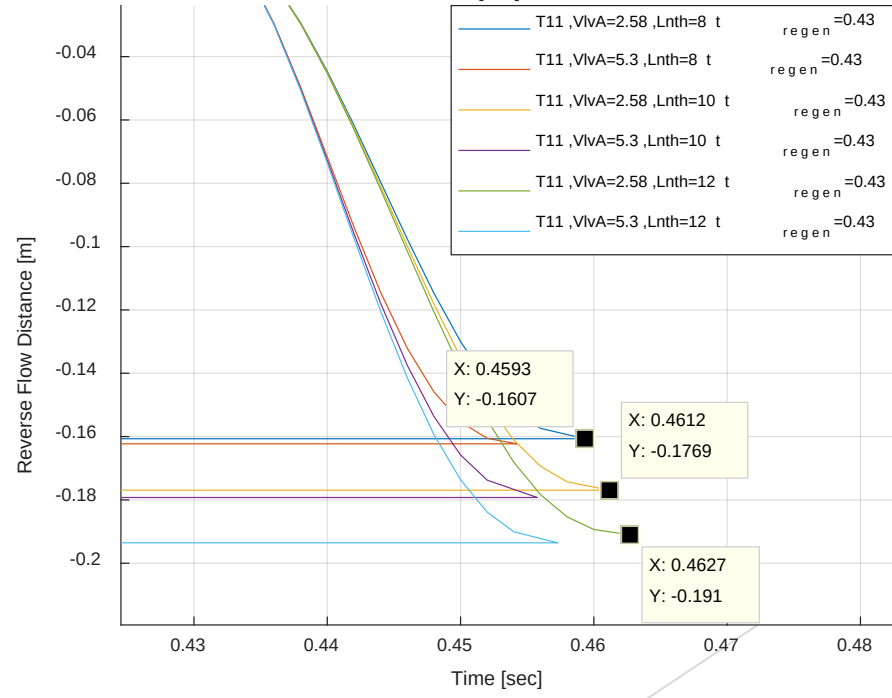
Pressure Profile



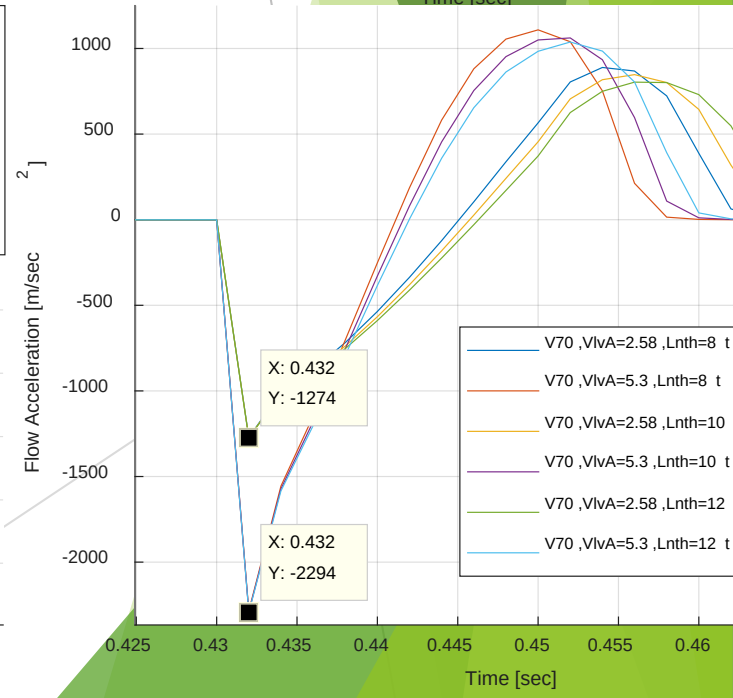
delta Pressure Profile

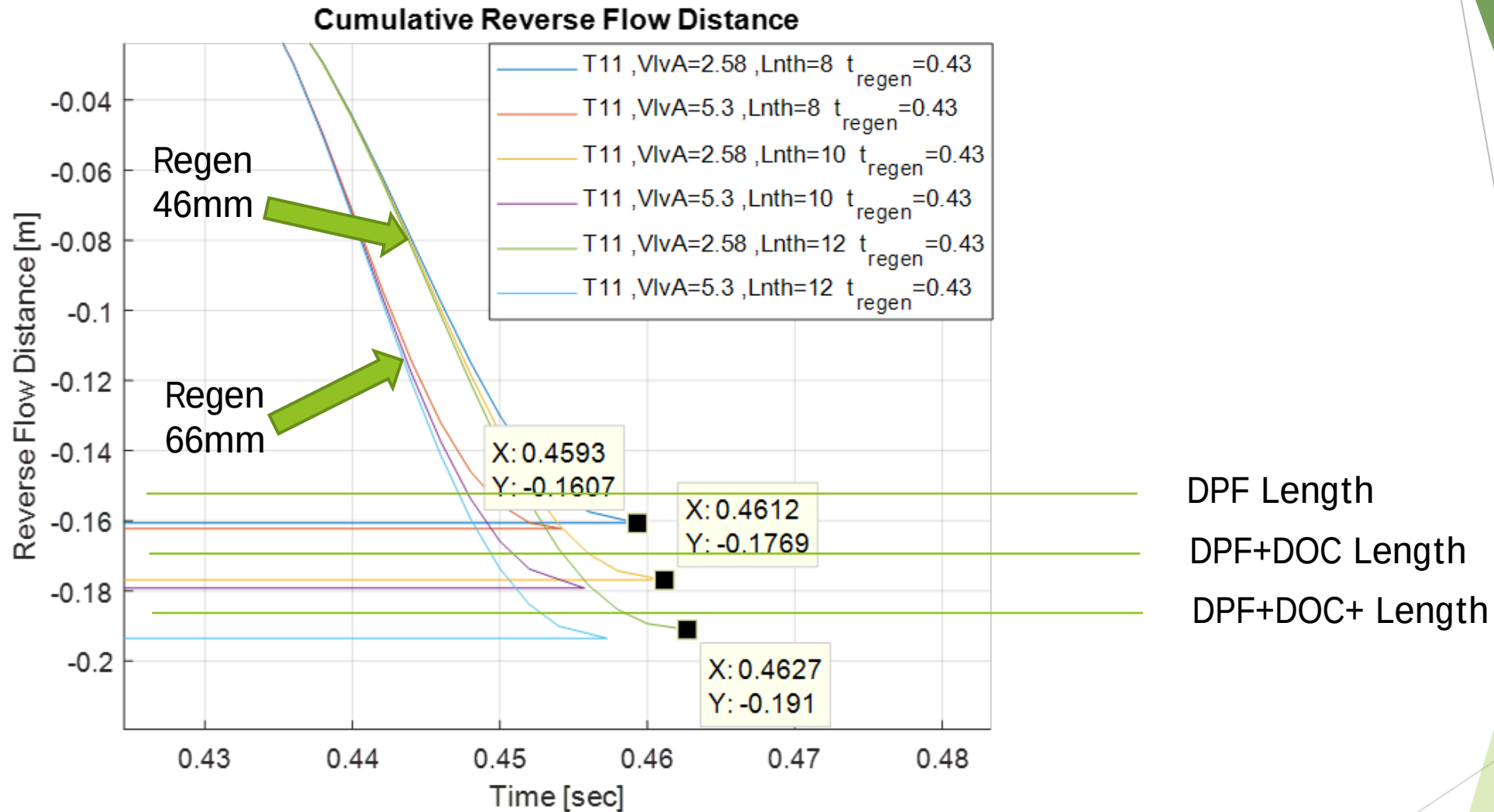


Cumulative Reverse Flow Distance



Flow Acceleration Profile





Lnth=8 - 10.3l (2.7gal) Tank

Lnth=10 - 15l (3.4 gal) Tank

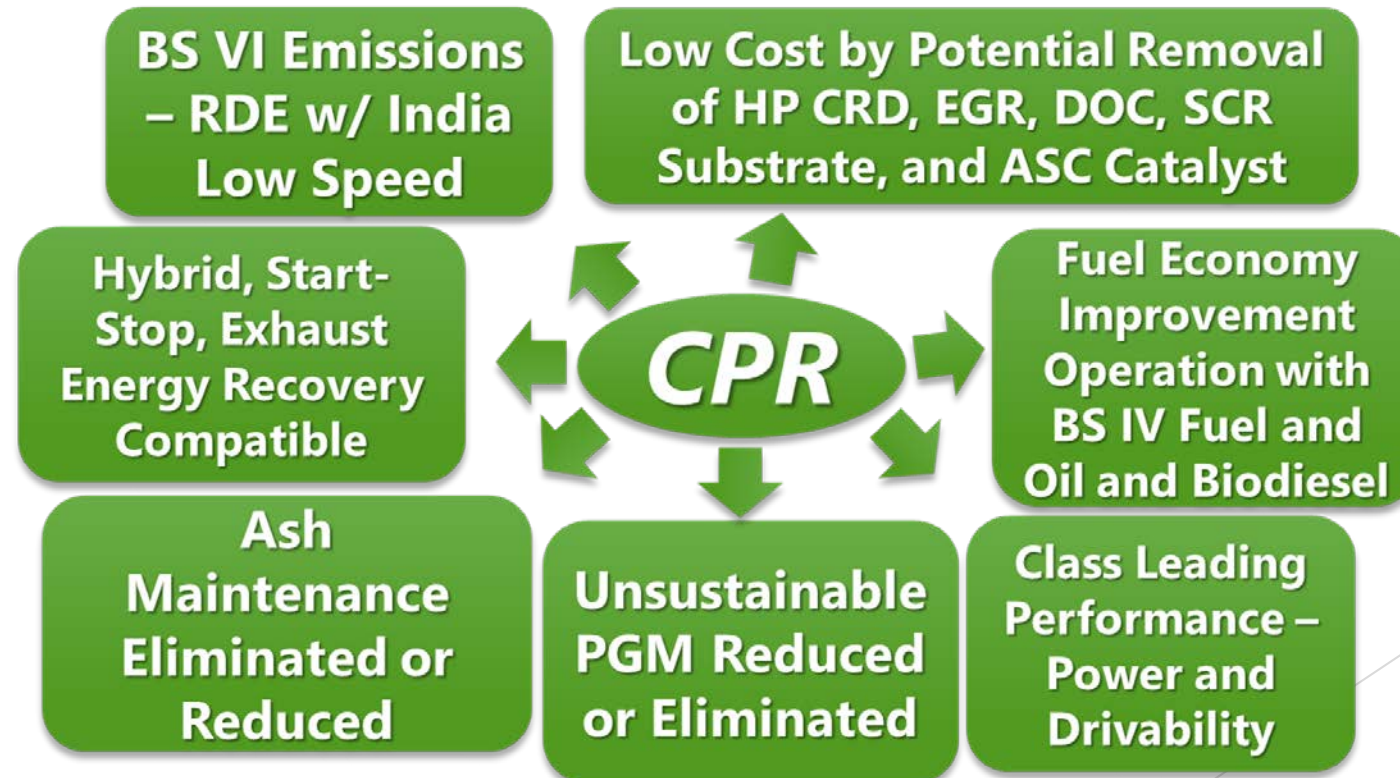
Lnth=12 - 15l (4gal) Tank

Conclusions

- ▶ **Retrofitting of millions of Volkswagen Jetta TDI and similar vehicles is possible**
 - ▶ **Power, Response, Emissions, and Efficiency can be achieved**
- ▶ **3D Modeling Demonstrating Flow Concerns**
- ▶ **1D Modeling Demonstrating Need for Reduced Translation Distance**

India Bharat VI Emissions Opportunity for Ford Motor Company

- ▶ India is a double digit growth market for Ford. It is a cost driven market requiring simple maintenance.
- ▶ Thermal Regeneration Expected to be Challenged by Low Speeds and Start/Stop Operation of India
- ▶ Success in India Replicated in Europe and US Diesel Hybrids



Please Contact for Additional Information

Contact Information:

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President and CEO

IVHCO

Mobile 239-284-9992

Email: brett.bailey@ivhco.com

Web: www.ivhco.com

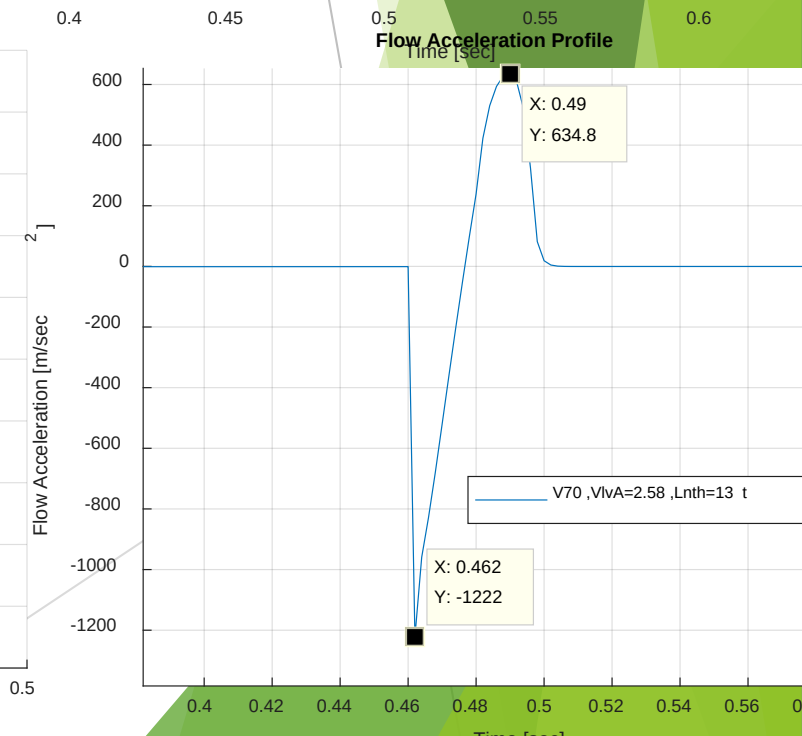
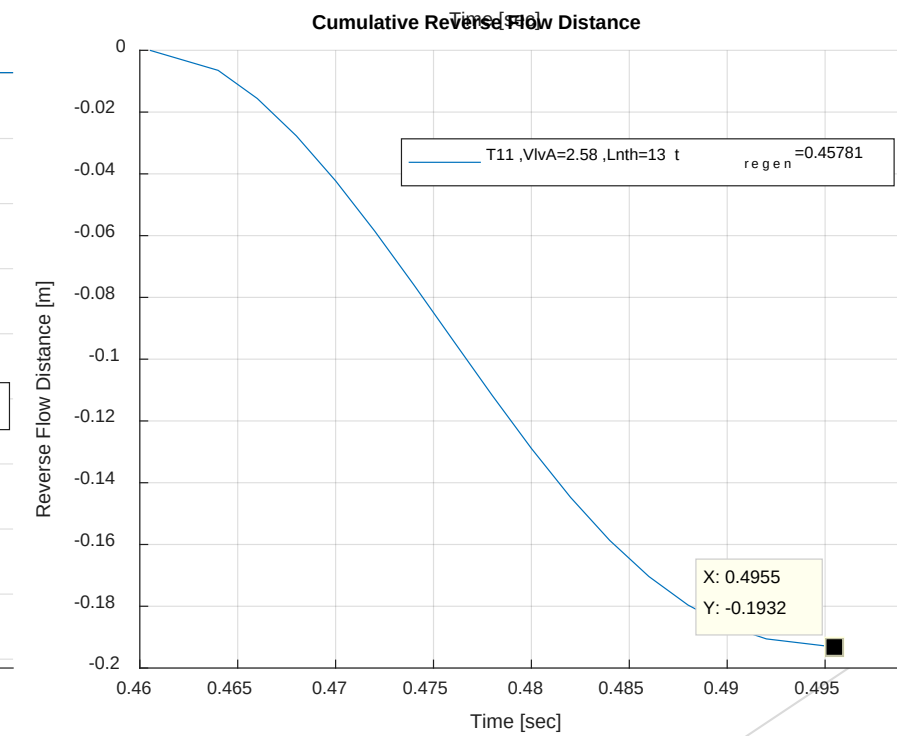
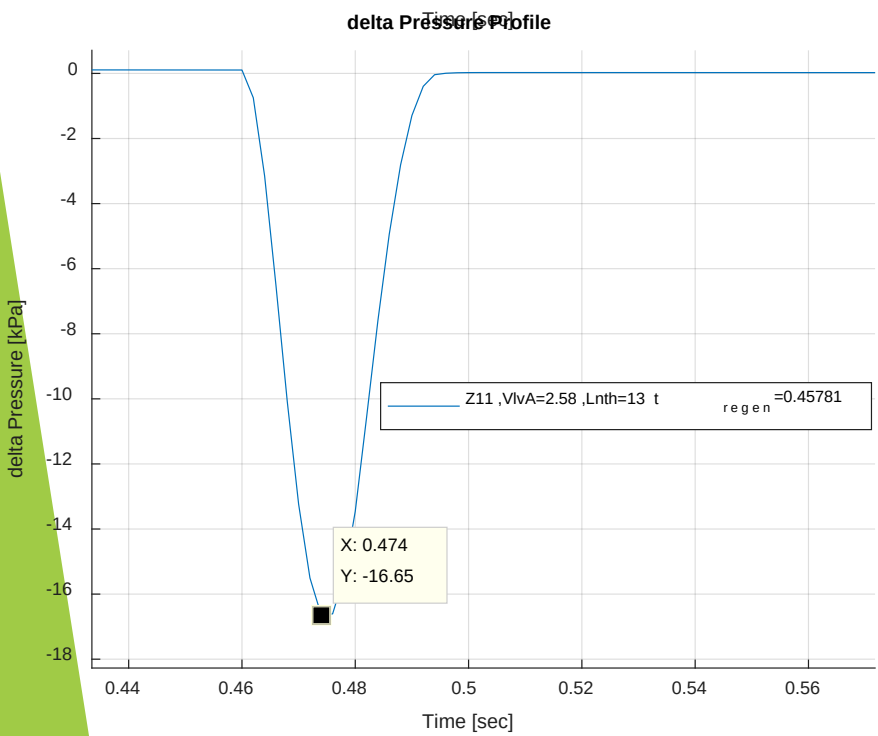
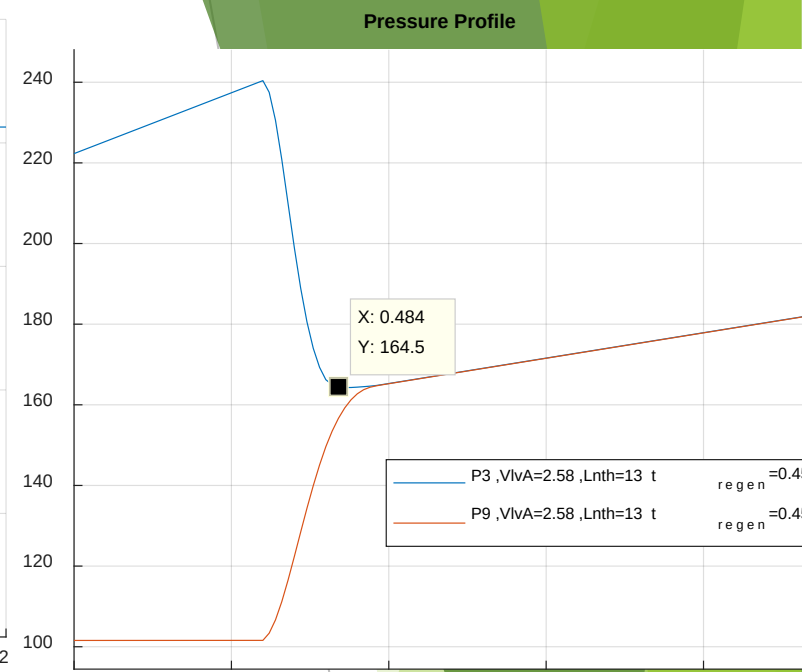
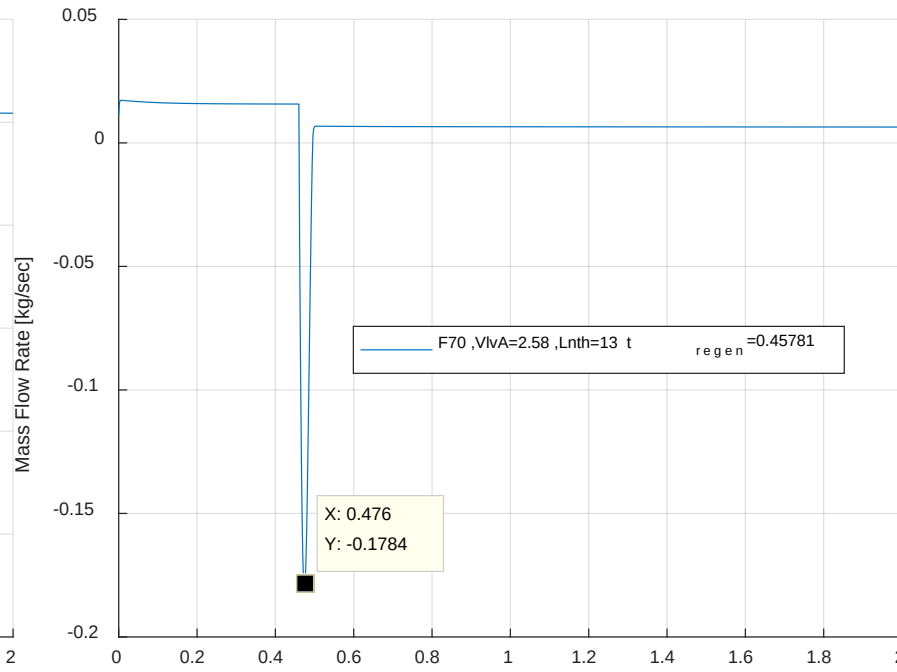
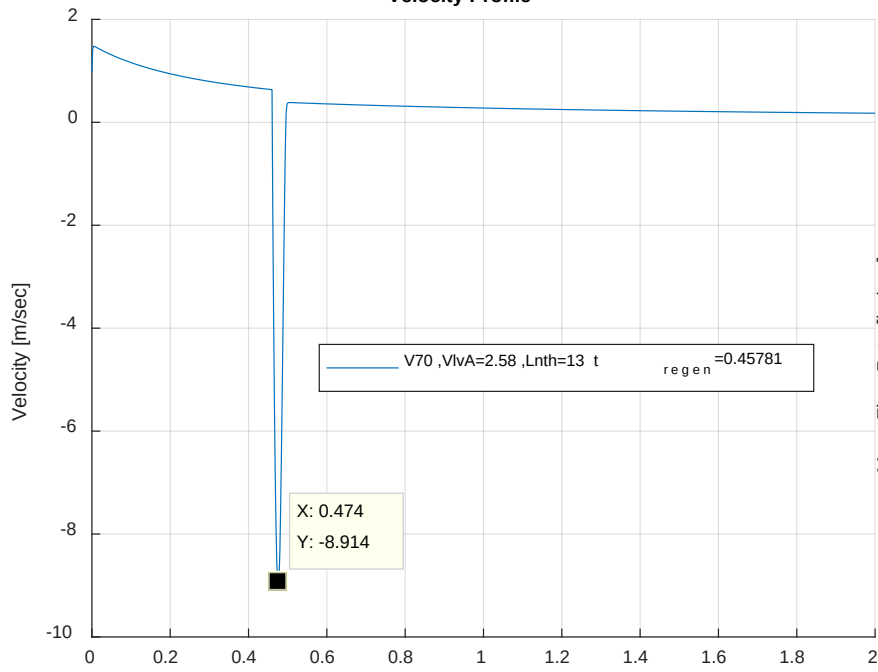


240kPa Slightly Oversized Settling Tank

Baseline without Vacuum

17 L (4.5 Gal) Tank

46mm Valve



Demonstration of Acceptable EU Retrofit 240kPa Slightly Oversized Settling Tank Plugging of Filter and CPR Adaptability

17 L (4.5 Gal) Tank

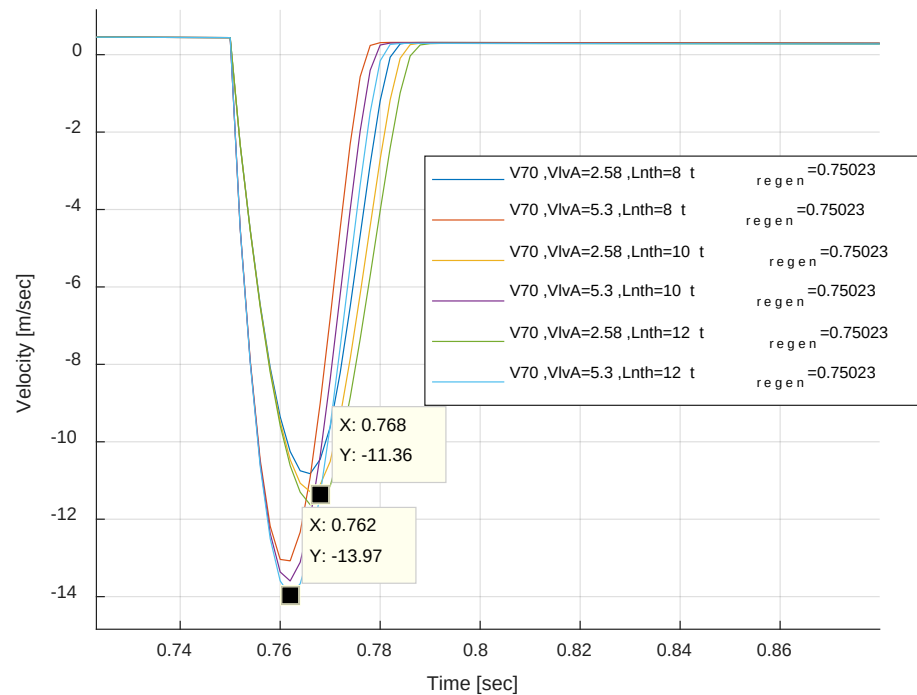
46mm Valve

250kPa (35psi) Regen Pres

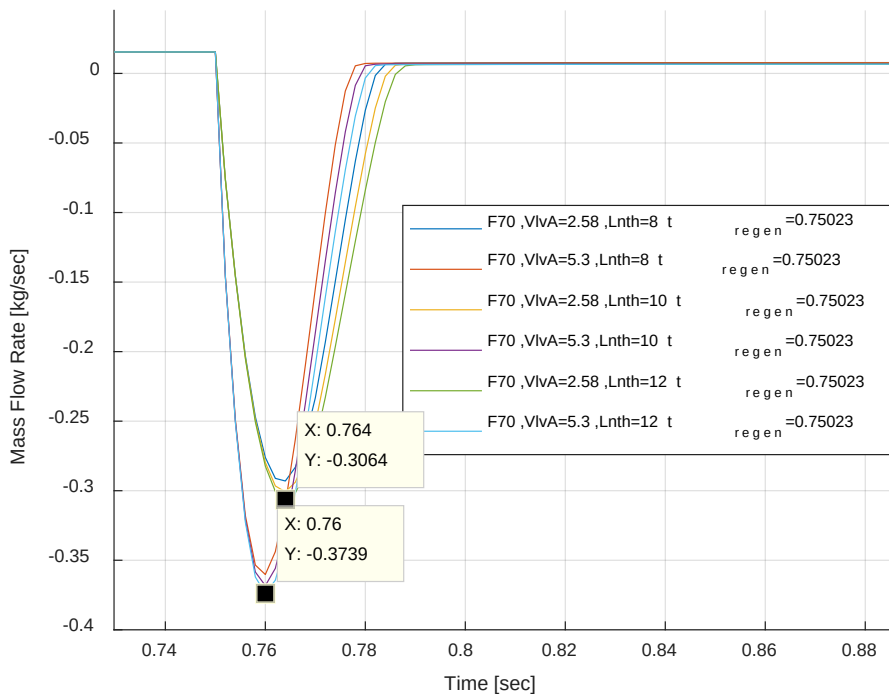
46 & 66mm Regeneration Valve Diameters

10.3L, 12.8, 15.4 (2.75, 3.4, & 4 gal) Holding Tank

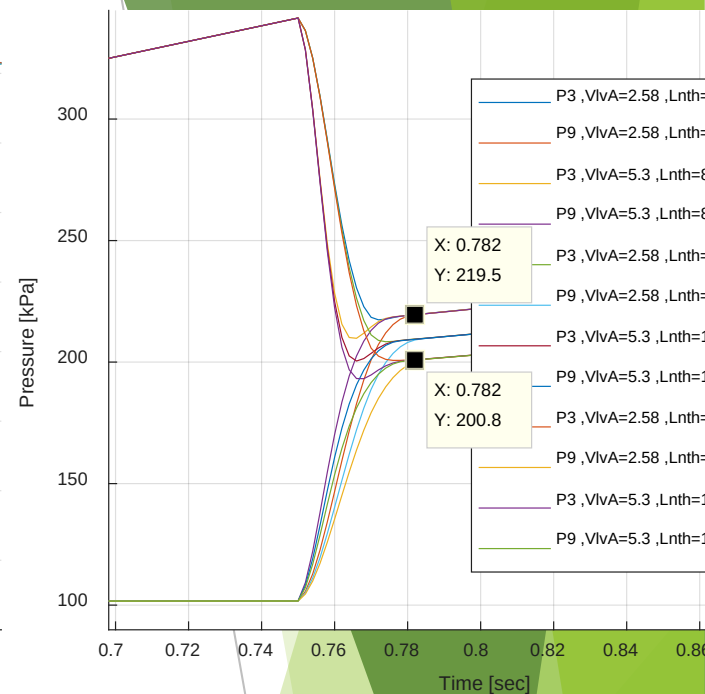
Velocity Profile



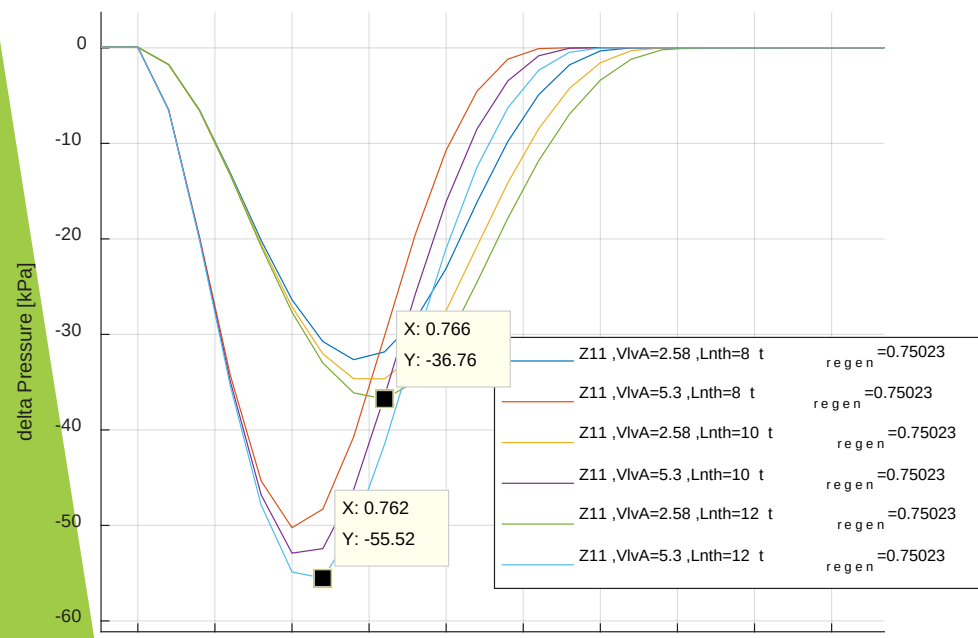
Mass Flow Rate Profile



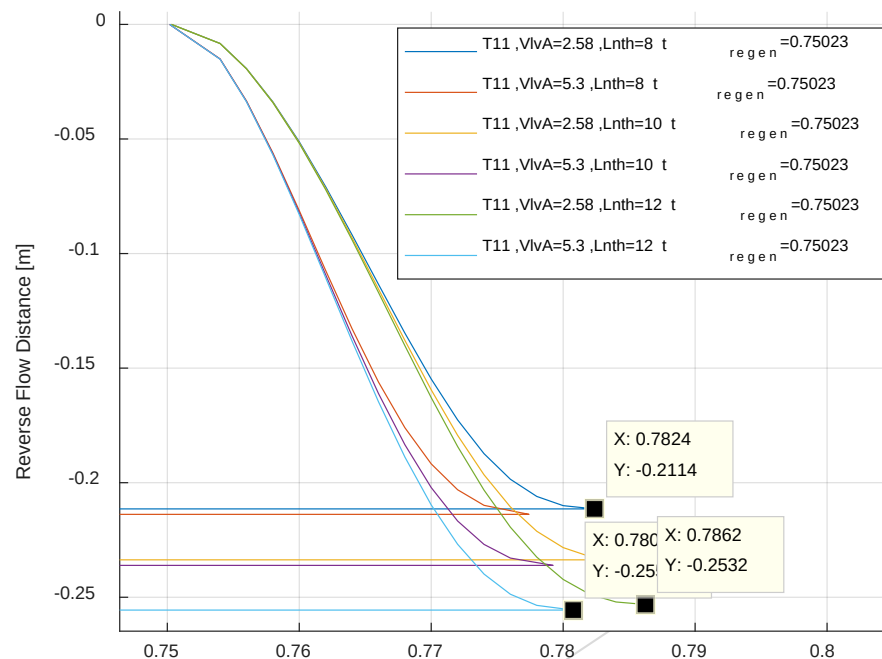
Pressure Profile



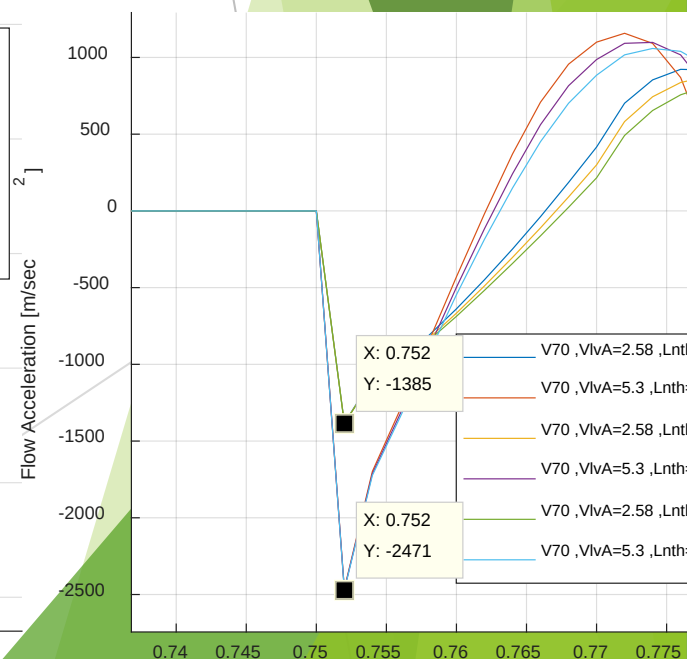
delta Pressure Profile



Cumulative Reverse Flow Distance



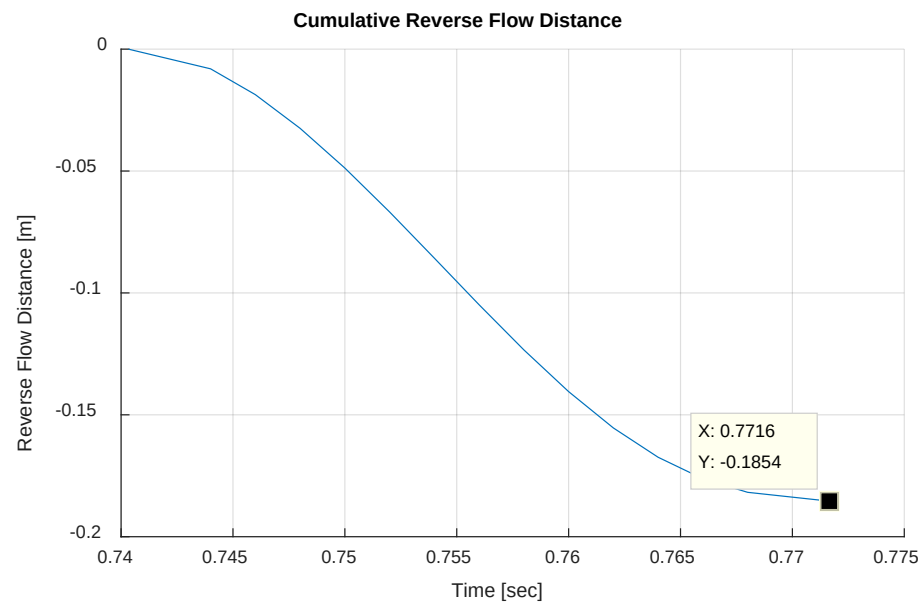
Flow Acceleration Profile



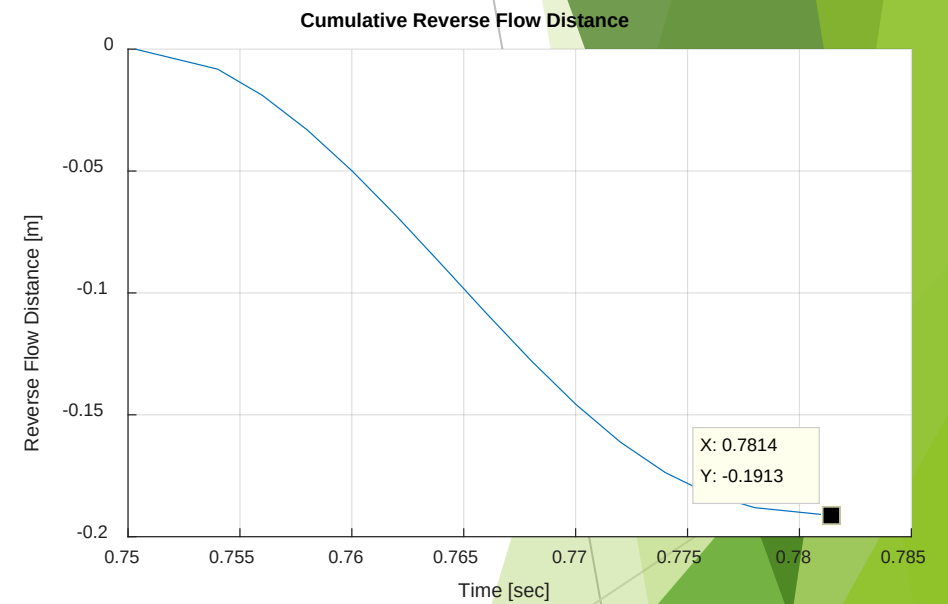
250kPa (35psi) Regen Pres

46 & 66mm Regeneration Valve Diameters

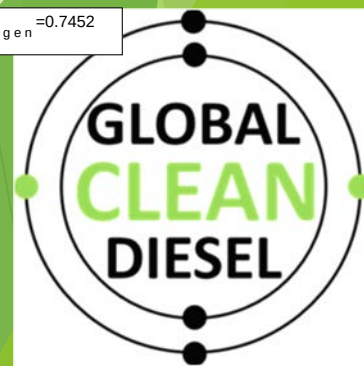
10.3L, 12.8, 15.4 (2.75, 3.4, & 4 gal) Holding Tank

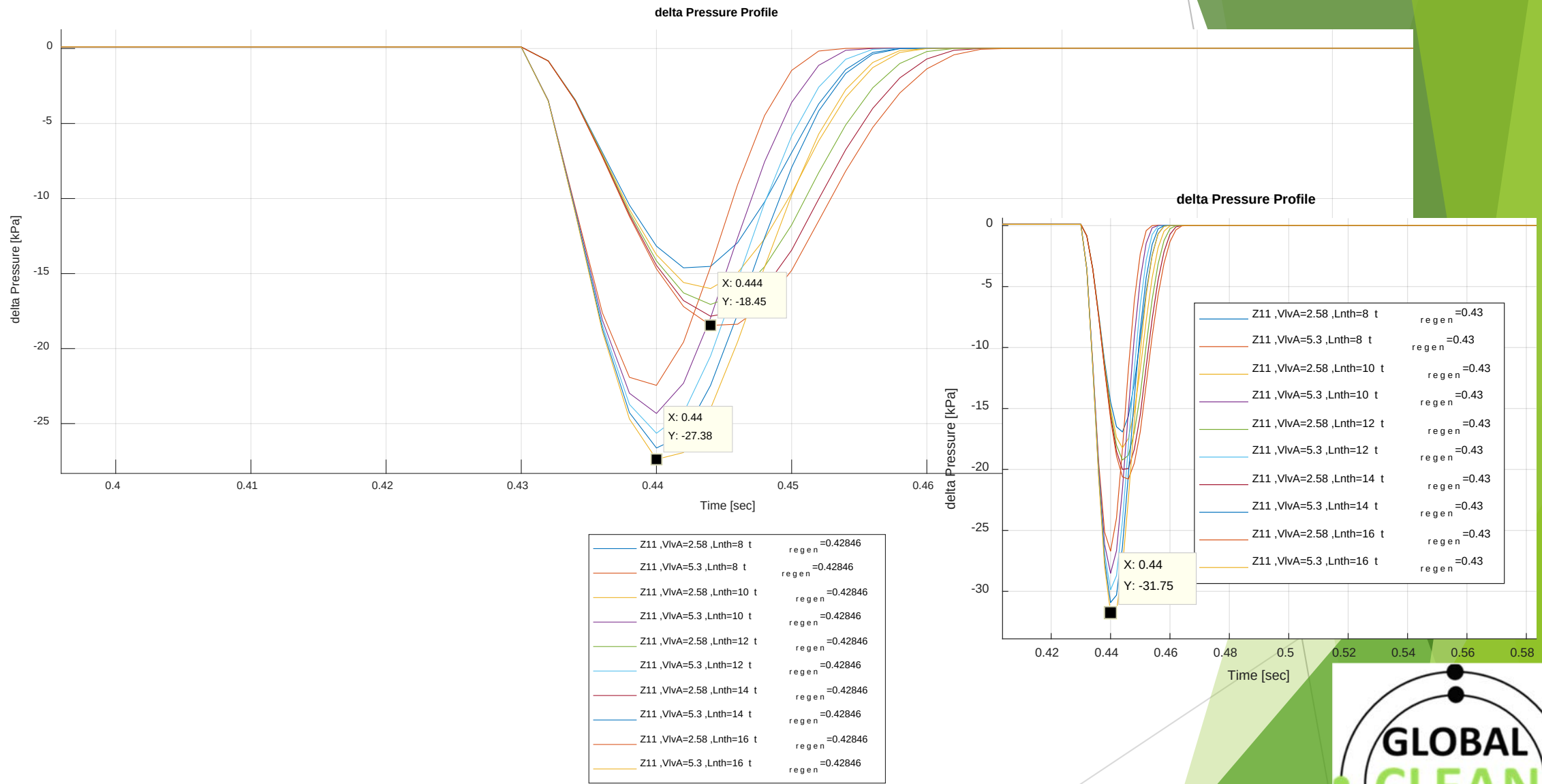


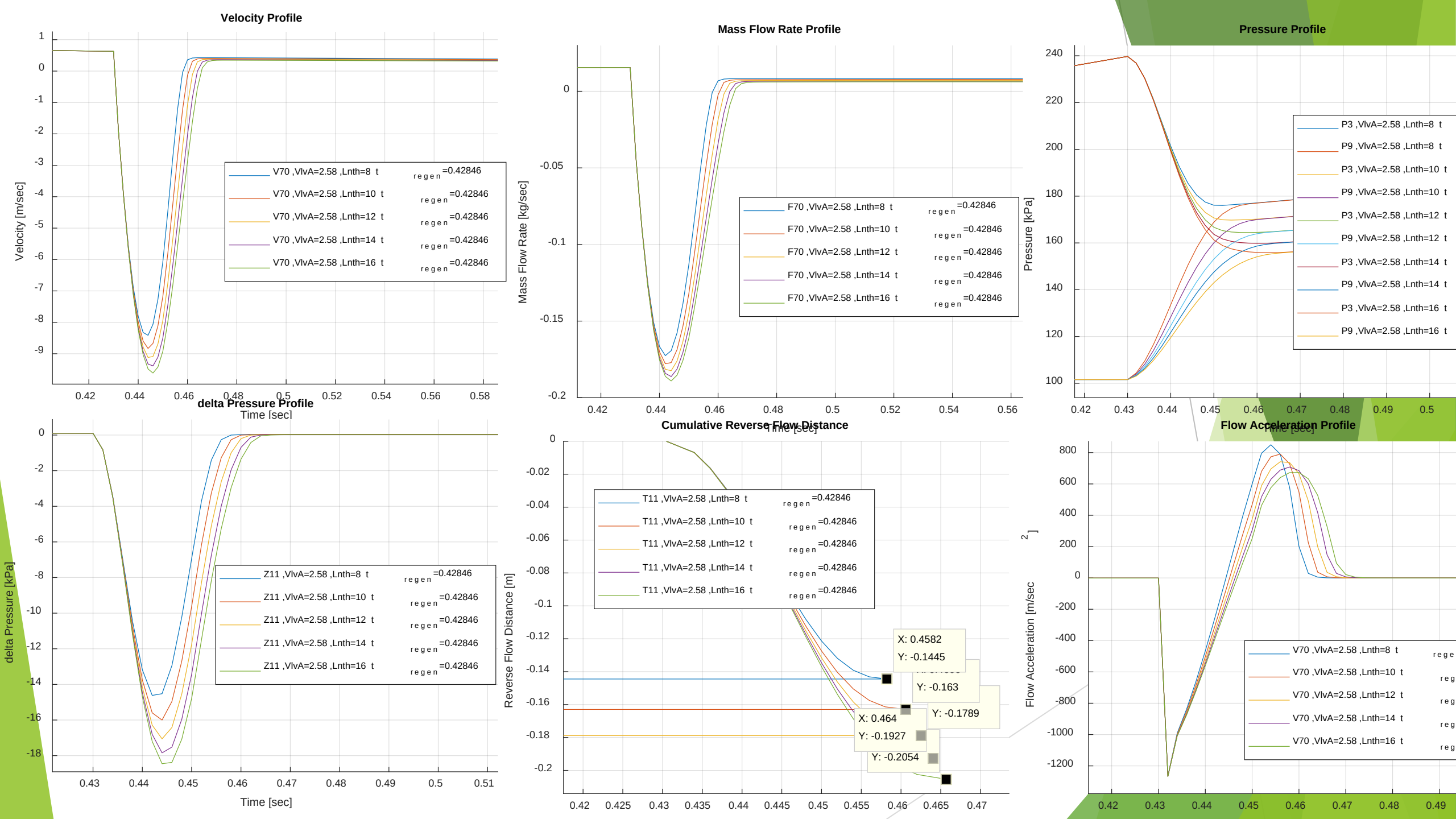
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T11 ,VlvA=2.58 ,Lnth=8 t_{regen}=0.7452



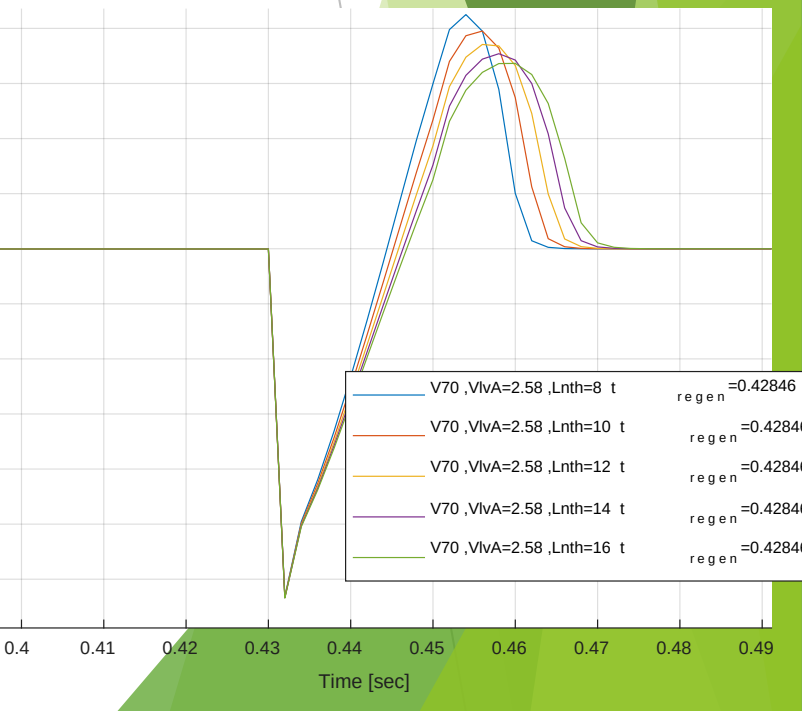
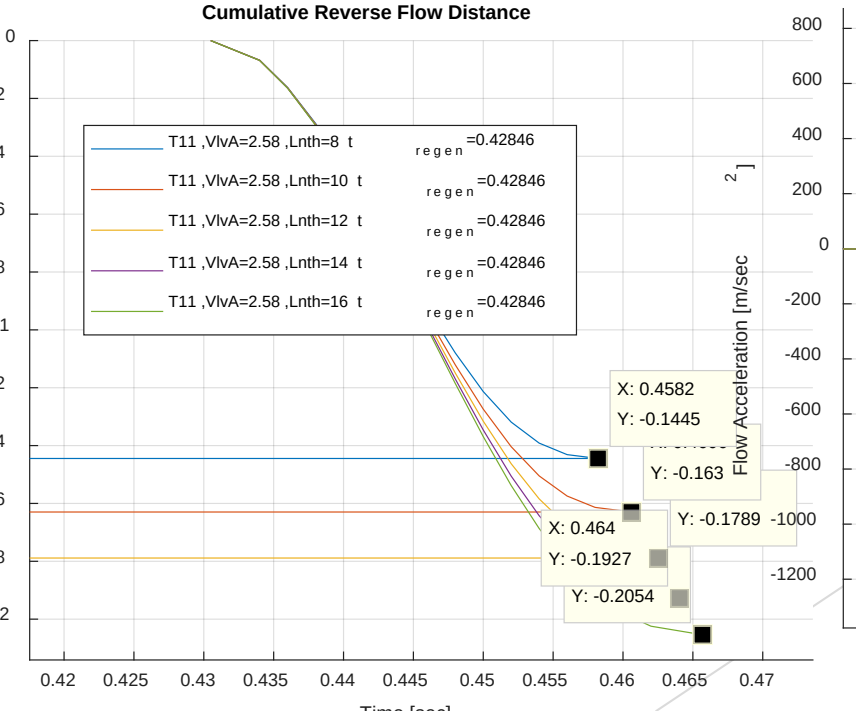
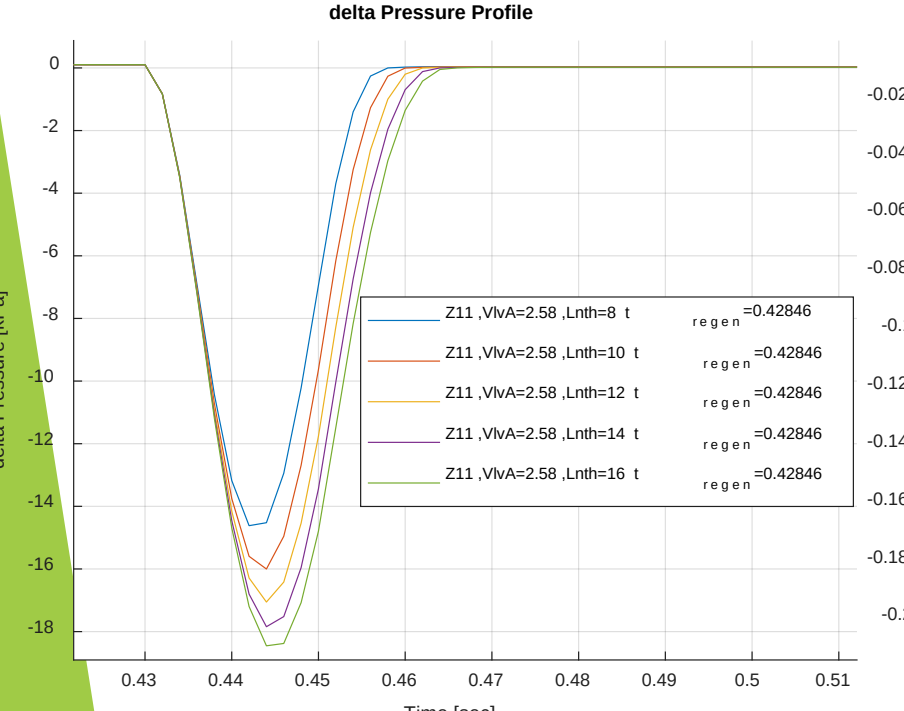
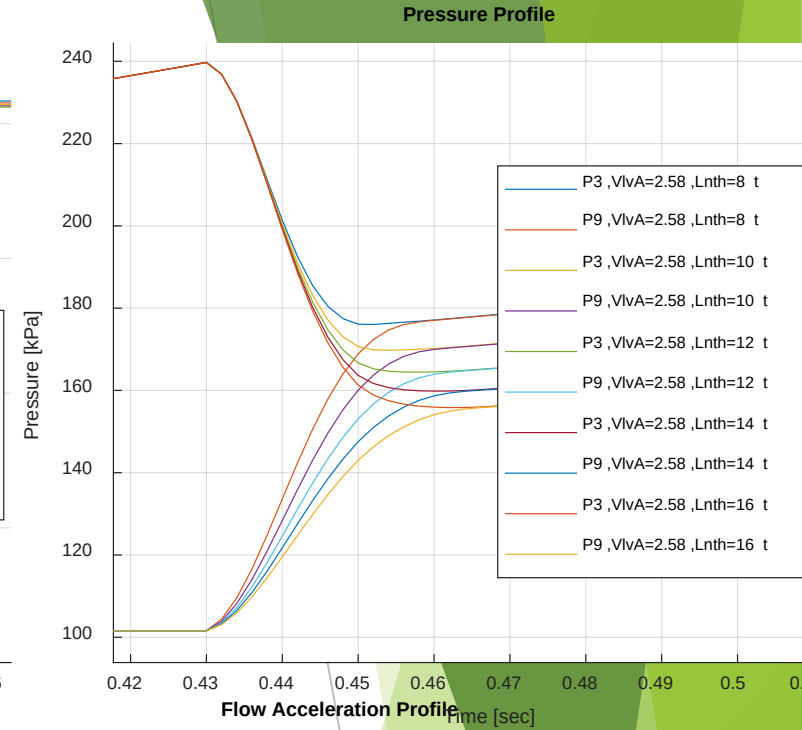
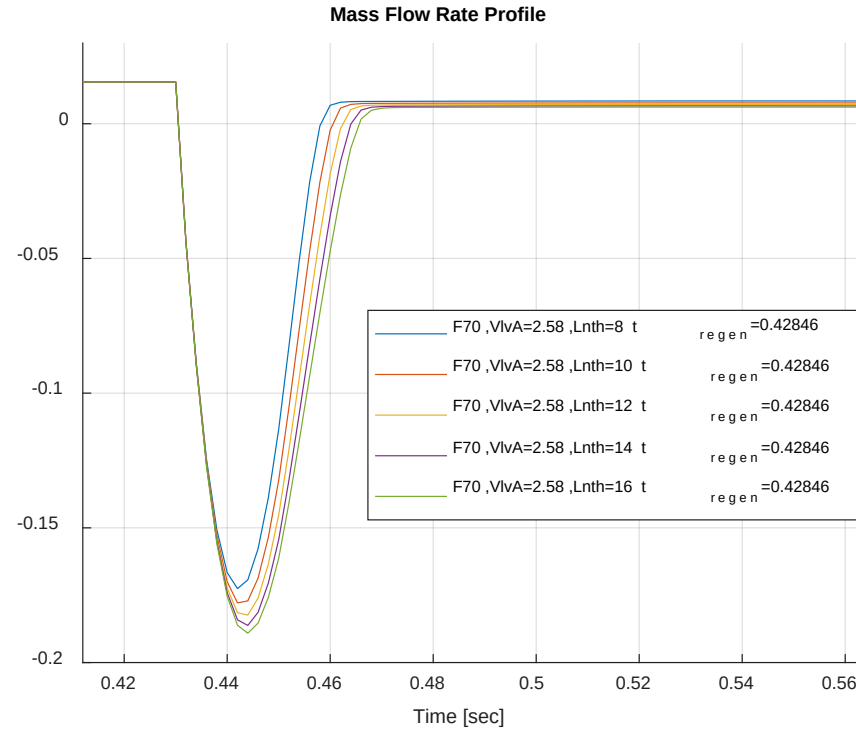
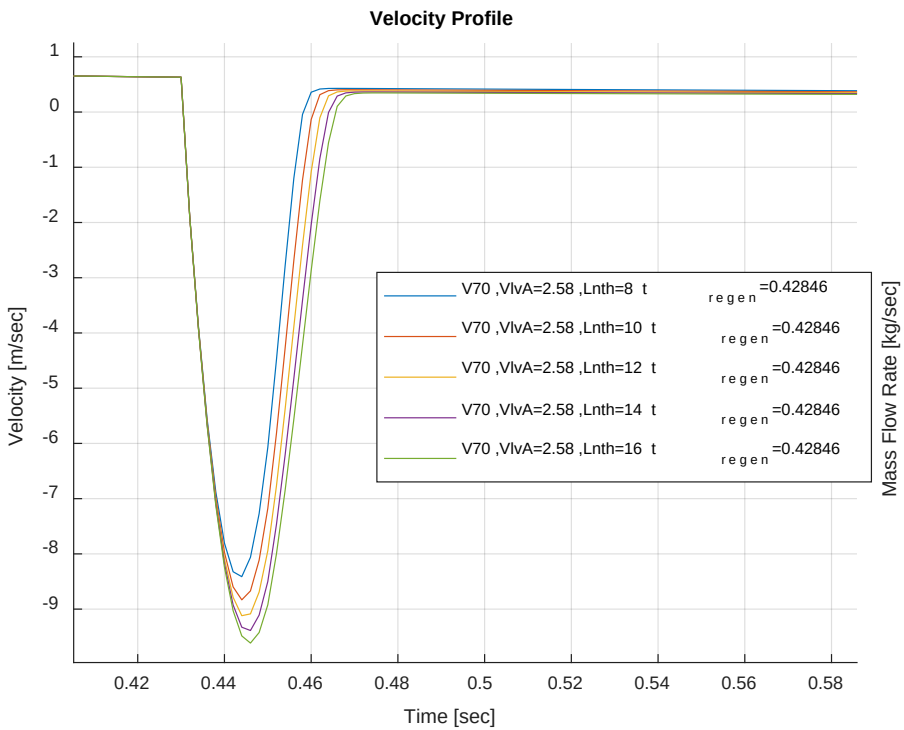




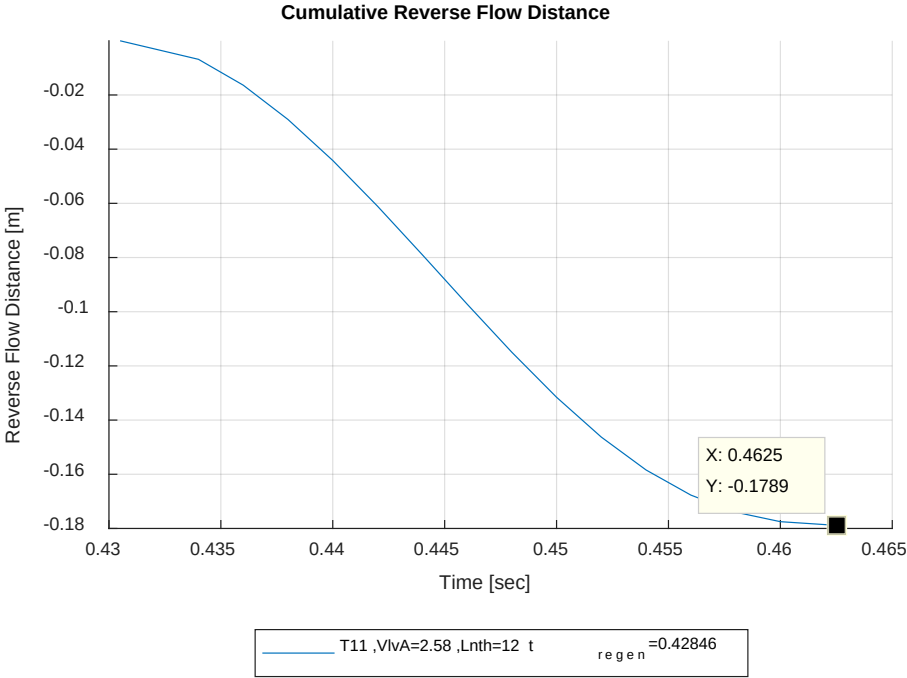
240kPa Regen Press

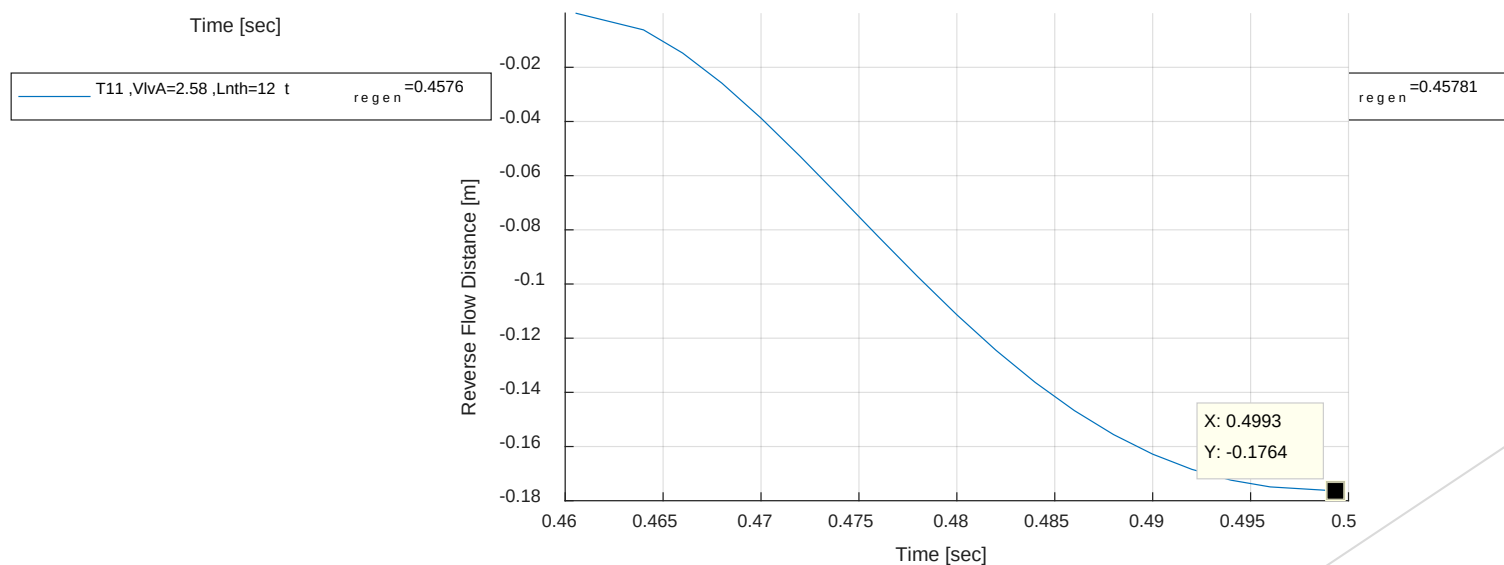
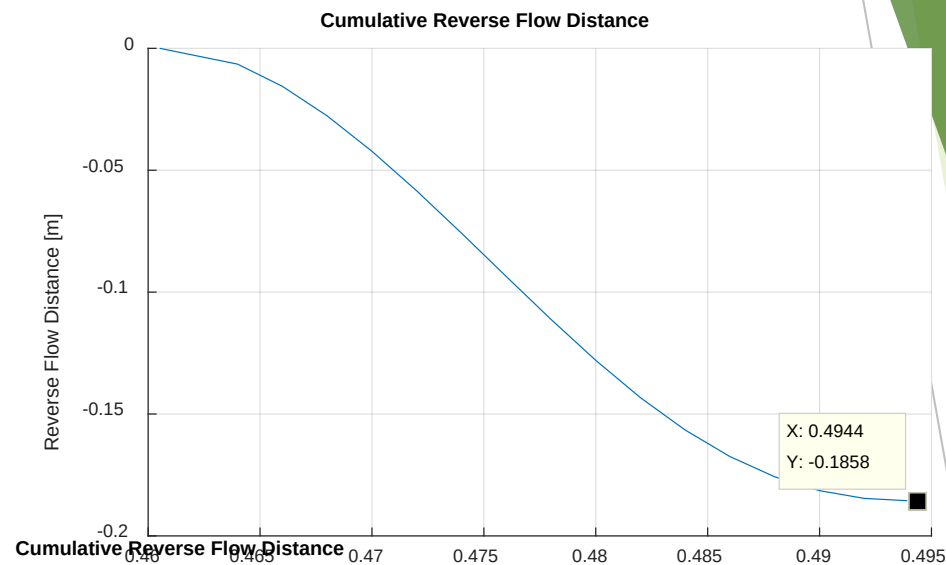
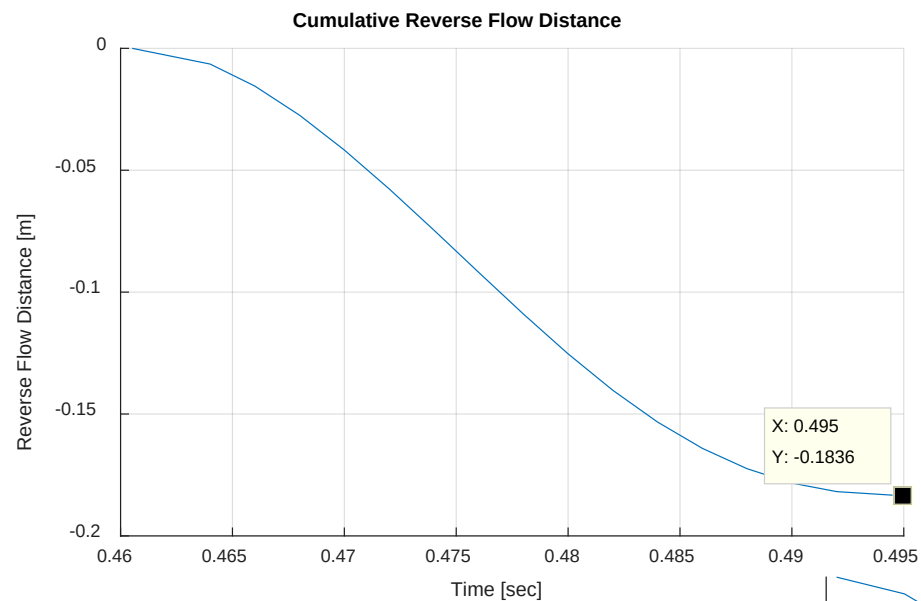
46mm Regen Valve Dia

Parametric Study of Settling tank Size

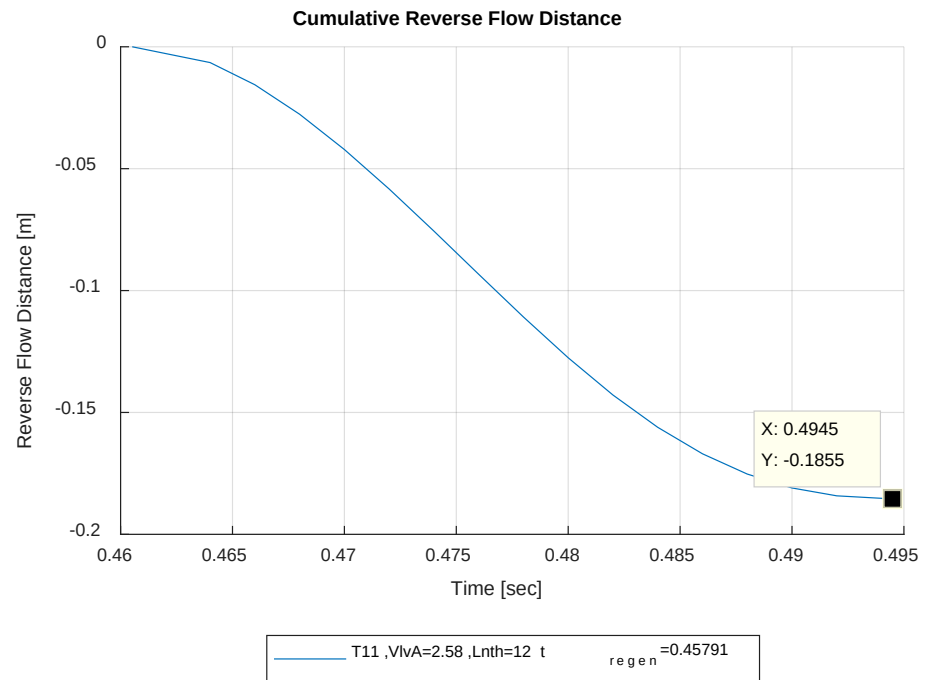
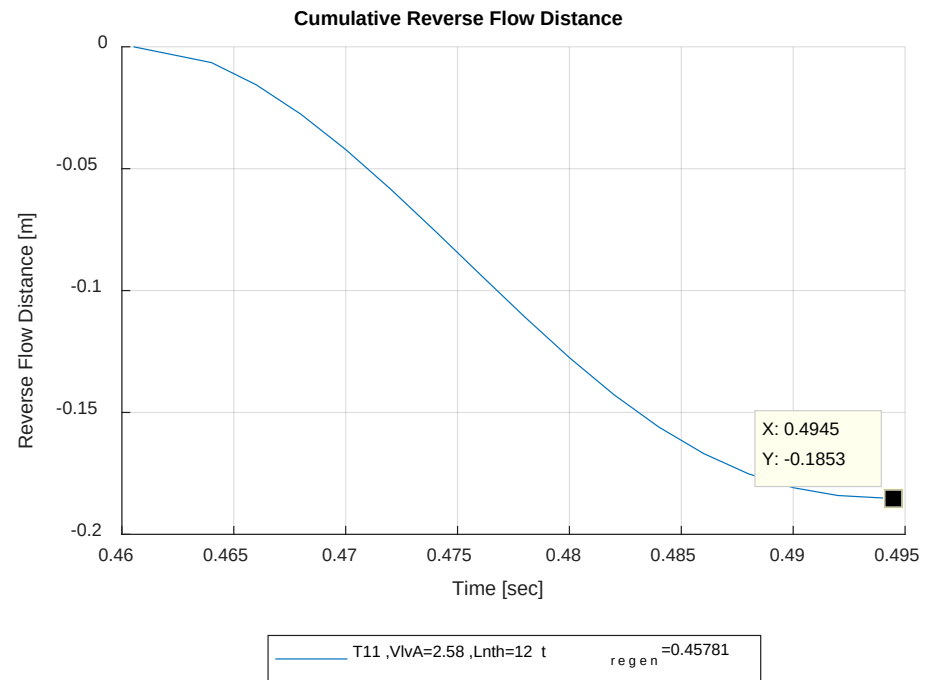


4 gal settling tank 35psi Regen Press





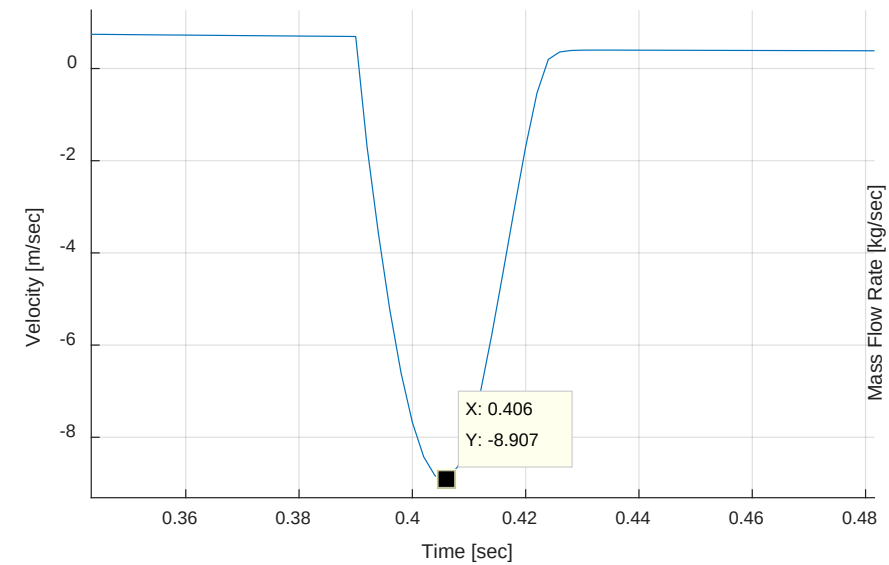
T11 ,VlvA=2.58 ,Lnth=12 t
re gen =0.45716



240kPa 10l Holding

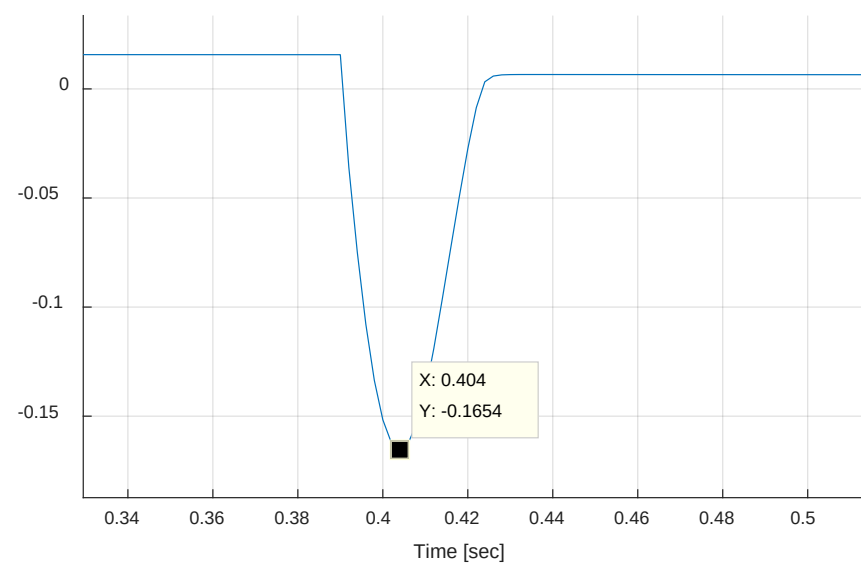
Base Holding Tank Size (12L)

Velocity Profile



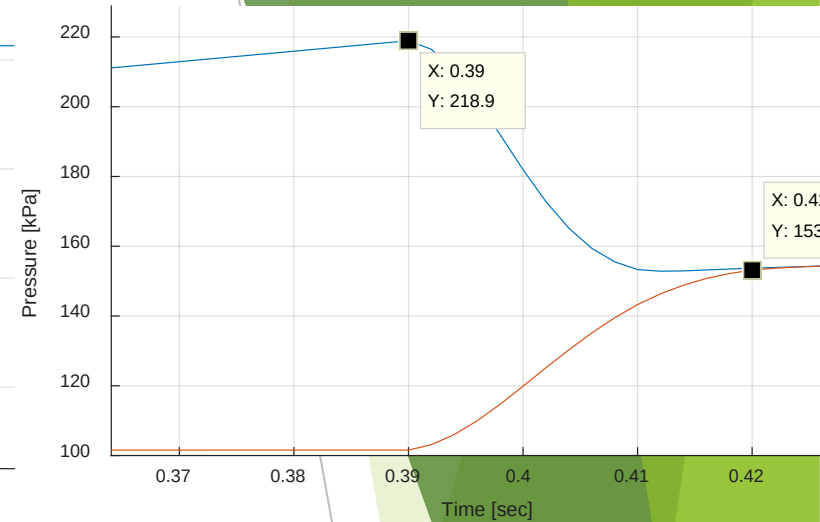
V70 ,VlvA=2.58 ,Lnth=12 t
=0.38654
re gen

Mass Flow Rate Profile



F70 ,VlvA=2.58 ,Lnth=12 t
=0.38654
re gen

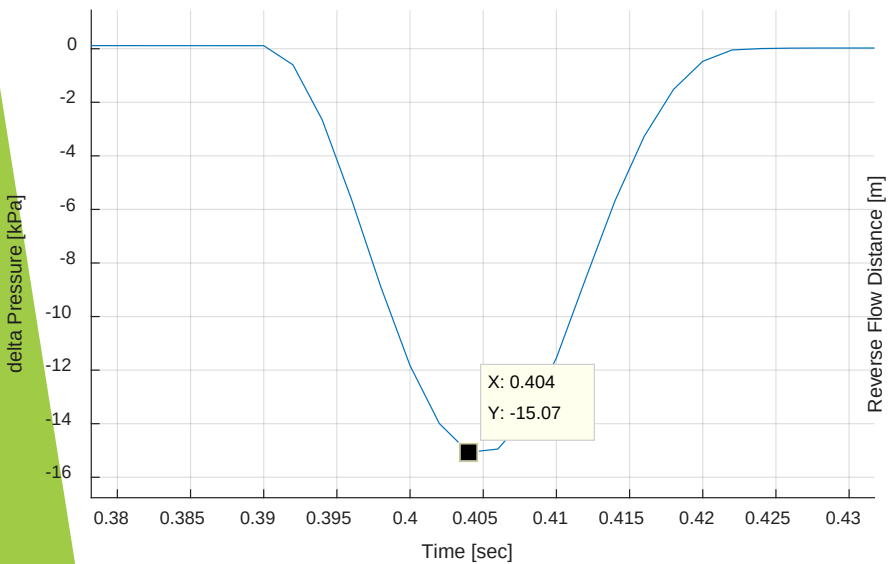
Pressure Profile



P3 ,VlvA=2.58 ,Lnth=12 t
=0.38654
re gen

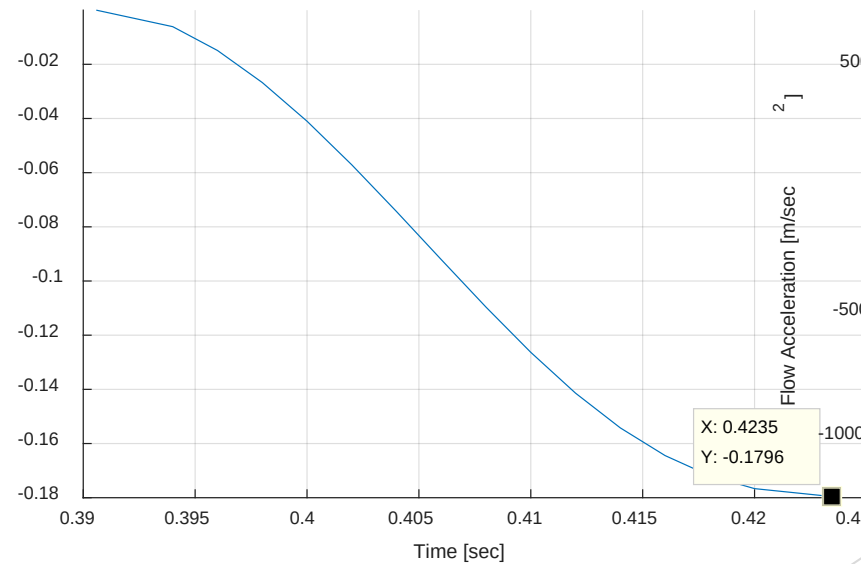
P9 ,VlvA=2.58 ,Lnth=12 t
=0.38654
re gen

delta Pressure Profile



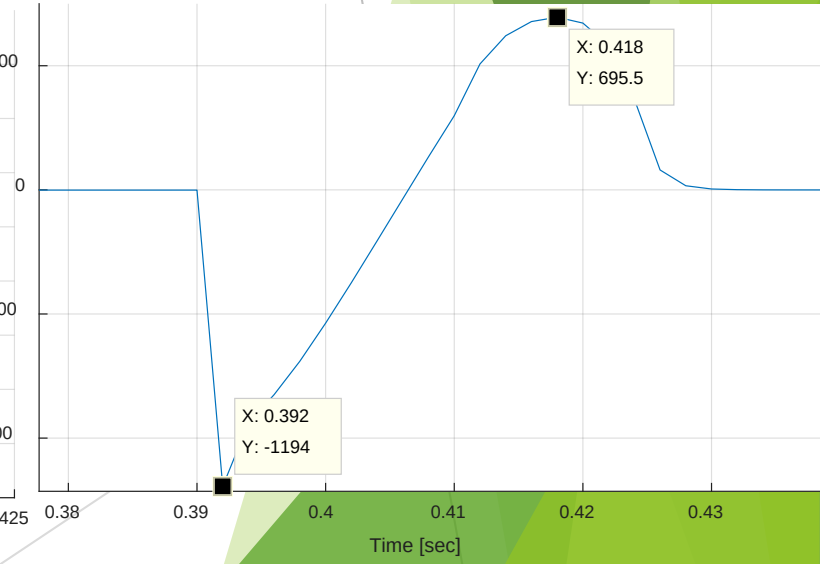
Z11 ,VlvA=2.58 ,Lnth=12 t
=0.38654
re gen

Cumulative Reverse Flow Distance



T11 ,VlvA=2.58 ,Lnth=12 t
=0.38654
re gen

Flow Acceleration Profile

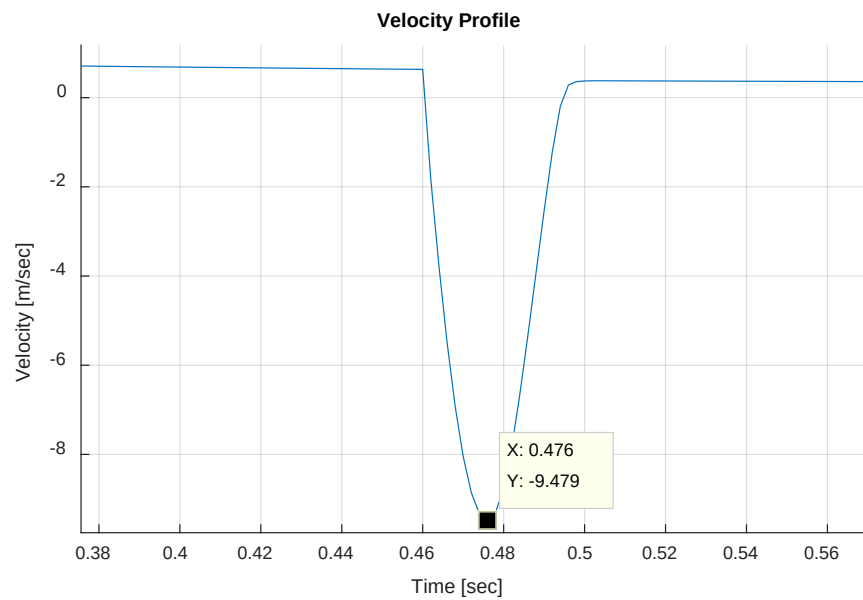


V70 ,VlvA=2.58 ,Lnth=12 t
=0.38654
re gen

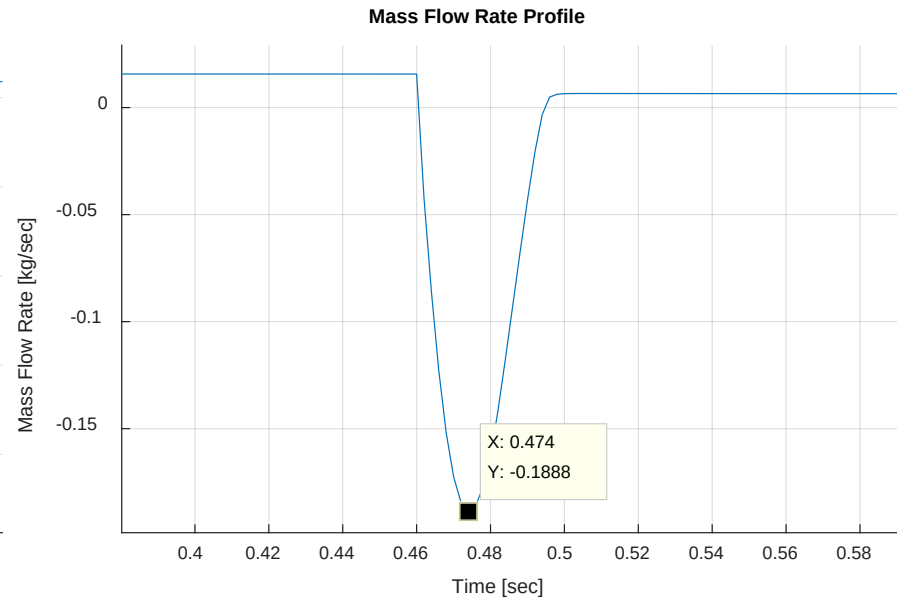
The background features abstract, overlapping green geometric shapes, primarily triangles and polygons, in various shades of green, creating a modern, layered effect on the right side of the slide.

240kPa 15l Holding

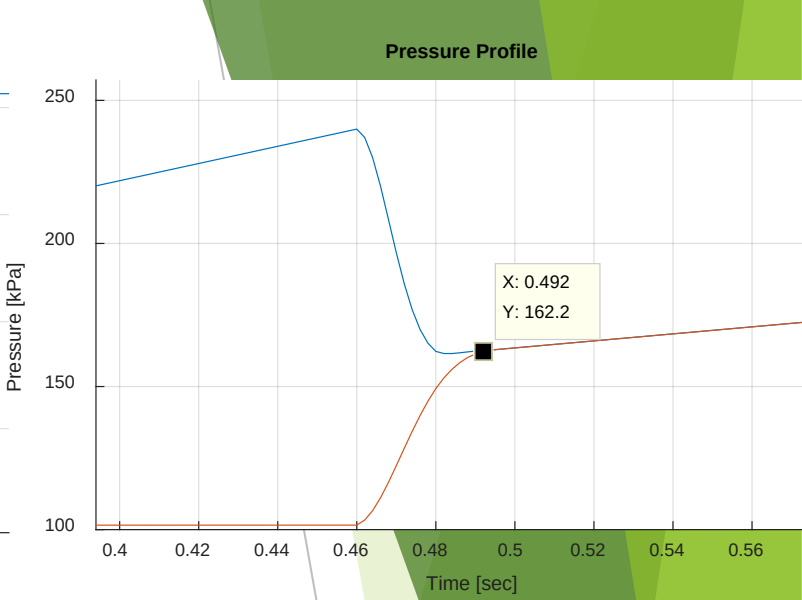
46mm Regeneration valve



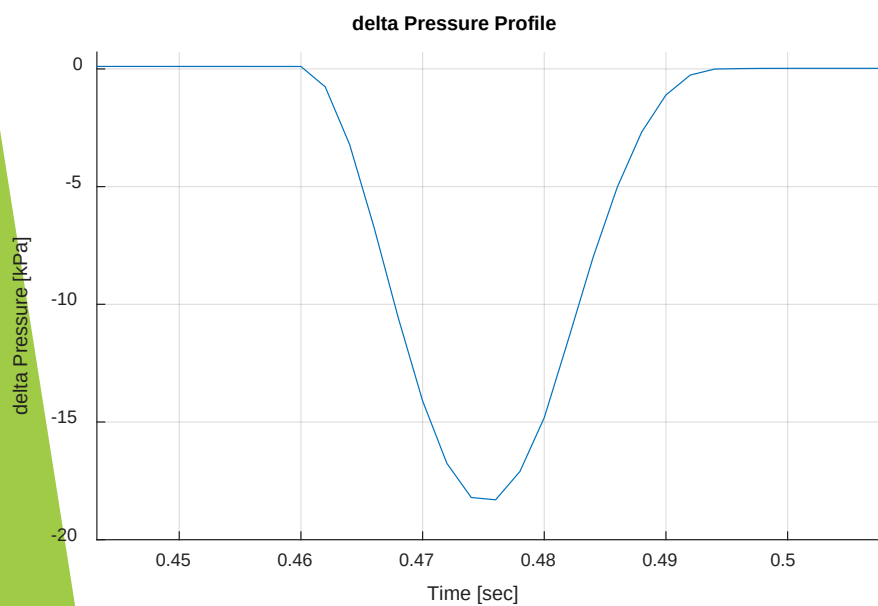
V70 ,VlvA=2.58 ,Lnth=12 t_{regen}=0.4591



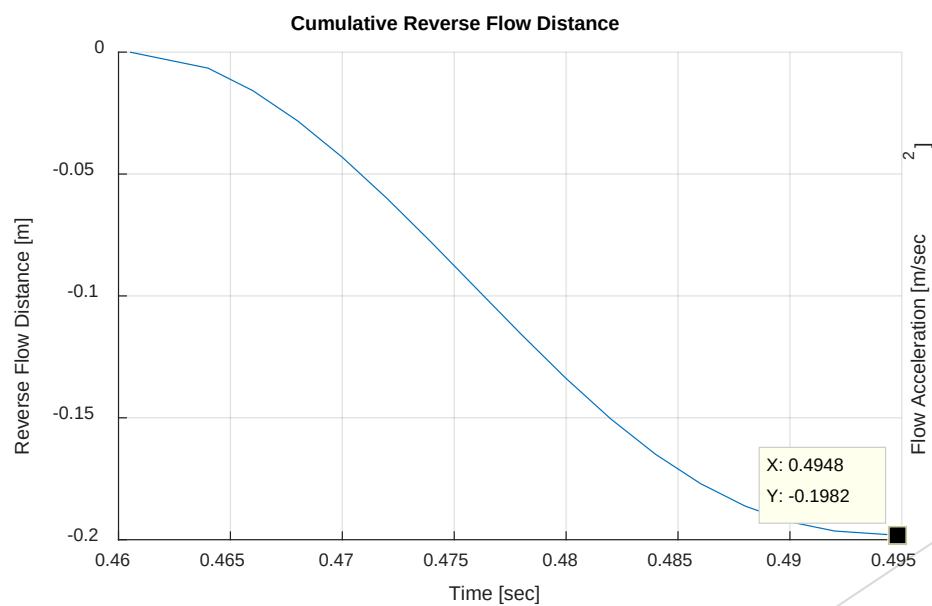
F70 ,VlvA=2.58 ,Lnth=12 t_{regen}=0.4591



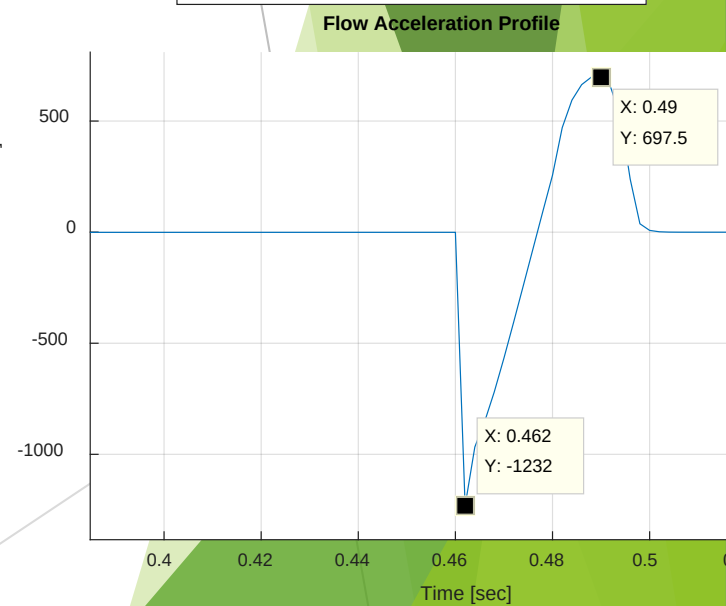
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P9 ,VlvA=2.58 ,Lnth=12 t_{regen}=0.4591



Z11 ,VlvA=2.58 ,Lnth=12 t_{regen}=0.4591



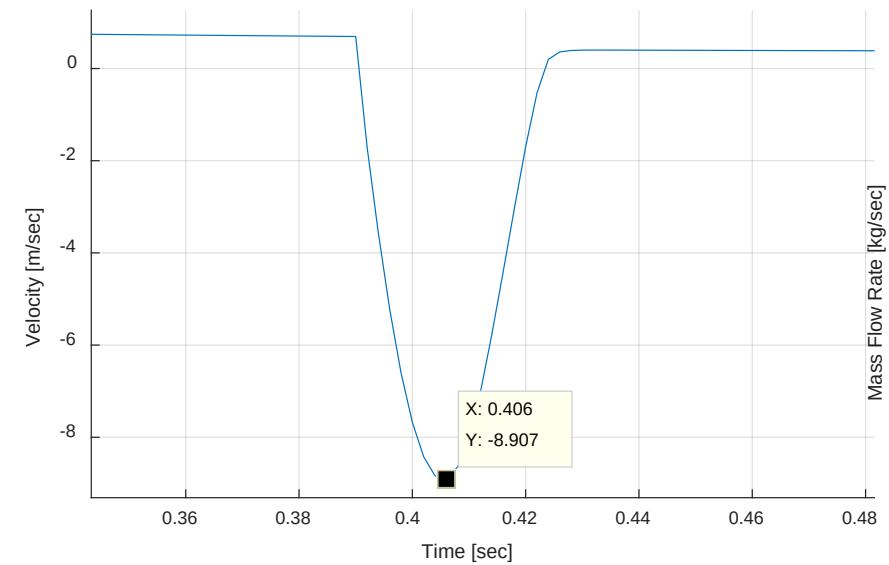
T11 ,VlvA=2.58 ,Lnth=12 t_{regen}=0.4591



V70 ,VlvA=2.58 ,Lnth=12 t_{regen}=0.4591

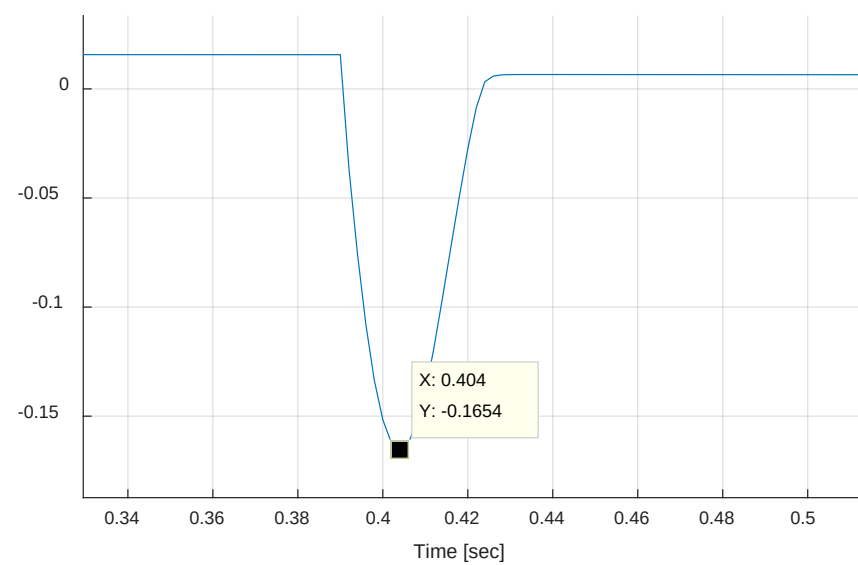
220kPa 15l Holding

Velocity Profile



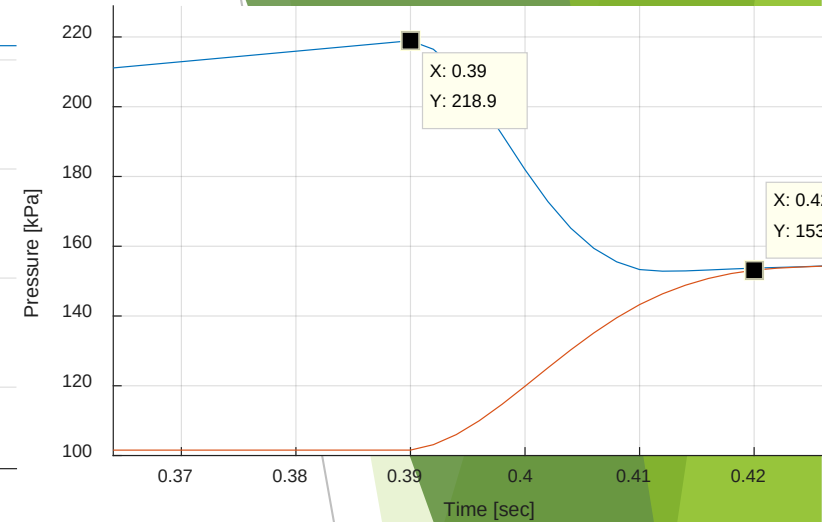
V70 ,VlvA=2.58 ,Lnth=12 t
=0.38654
re gen

Mass Flow Rate Profile



F70 ,VlvA=2.58 ,Lnth=12 t
=0.38654
re gen

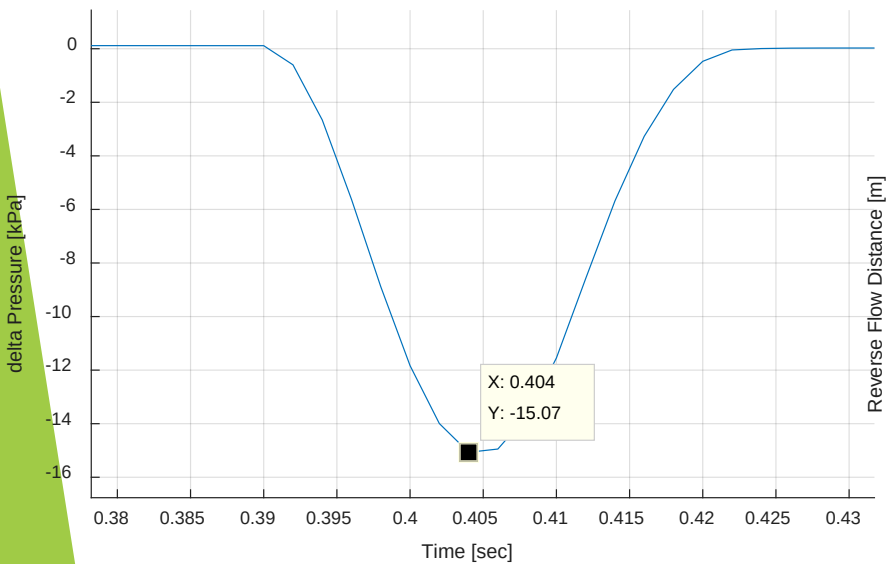
Pressure Profile



P3 ,VlvA=2.58 ,Lnth=12 t
=0.38654
re gen

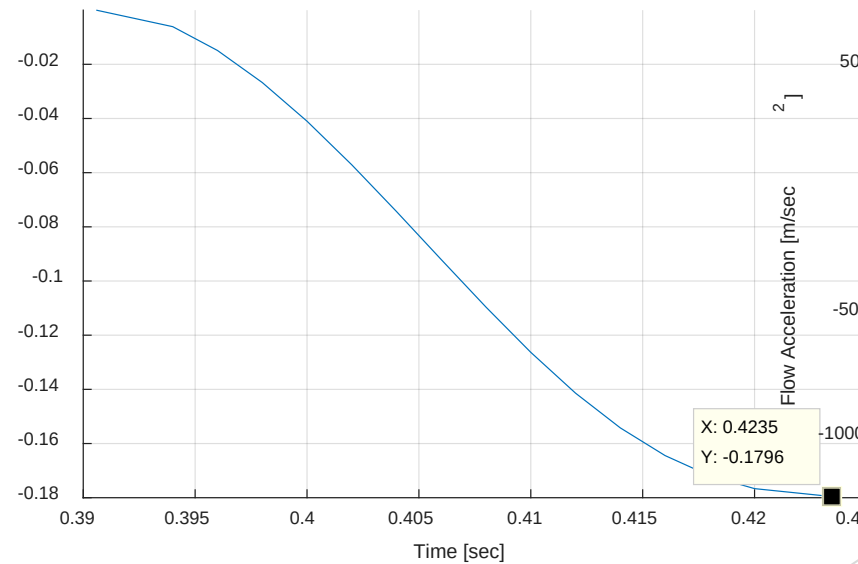
P9 ,VlvA=2.58 ,Lnth=12 t
=0.38654
re gen

delta Pressure Profile



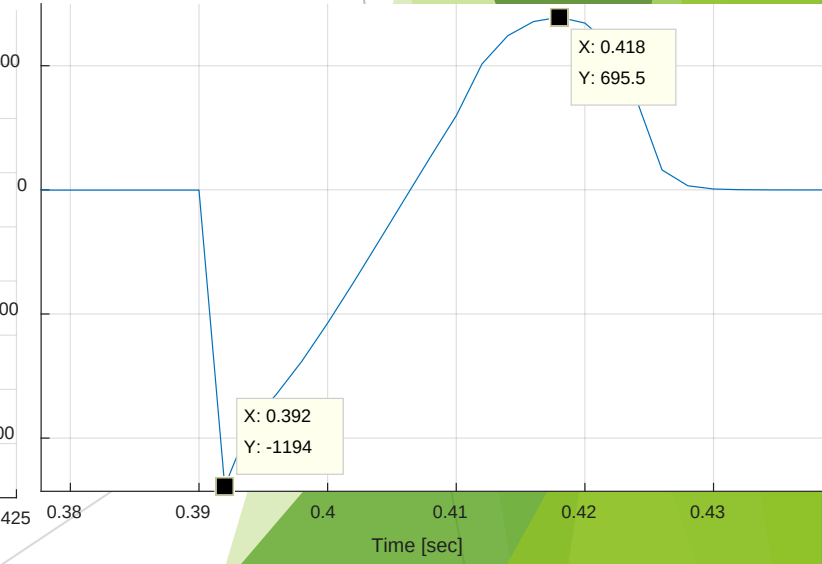
Z11 ,VlvA=2.58 ,Lnth=12 t
=0.38654
re gen

Cumulative Reverse Flow Distance



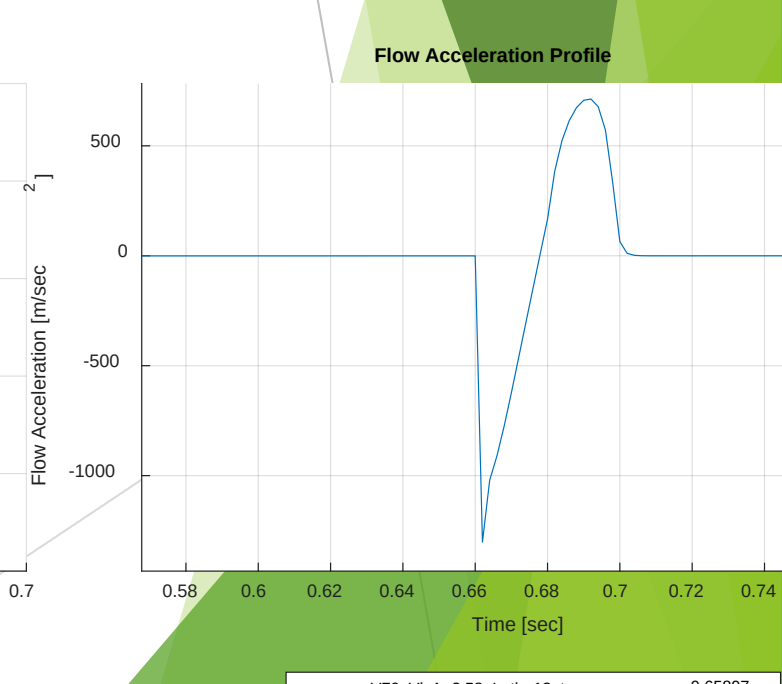
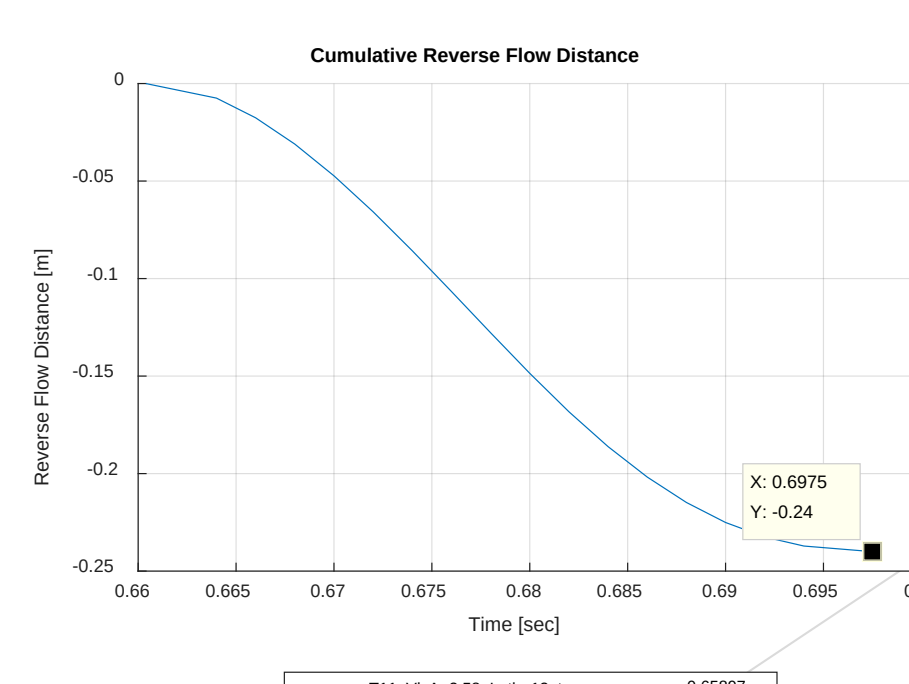
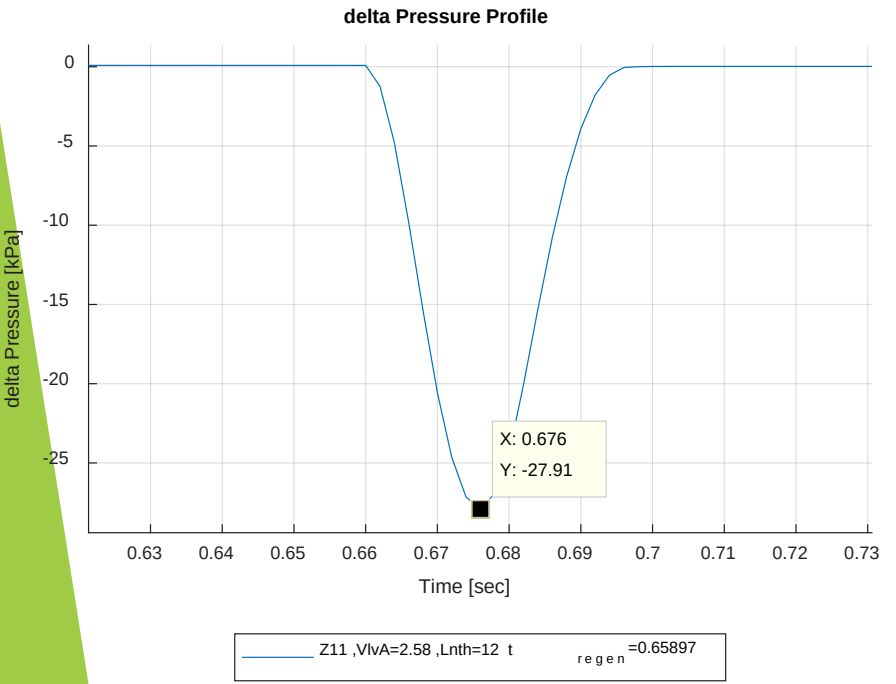
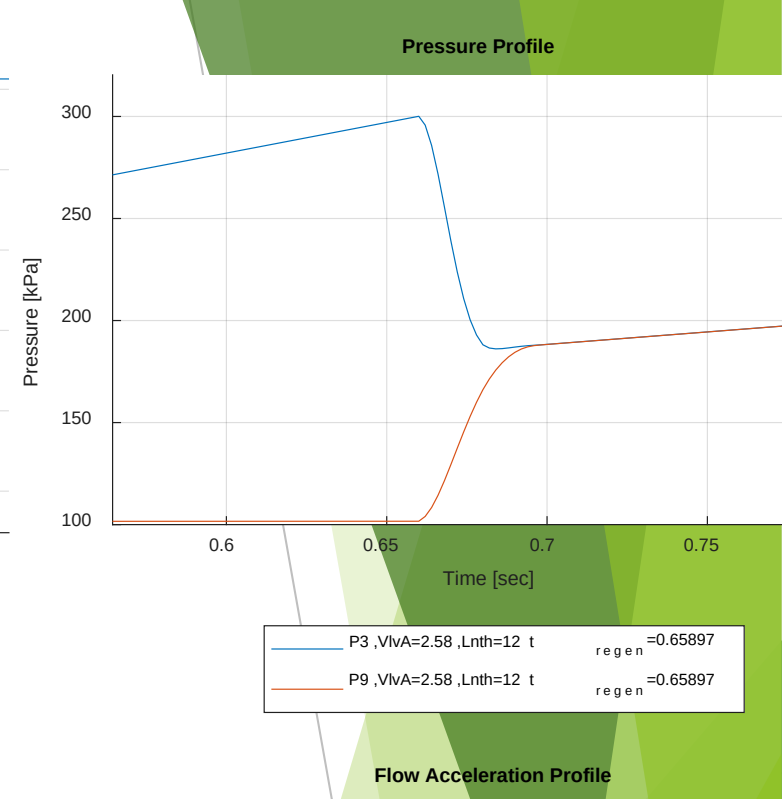
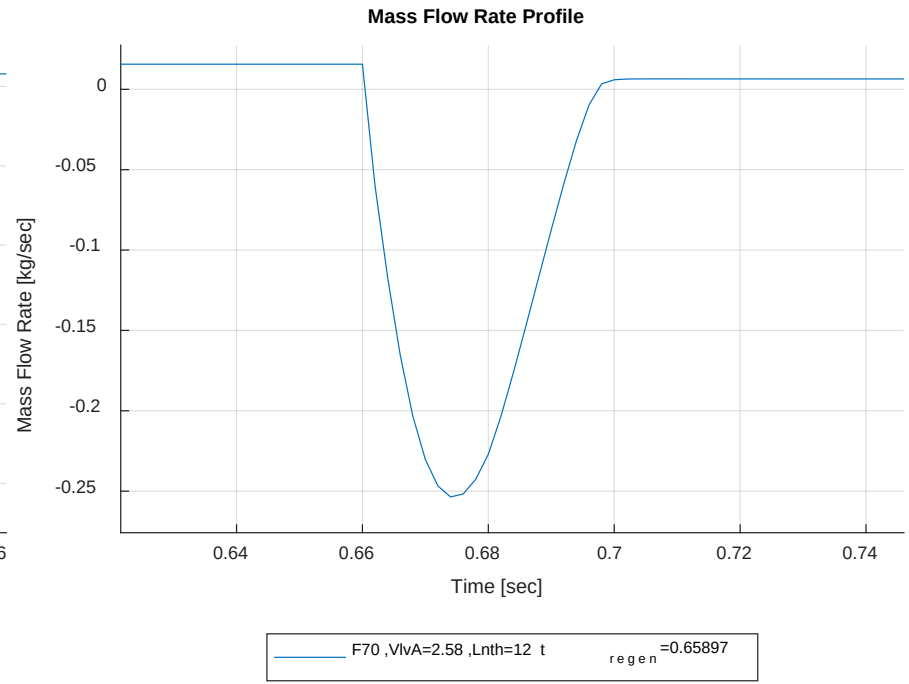
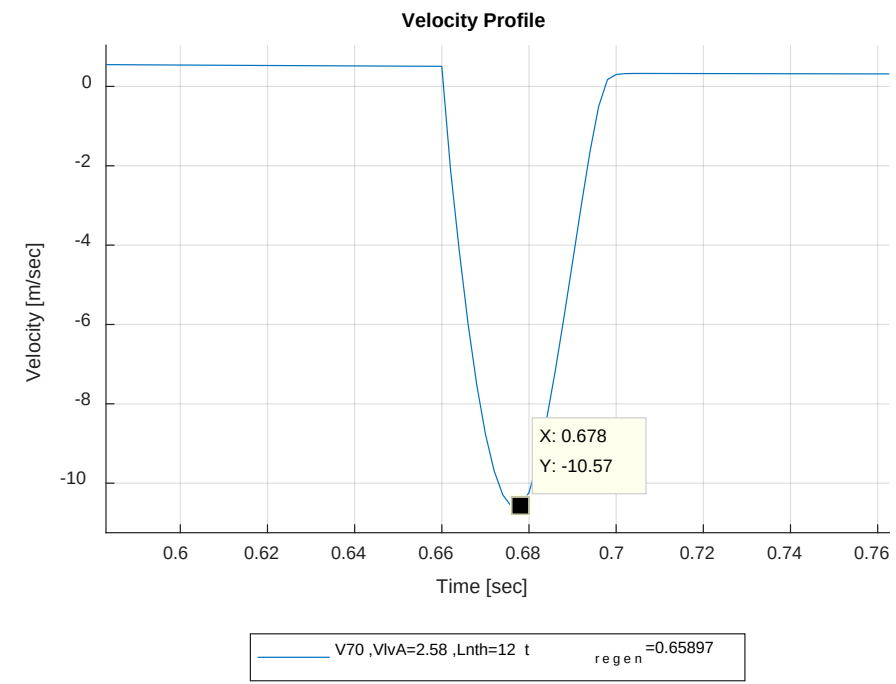
T11 ,VlvA=2.58 ,Lnth=12 t
=0.38654
re gen

Flow Acceleration Profile

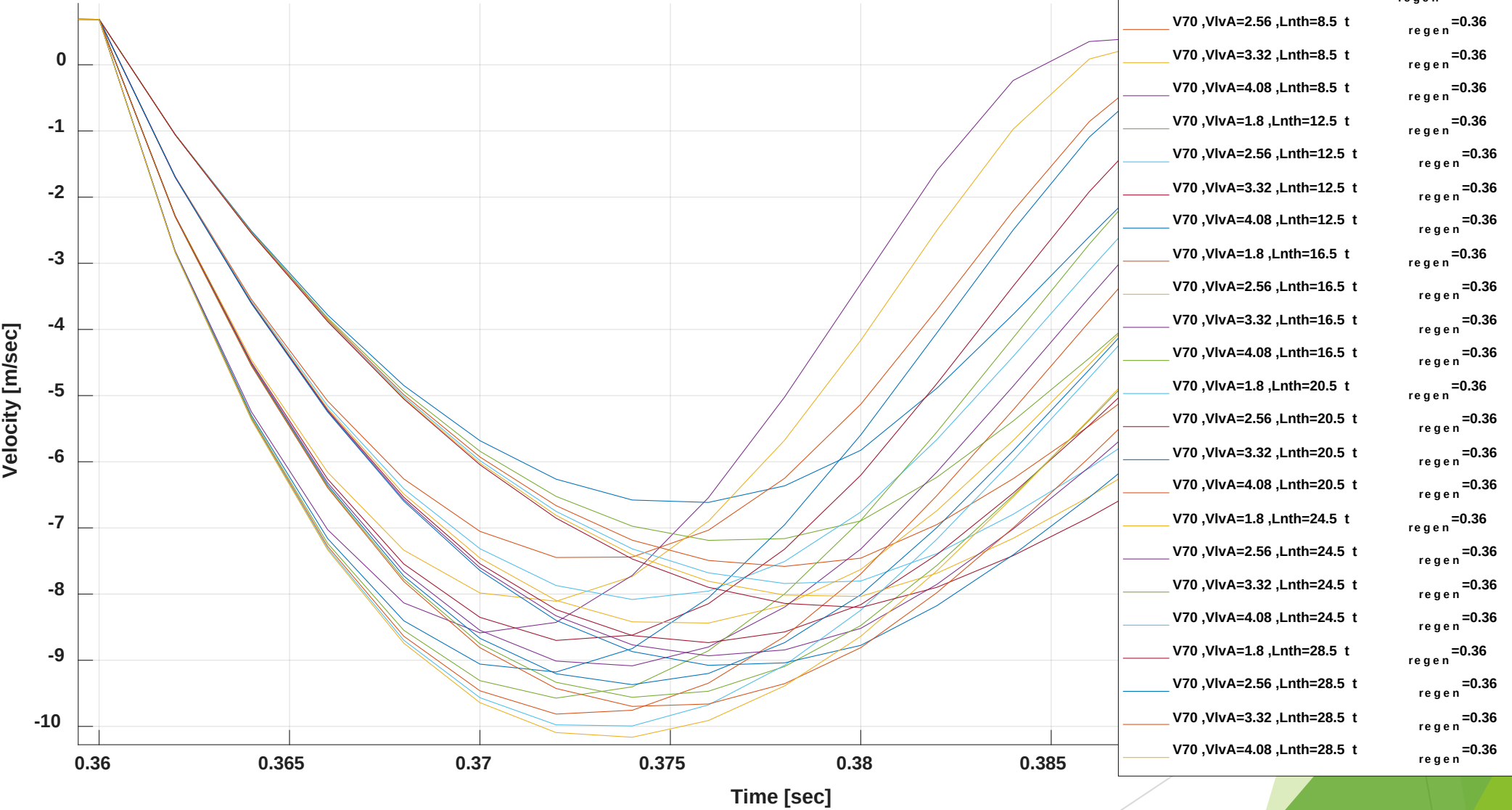


V70 ,VlvA=2.58 ,Lnth=12 t
=0.38654
re gen

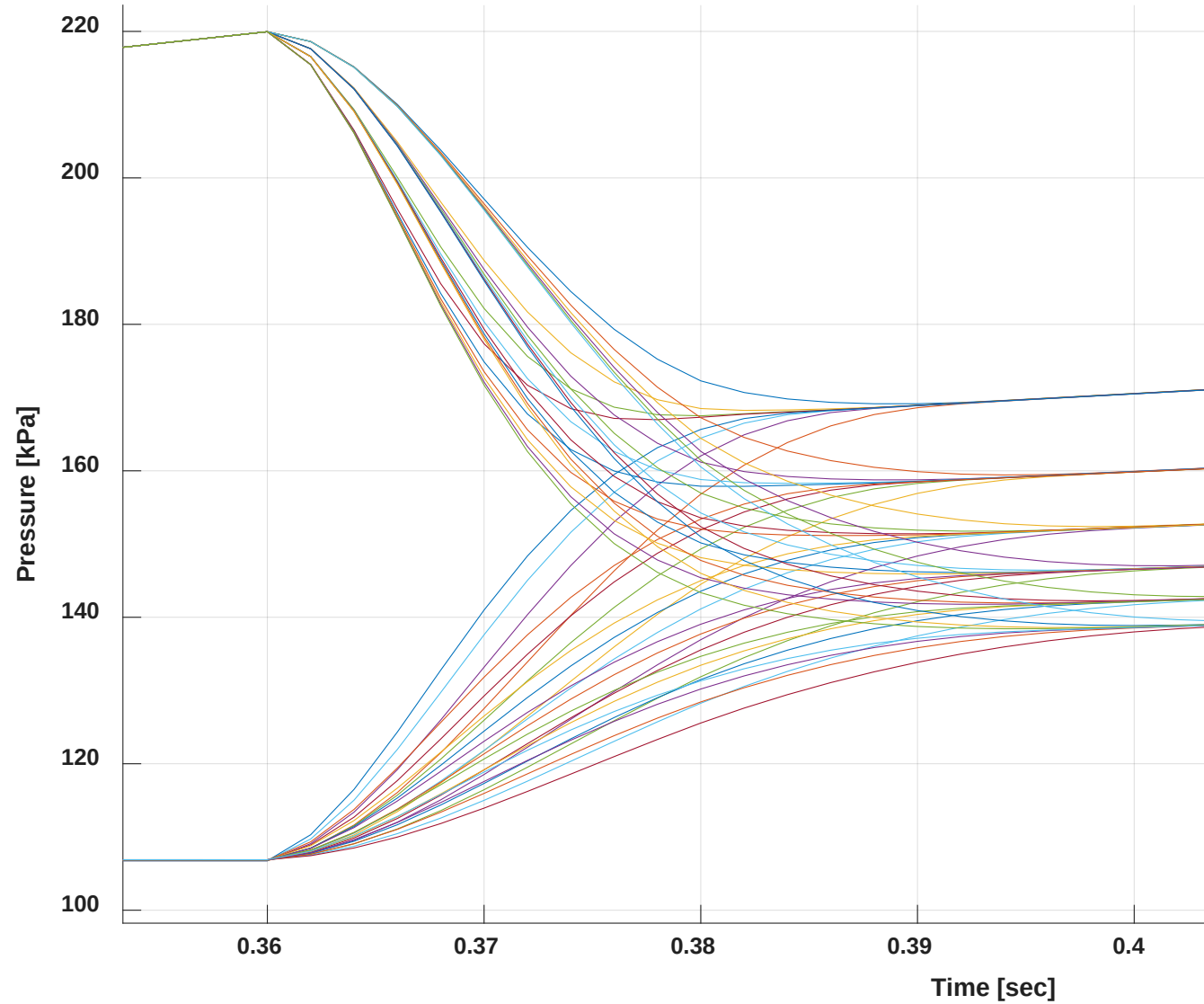
300kPa 15l Holding



Velocity Profile

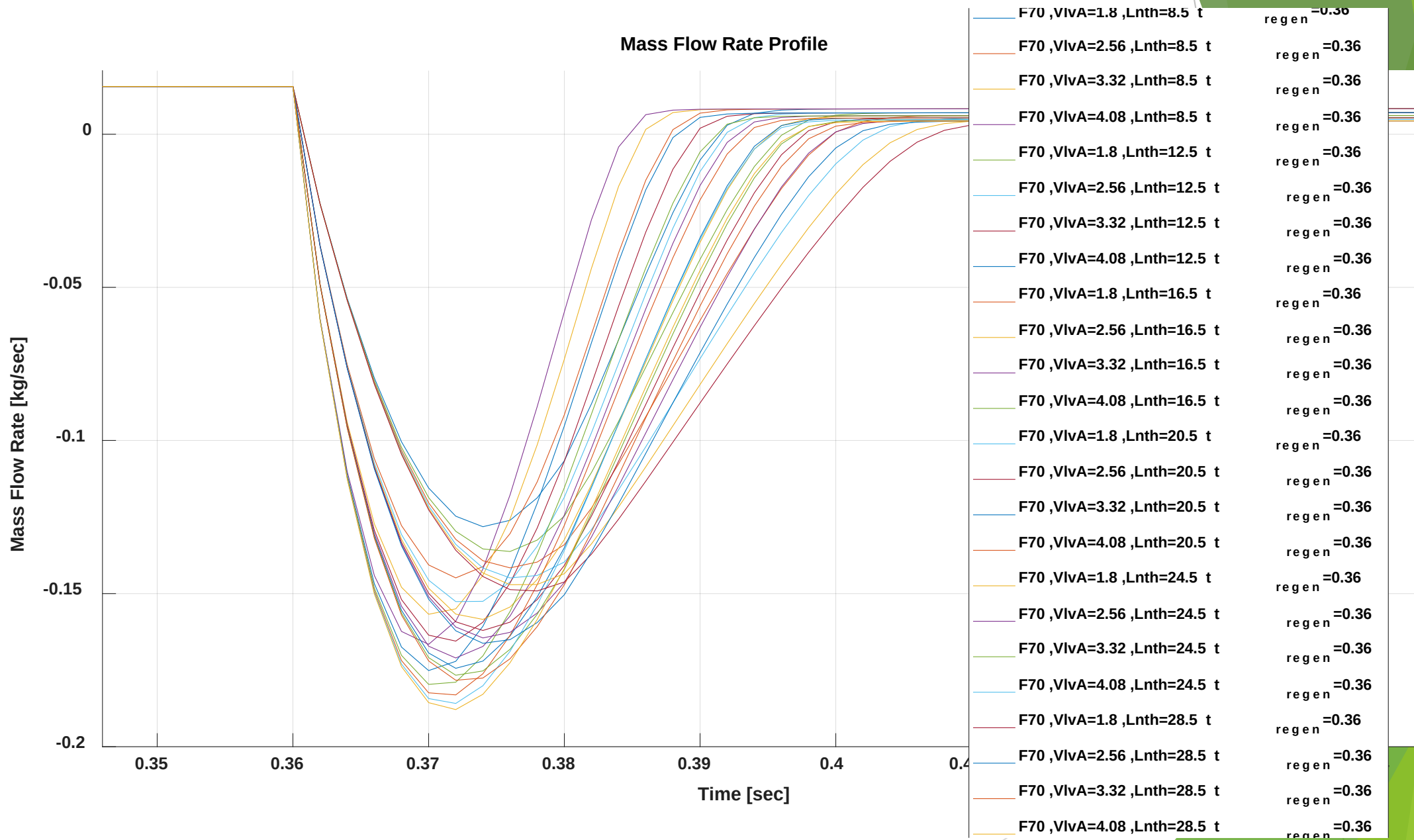


Pressure Profile

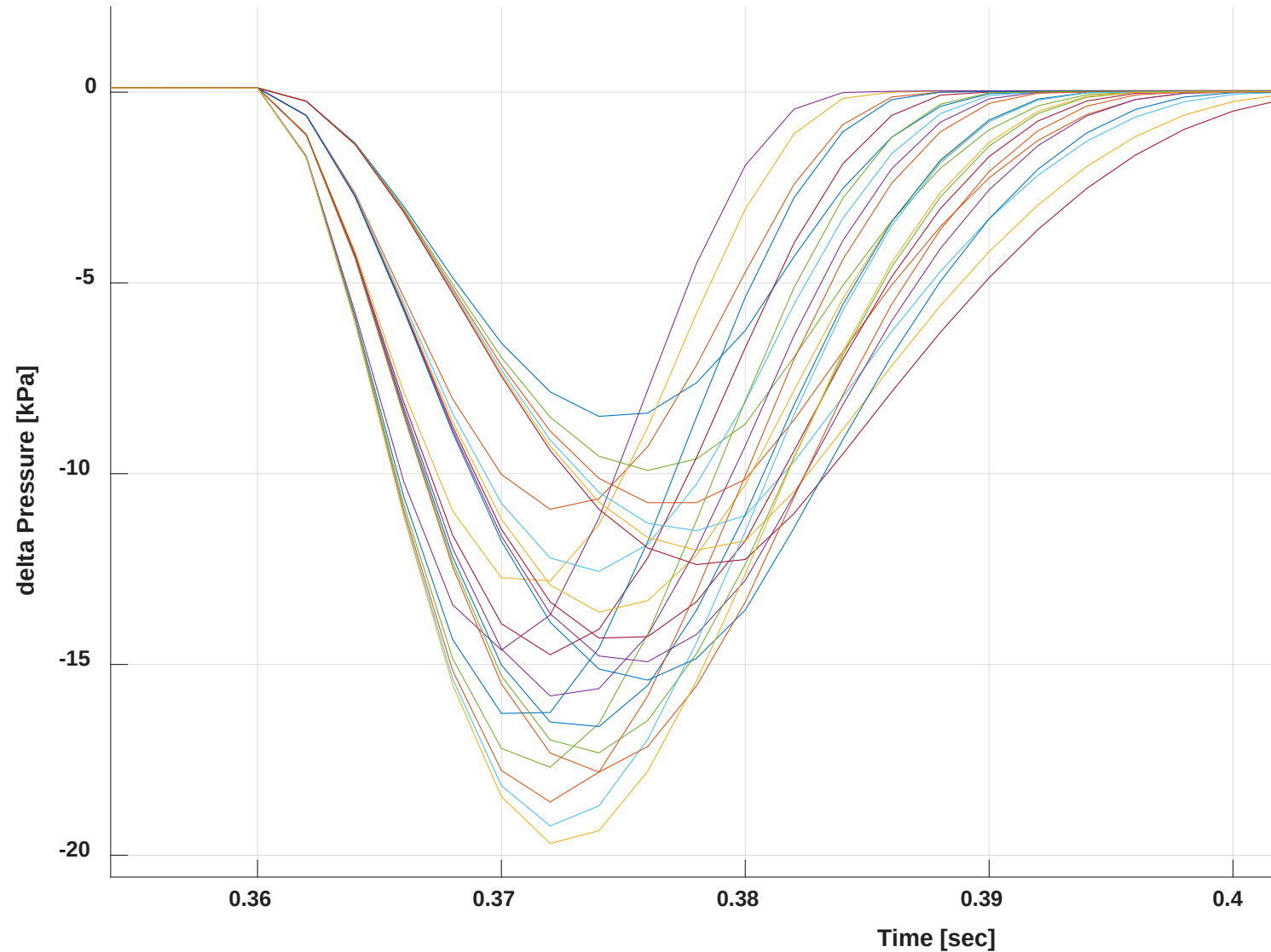


P100 ,VlvA=1.8 ,Lnth=16.5 t	re gen =0.36
P9 ,VlvA=1.8 ,Lnth=16.5 t	re gen =0.36
P100 ,VlvA=2.56 ,Lnth=16.5 t	re gen =0.36
P9 ,VlvA=2.56 ,Lnth=16.5 t	re gen =0.36
P100 ,VlvA=3.32 ,Lnth=16.5 t	re gen =0.36
P9 ,VlvA=3.32 ,Lnth=16.5 t	re gen =0.36
P100 ,VlvA=4.08 ,Lnth=16.5 t	re gen =0.36
P9 ,VlvA=4.08 ,Lnth=16.5 t	re gen =0.36
P100 ,VlvA=1.8 ,Lnth=20.5 t	re gen =0.36
P9 ,VlvA=1.8 ,Lnth=20.5 t	re gen =0.36
P100 ,VlvA=2.56 ,Lnth=20.5 t	re gen =0.36
P9 ,VlvA=2.56 ,Lnth=20.5 t	re gen =0.36
P100 ,VlvA=3.32 ,Lnth=20.5 t	re gen =0.36
P9 ,VlvA=3.32 ,Lnth=20.5 t	re gen =0.36
P100 ,VlvA=4.08 ,Lnth=20.5 t	re gen =0.36
P9 ,VlvA=4.08 ,Lnth=20.5 t	re gen =0.36
P100 ,VlvA=1.8 ,Lnth=24.5 t	re gen =0.36
P9 ,VlvA=1.8 ,Lnth=24.5 t	re gen =0.36
P100 ,VlvA=2.56 ,Lnth=24.5 t	re gen =0.36
P9 ,VlvA=2.56 ,Lnth=24.5 t	re gen =0.36
P100 ,VlvA=3.32 ,Lnth=24.5 t	re gen =0.36
P9 ,VlvA=3.32 ,Lnth=24.5 t	re gen =0.36
P100 ,VlvA=4.08 ,Lnth=24.5 t	re gen =0.36
P9 ,VlvA=4.08 ,Lnth=24.5 t	re gen =0.36
P100 ,VlvA=1.8 ,Lnth=28.5 t	re gen =0.36
P9 ,VlvA=1.8 ,Lnth=28.5 t	re gen =0.36
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P9 ,VlvA=2.56 ,Lnth=28.5 t	re gen =0.36

0.43

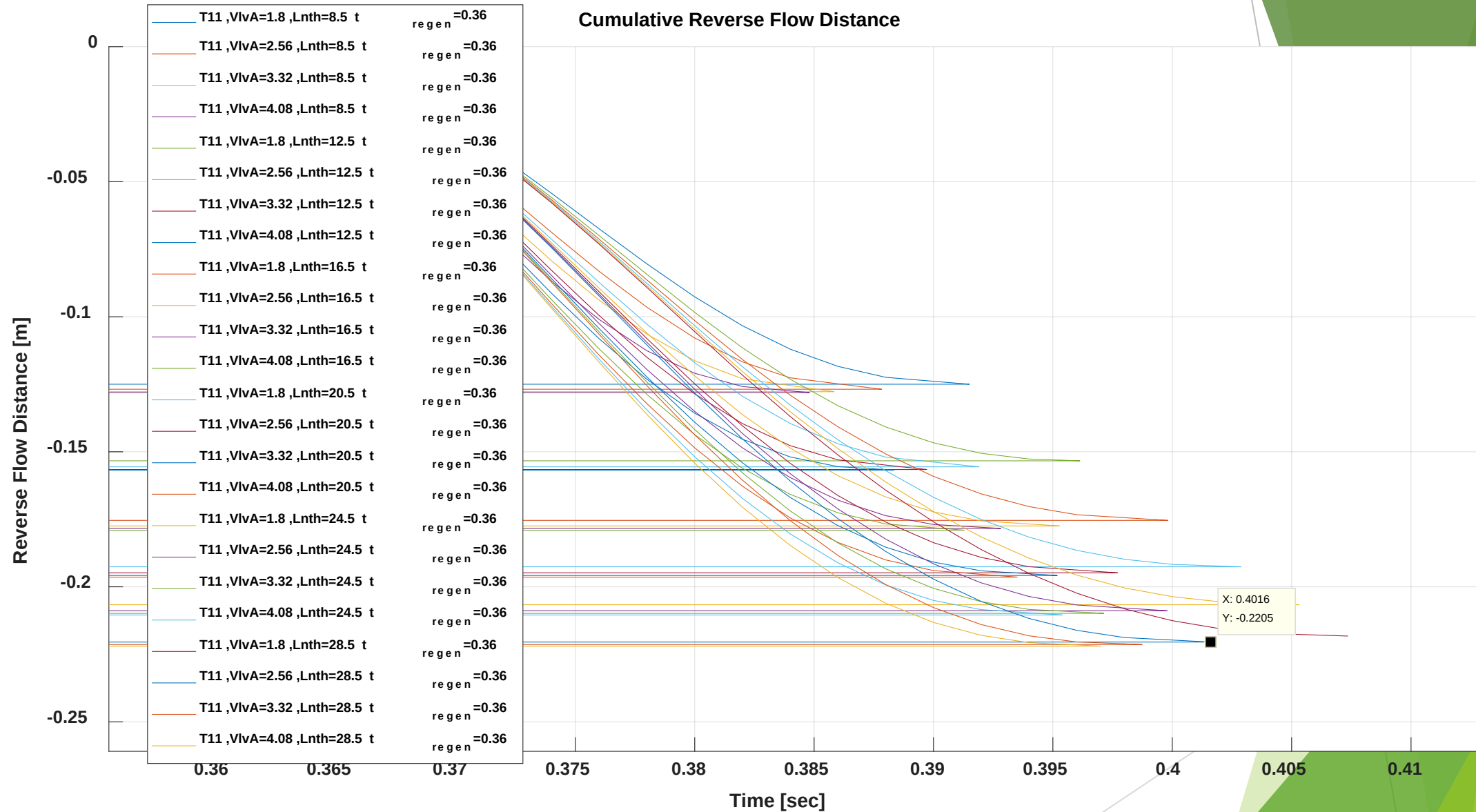


delta Pressure Profile

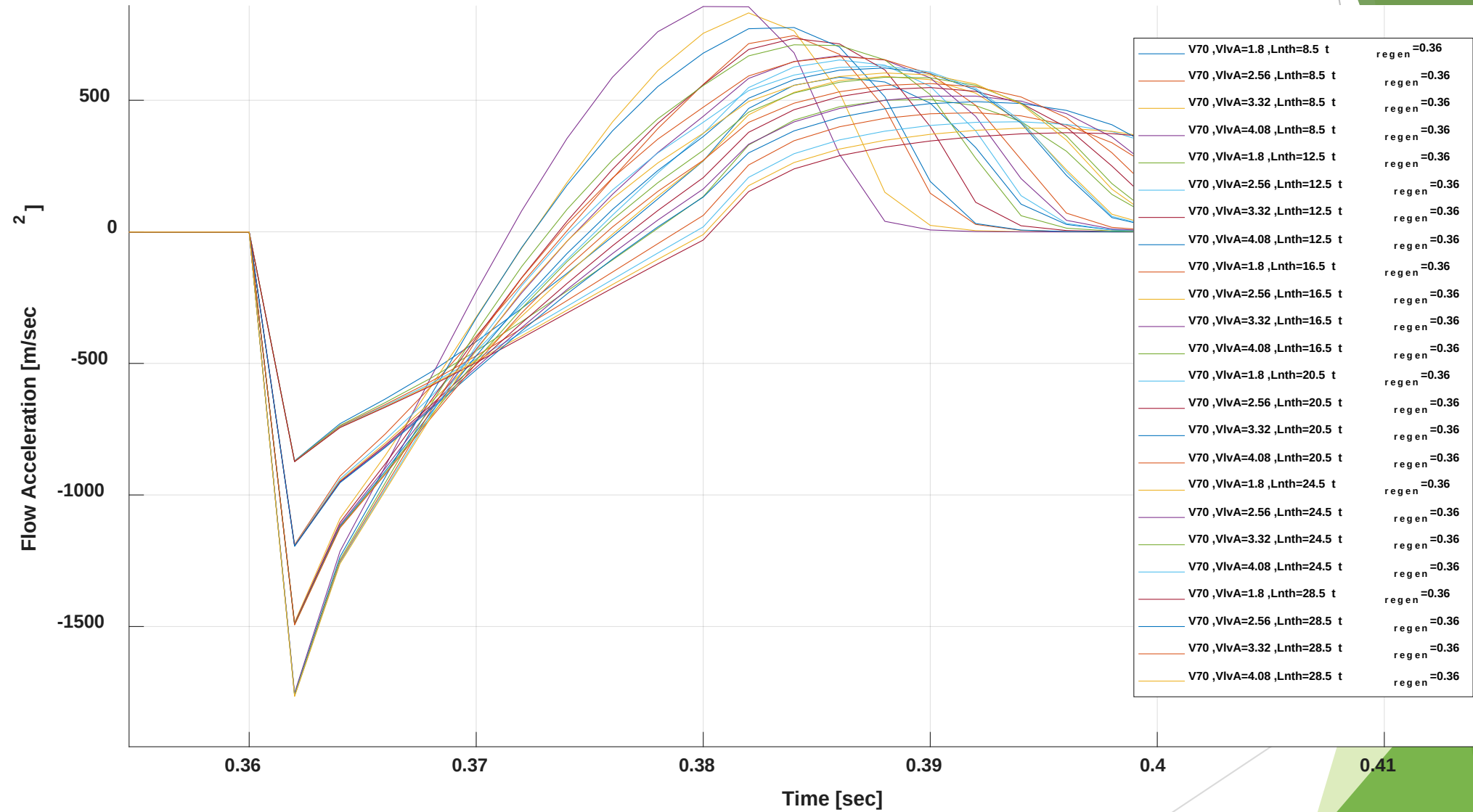


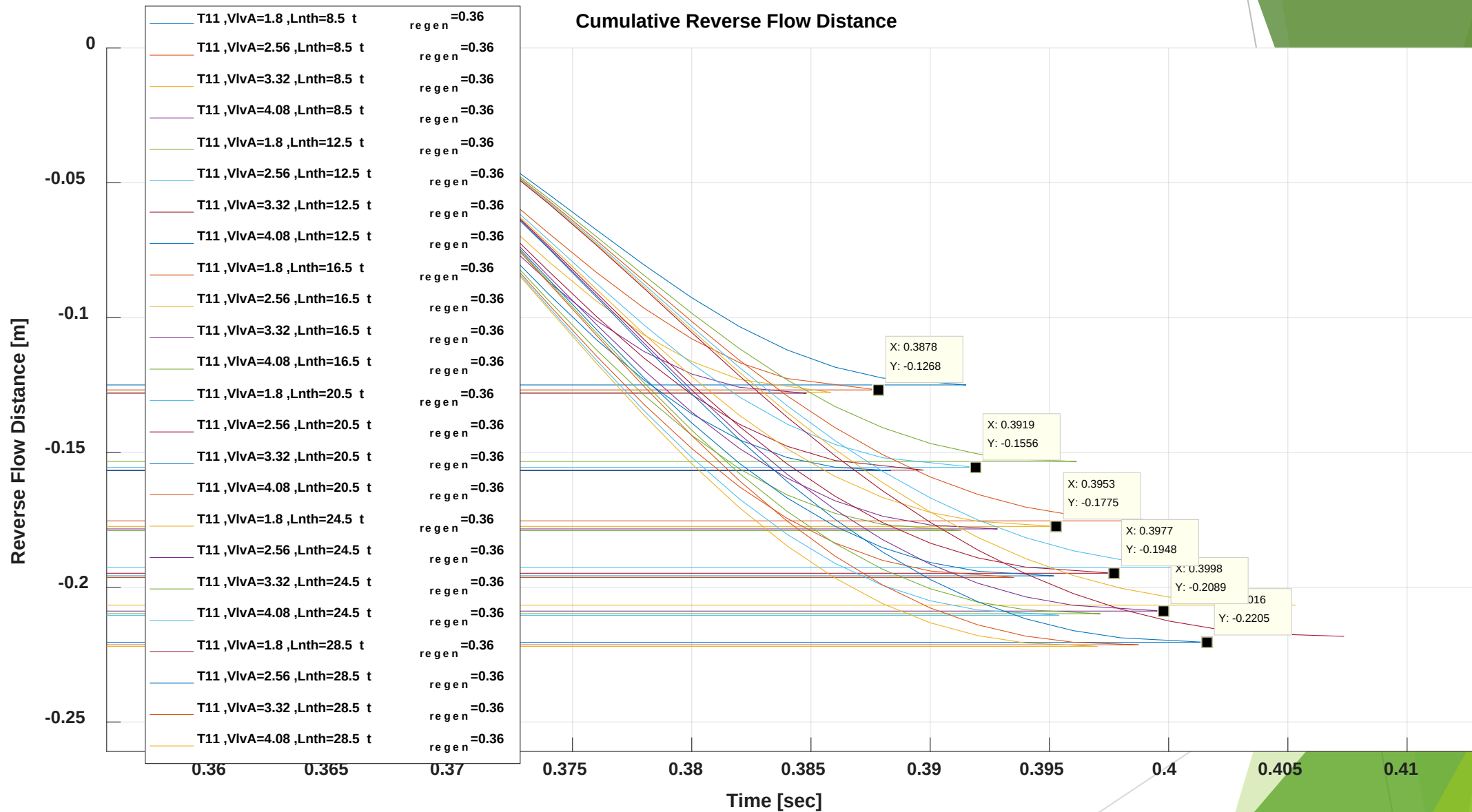
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Z11 ,VlvA=2.56 ,Lnth=8.5 t	reg en =0.36
Z11 ,VlvA=3.32 ,Lnth=8.5 t	reg en =0.36
Z11 ,VlvA=4.08 ,Lnth=8.5 t	reg en =0.36
Z11 ,VlvA=1.8 ,Lnth=12.5 t	reg en =0.36
Z11 ,VlvA=2.56 ,Lnth=12.5 t	reg en =0.36
Z11 ,VlvA=3.32 ,Lnth=12.5 t	reg en =0.36
Z11 ,VlvA=4.08 ,Lnth=12.5 t	reg en =0.36
Z11 ,VlvA=1.8 ,Lnth=16.5 t	reg en =0.36
Z11 ,VlvA=2.56 ,Lnth=16.5 t	reg en =0.36
Z11 ,VlvA=3.32 ,Lnth=16.5 t	reg en =0.36
Z11 ,VlvA=4.08 ,Lnth=16.5 t	reg en =0.36
Z11 ,VlvA=1.8 ,Lnth=20.5 t	reg en =0.36
Z11 ,VlvA=2.56 ,Lnth=20.5 t	reg en =0.36
Z11 ,VlvA=3.32 ,Lnth=20.5 t	reg en =0.36
Z11 ,VlvA=4.08 ,Lnth=20.5 t	reg en =0.36
Z11 ,VlvA=1.8 ,Lnth=24.5 t	reg en =0.36
Z11 ,VlvA=2.56 ,Lnth=24.5 t	reg en =0.36
Z11 ,VlvA=3.32 ,Lnth=24.5 t	reg en =0.36
Z11 ,VlvA=4.08 ,Lnth=24.5 t	reg en =0.36
Z11 ,VlvA=1.8 ,Lnth=28.5 t	reg en =0.36
Z11 ,VlvA=2.56 ,Lnth=28.5 t	reg en =0.36
Z11 ,VlvA=3.32 ,Lnth=28.5 t	reg en =0.36
Z11 ,VlvA=4.08 ,Lnth=28.5 t	reg en =0.36

Cumulative Reverse Flow Distance



Flow Acceleration Profile





```

%Specify the complete path where the executable is located
path_exe = 'C:\Program Files (x86)\GFSSP701\';
%Specify the complete path where the case files are located
%Files required in the directory include:
 %[case]IN.dat,ExhBrake.dat,RegenValve.dat,
 path_case = 'C:\Users\brett\Desktop\1dCode\Jetta1DMod2\';
 name_case = 'Jetta220kPa4inafterDPF'; %name of the case, used to generate
 file names
 Run_steady = 1; %Do you want to perform a steady simulation? 0=No 1=Yesgf
 Run_unsteady = 1; %Do you want to perform an unsteady simulation? 0=No 1=Yes
 % if both steady and unsteady sumlations are performed, the steady
 % siumulation will occur first.

%OUTPUT OPTIONS
SIoutput = 1; %set to 0 for english units on plots, or 1 for SI
PressurePlot = [100,9];% enter the NODE numbers for the pressures you want to
plot
VelPlot = [70];% enter the BRANCH numbers for the velocities you want to plot
MFPlot = [70];% enter the BRANCH numbers for the mass flows you want to plot
dPPlot = [70];% enter the BRANCH numbers for the Pressure Difference you want
to plot
DistRevF = [70];%enter the BRANCH numbers for the reverse flow distance you
want to plot

TargetPressure = 220;%kPa %note: set in kPa for SIoutput=1, in psi for
SIoutput=0
dP = 1.0; %kPa %accuracy desired for target pressure match, set to high
number to skip
tGuessSav = 0.36; %initial guess for regen valve timing
PNode = 3; %please enter the node entry position for the intereated pressure,
i.e. 3 = third entry in pressure output

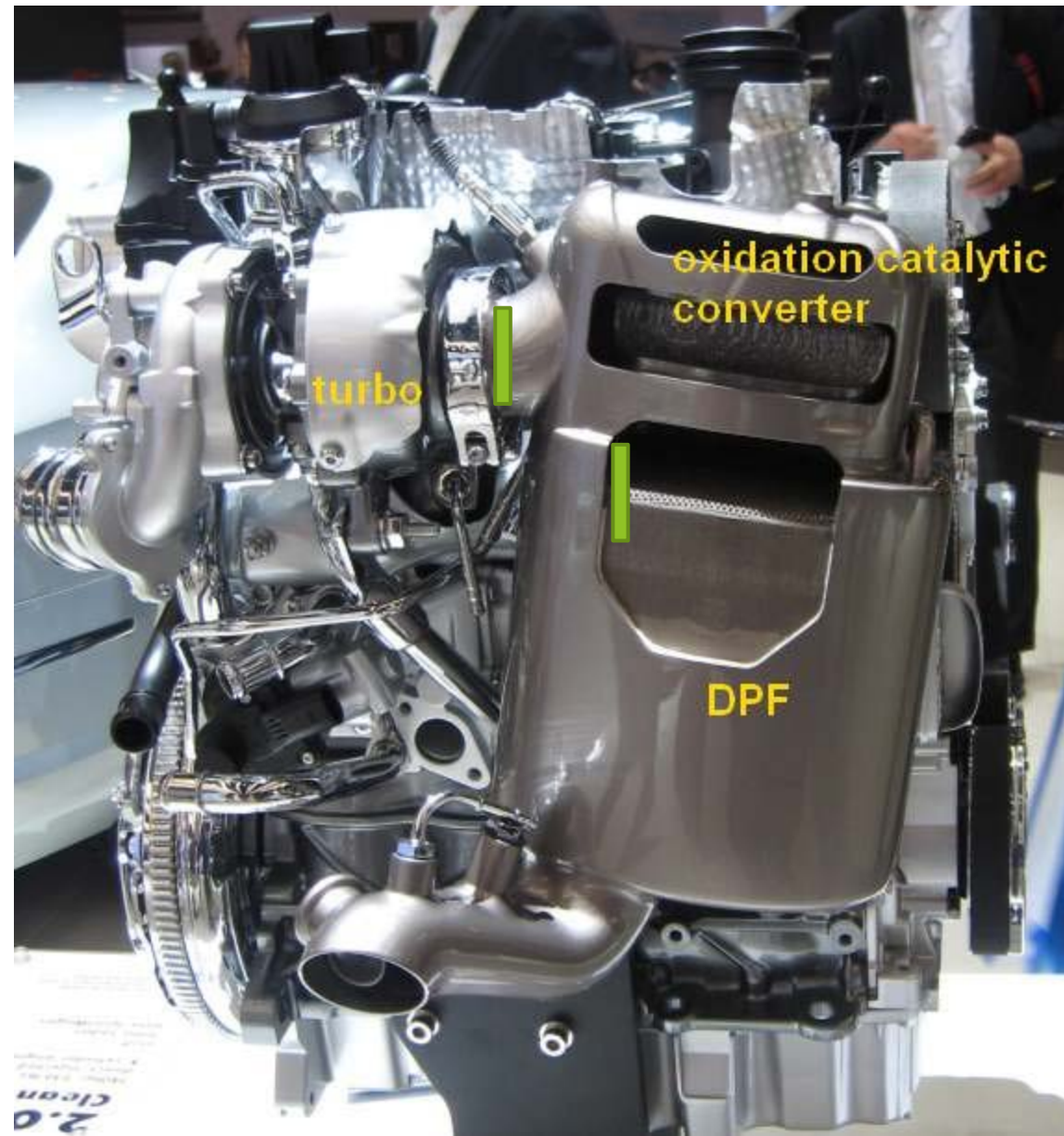
%Note: these inputs are in english units
Variable = 'VlvA'; %specify for output labeling
Range1 = [1.8:0.76:4.08];

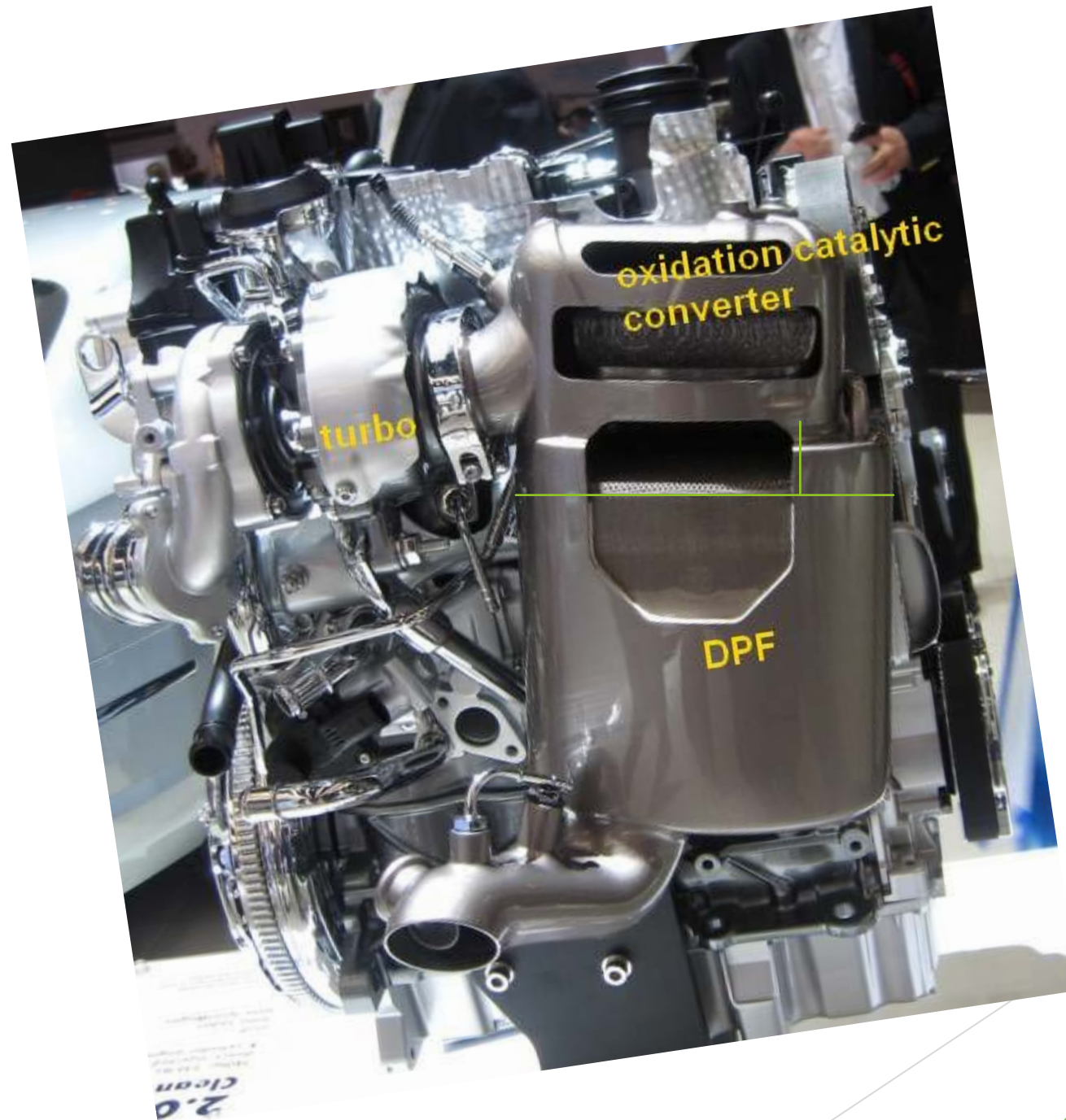
Variable2 = 'Lnth';
Range2 = [8.5:4:28.5]; %currently set to valve area

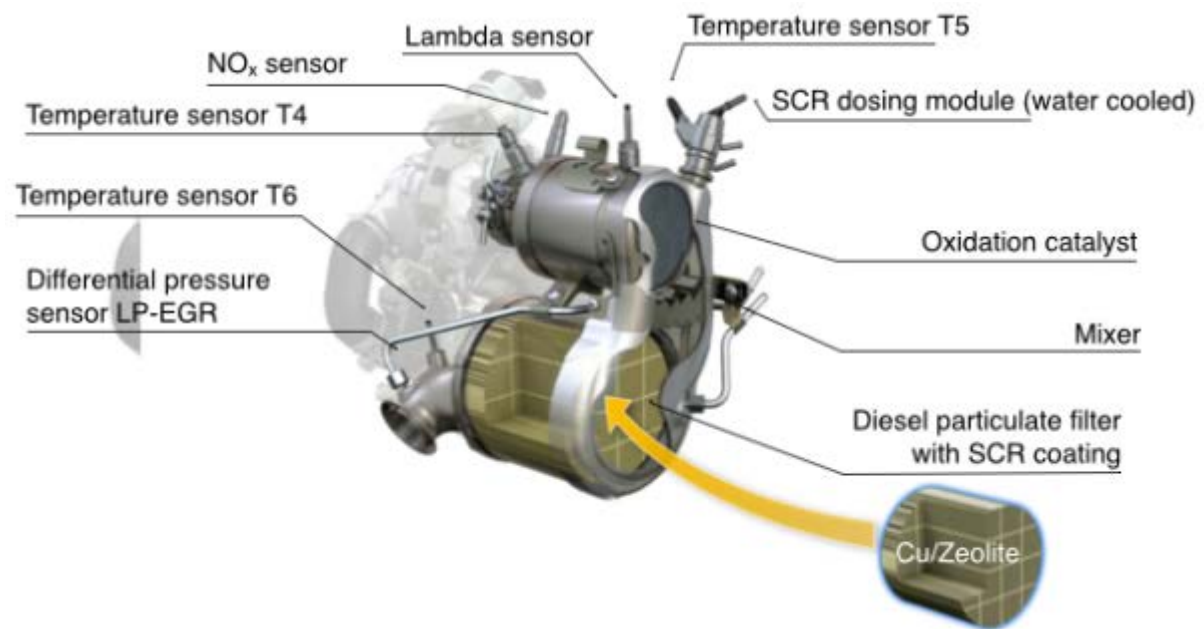
```

[https://www.performanceexhaust.com.au/p/Mitsubishi/
Pajero-NS-NT-NW/3.2TD/MITSUBISHI-PAJERO-NS-NT-NW-
3.2LT-TD-DIESEL-DPF/DPF036](https://www.performanceexhaust.com.au/p/Mitsubishi/Pajero-NS-NT-NW/3.2TD/MITSUBISHI-PAJERO-NS-NT-NW-3.2LT-TD-DIESEL-DPF/DPF036)









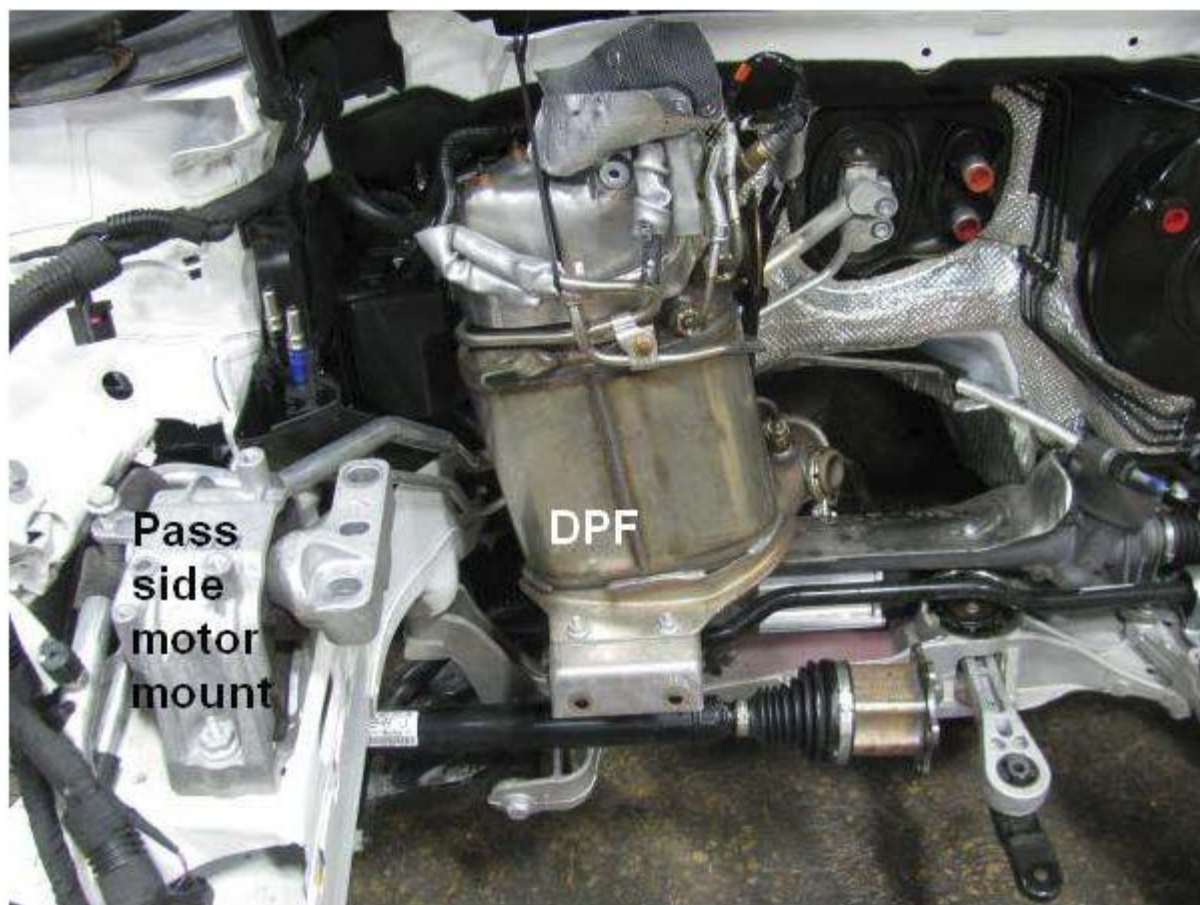


Volkswagen Jetta DPF Outlet

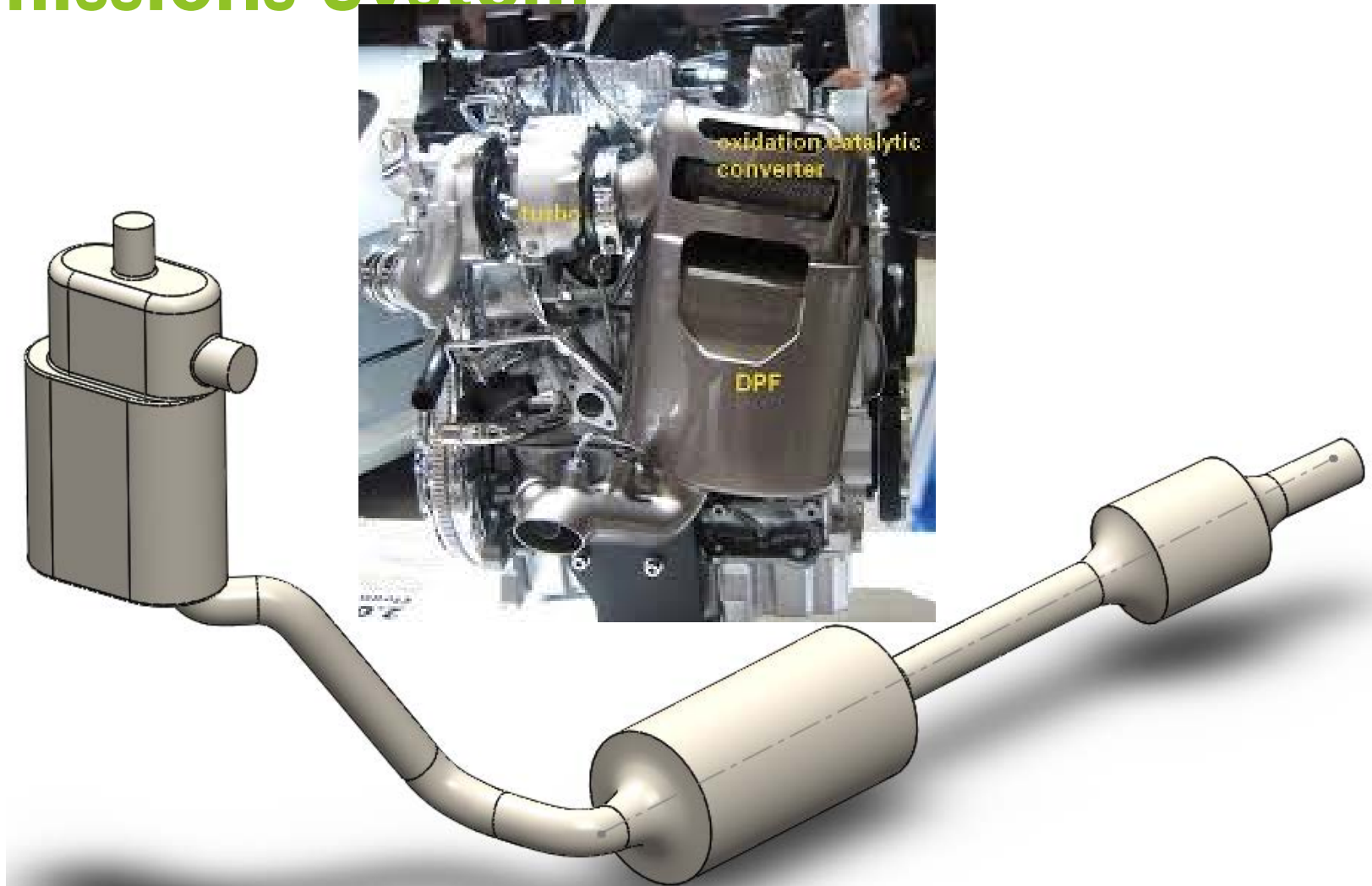






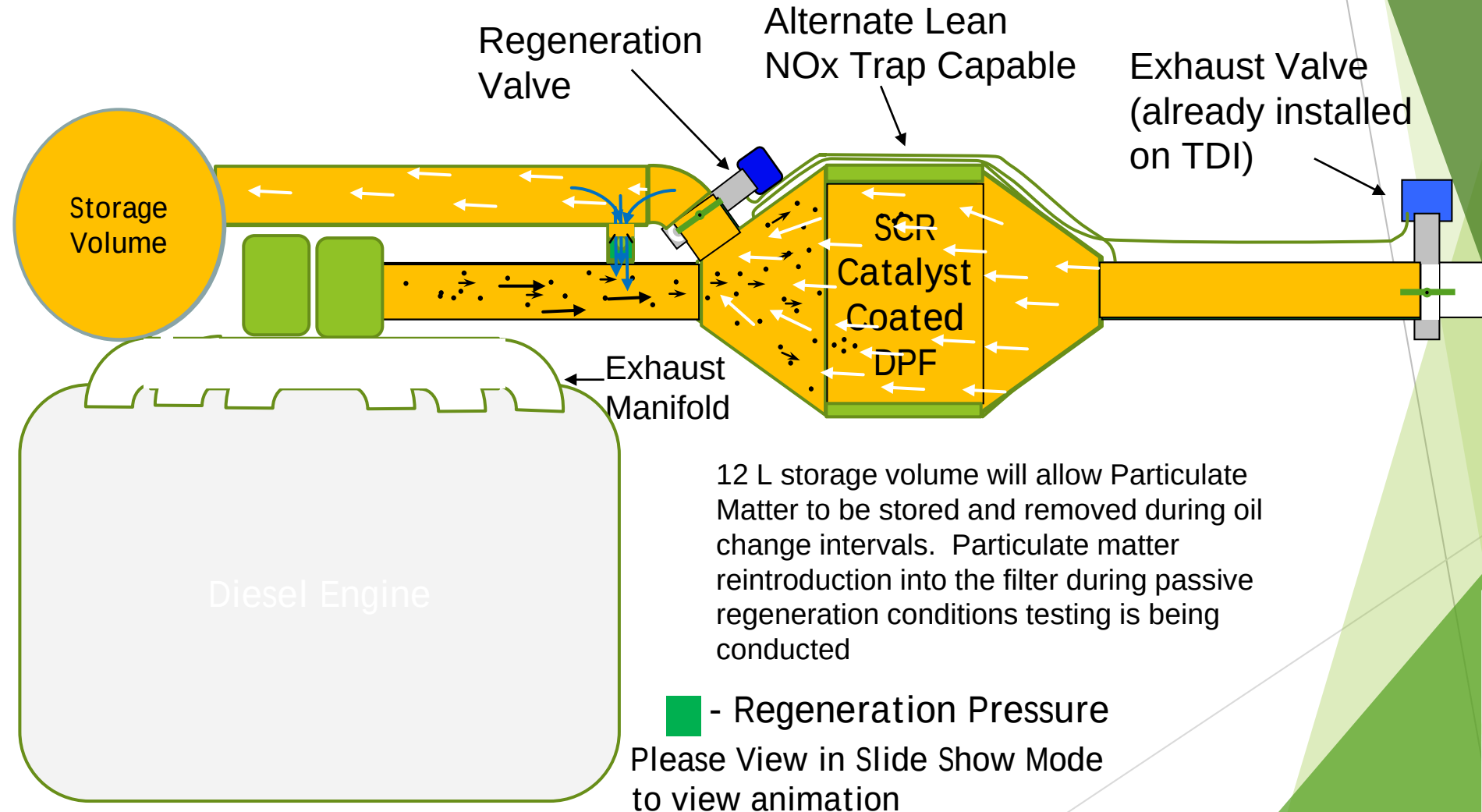


2011 Volkswagen Jetta Sportwagen Emissions System



Cool Particulate Regeneration™, CPR™

How it works



DOC – Diesel Oxidation Catalyst DPF – Diesel Particulate Filter LNT – Lean NOx Trap



Table 4-17 Estimated costs of emission control technologies for European diesel LDV, Vd>2.0 L

REGULATION	EURO 1	EURO 2	EURO 3	EURO 4	EURO 5	EURO 6
INTRODUCTION YEAR	1992	1996	2000	2005	2009	2014
REGULATED POLLUTANTS	(NO _x +HC/PM/CO)	(NO _x +HC/PM/CO)	(NO _x /PM/CO)	(NO _x /PM/CO)	(NO _x /PM/CO)	(NO _x /PM/CO)
EMISSION LEVELS, G/KM	1.13/0.18/3.16	0.7/0.08/1	0.5/0.05/0.64	0.25/0.025/0.5	0.18/0.005/0.5	0.08/.0045/0.5
1. A/F control & engine-out emissions	Assuming a 4-cyl engine, Vd= 2.5 liters					
Fuel system - 50% of cost (a)	-	\$50	\$345	\$380	\$417	\$459
Turbocharger - 50% of cost (b)	-	-	-	\$80	\$80	\$147
Intercooler - 50% of costs (b)	-	-	-	\$34	\$34	\$34
VGT (extra cost) - 50% of costs (b)	-	-	-	-	\$60	\$60
EGR valves (c)	\$30	\$30	\$30	\$38	\$38	\$38
EGR cooling system (c)	-	\$39	\$39	\$47	\$54	\$62
Engine mapping and tuning (d)	-	R&D	R&D	R&D	R&D	R&D
Improvements on combustion chamber & nozzle geometry (e)	-	R&D	R&D	R&D	R&D	R&D
Cost of A/F control & engine-out emissions	\$30	\$119	\$414	\$578	\$684	\$800
2. Aftertreatment systems	Assuming a 4-cyl engine, Vd= 2.5 liters					
Diesel oxidation catalyst (DOC) (f)	-	-	\$99	\$99	\$99	\$99
Diesel particulate filter (DPF) (f)	-	-	-	-	\$402	\$402
Lean NO _x trap (LNT) (f)	-	-	-	-	-	\$509
Selective catalytic reduction (SCR) (f,g)	-	-	-	-	-	-
Cost of aftertreatment systems (h)	\$0	\$0	\$99	\$99	\$501	\$1,011
3. Total cost of hardware [1+2]	\$30	\$119	\$513	\$677	\$1,185	\$1,811
4. Fixed costs (R&D, tooling, certification)	\$26	\$26	\$51	\$51	\$51	\$51
5. Total cost of emissions control tech [3+4]	\$56	\$145	\$564	\$728	\$1,236	\$1,862

(a) Cost of rotary pump, HPFI pump, valves, common rail, and injectors. 50% of cost is charged to non-CO₂ regulated emissions

(b) Single stage turbocharging assumed. 50% of cost is charged to non-CO₂ regulated emissions

(c) Single loop EGR. 50% of cost is charged to non-CO₂ regulated emissions

(d) Maximization of fuel economy and minimization of emissions by fuel injection strategies, air management, turbo, EGR.

(e) Research and development focused on improving combustion (fuel efficiency and emissions) through modeling (CFD) and experiments.

(f) See diesel aftertreatment detail.

(g) SCR cost includes the cost of dosage unit and tank. NH₃ slip catalyst included. NO_x sensor and H₂S catalyst not included in cost.

(h) The cost of aftertreatment systems includes a minimum of devices for control of HC, CO, PM and NO_x. The cost presented may vary depending on specific engine applications.

Table 4-18 Estimated costs of emission control technologies for diesel passenger cars under US Tier 2-Bin 5 emission levels

DIESEL	4-CYL OR Vd<2.5 LITERS	6-CYL OR Vd>2.5 LITERS
REGULATION	TIER 2 BIN 5	TIER 2 BIN 5
INTRODUCTION YEAR	2009	2009
REGULATED POLLUTANTS	(NO _x /PM/CO)	(NO _x /PM/CO)
EMISSION LEVELS, G/KM	0.04/0.006/2.5	0.04/0.006/2.5
1. A/F control & engine-out emission	Assuming 4-cyl, Vd=2.0L	Assuming 6-cyl, Vd=3.0L
Fuel system - 50% of cost (a)	\$420	\$459
Turbocharger - 50% of cost (b)	\$138	\$155
Intercooler - 50% of costs (b)	\$32	\$39
VGT (extra cost) - 50% of costs (b)	\$50	\$60
EGR valves (c)	\$38	\$38
EGR cooling system (c)	\$58	\$66
Engine mapping and tuning (d)	R&D	R&D
Improvements on combustion chamber & nozzle geometry (e)	R&D	R&D
Cost of A/F control & engine-out emission	\$736	\$817

2. Aftertreatment systems	Vd=2.0 L	Vd=3.0 L
Diesel oxidation catalyst (DOC) (f)	\$78	\$116
Diesel particulate filter (DPF) (f)	\$332	\$468
Lean NO _x trap (LNT) (f)	\$413	(\$602)*
Selective catalytic reduction (SCR) (f, g)	-	\$633**
Cost of aftertreatment systems (h)	\$823	\$1,217
3. Total cost of hardware [1+2]	\$1,559	\$2,035
4. Fixed costs (R&D, tooling, certification)	\$51	\$51
5. Total cost of emission control tech. [3+4]	\$1,610	\$2,086

* Note: Cost numbers in parenthesis are informative, not added to final cost summation.

** SCR catalyst cost calculated with SVR corresponding to US requirements (SVR=2.0).

(a) Cost of rotary pump, HPFI pump, valves, common rail, and injectors. 50% of cost is charged to non-CO₂ regulated emissions.

(b) Single stage turbocharging assumed. 50% of cost is charged to non-CO₂ regulated emissions.

(c) Single loop EGR. 50% of cost is charged to non-CO₂ regulated emissions.



Table 4-18 Estimated costs of emission control technologies for diesel passenger cars under US Tier 2-Bin 5 emission levels

DIESEL	4-CYL OR Vd<2.5 LITERS	6-CYL OR Vd>2.5 LITERS
REGULATION	TIER 2 BIN 5	TIER 2 BIN 5
INTRODUCTION YEAR	2009	2009
REGULATED POLLUTANTS	(NO _x /PM/CO)	(NO _x /PM/CO)
EMISSION LEVELS, G/KM	0.04/0.006/2.5	0.04/0.006/2.5
1. A/F control & engine-out emission	Assuming 4-cyl, Vd=2.0L	Assuming 6-cyl, Vd=3.0L
Fuel system - 50% of cost (a)	\$420	\$459
Turbocharger - 50% of cost (b)	\$138	\$155
Intercooler - 50% of costs (b)	\$32	\$39
VGT (extra cost) - 50% of costs (b)	\$50	\$60
EGR valves (c)	\$38	\$38
EGR cooling system (c)	\$58	\$66
Engine mapping and tuning (d)	R&D	R&D
Improvements on combustion chamber & nozzle geometry (e)	R&D	R&D

2. Aftertreatment systems	Vd=2.0 L	Vd=3.0 L
Diesel oxidation catalyst (DOC) (f)	\$78	\$116
Diesel particulate filter (DPF) (f)	\$332	\$468
Lean NO _x trap (LNT) (f)	\$413	(\$602)*
Selective catalytic reduction (SCR) (f, g)	-	\$633**
Cost of aftertreatment systems (h)	\$823	\$1,217
3. Total cost of hardware [1+2]	\$1,559	\$2,035
4. Fixed costs (R&D, tooling, certification)	\$51	\$51
5. Total cost of emission control tech. [3+4]	\$1,610	\$2,086

* Note: Cost numbers in parenthesis are informative, not added to final cost summation.

** SCR catalyst cost calculated with SVR corresponding to US requirements (SVR=2.0).

(a) Cost of rotary pump, HPFI pump, valves, common rail, and injectors. 50% of cost is charged to non-CO₂ regulated emissions.

(b) Single stage turbocharging assumed. 50% of cost is charged to non-CO₂ regulated emissions.

(c) Single loop EGR. 50% of cost is charged to non-CO₂ regulated emissions.



- Higher Backpress
- Vacuum Assist
- Vacuum Assist

